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ULTRA-FAST MISSION ANALYSIS ROUTINE
FOR APOLLO BLOCK II ENVIRONMENTAL
CONTROL SYSTEM RADIATORS

Report No. 350.12

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Submitted by

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TABLE OF CONTENTS

	<u>Page</u>
1.0 SUMMARY	1
2.0 INTRODUCTION	2
3.0 ANALYTICAL METHODS	3
3.1 Heat Transfer Analysis	3
3.2 Pressure Drop Analysis	6
3.3 Bypass Valve Characterization	7
3.4 Proportioning Valve Characterization	7
4.0 RADIATOR CHARACTERIZATION	9
4.1 Thermal Model	9
4.2 Absorbed Heat	9
4.3 Capability	11
5.0 USER'S INSTRUCTIONS	
5.1 Program Description	34
5.2 Data Preparation	34
5.3 Output	38
5.4 Error Diagnostics	39
6.0 LIST OF SYMBOLS	40
References	42
APPENDIX A - Program Listing	43
APPENDIX B - Program Flow Chart	68
APPENDIX C - Dictionary of Fortran Terms	106

LIST OF FIGURES

<u>Figure Number</u>		<u>Page</u>
1	Apollo Block II ECS Radiator Simplified Thermal Model	10
2	Absorbed Heat Data for Lunar Orbit Broadside Orientation	12
3	Absorbed Heat Data for Lunar Orbit Nose Down Orientation	13
4	Absorbed Heat Data for 1.0 RPH Thermal Cycle	14
5	Comparison of Predicted Heat Rejection in Lunar Orbit; Heat Load = 8500 BTU/hr	16
6	Comparison of Predicted Panel Flow Rates and Outlet Temperatures in Lunar Orbit; Heat Load = 8500 BTU/hr	17
7	Comparison of Predicted Pressure Drops in Lunar Orbit; Heat Load = 8500 BTU/hr	18
8	Comparison of Predicted Heat Rejection in Lunar Orbit; Heat Load = 3470 BTU/hr	19
9	Comparison of Predicted Panel Flow Rates and Outlet Temperatures in Lunar Orbit; Heat Load = 3470 BTU/hr	20
10	Comparison of Predicted Pressure Drops in Lunar Orbit; Heat Load = 3470 BTU/hr	21
11	Comparison of Predicted Heat Rejection in Lunar Orbit; Single Panel and Redundant System Operation, Heat Load = 5600 BTU/hr	22
12	Comparison of Predicted Panel Flow Rates and Outlet Temperatures in Lunar Orbit; Single Panel and Redundant System Operation, Heat Load = 5600 BTU/hr	23
13	Comparison of Predicted Pressure Drops in Lunar Orbit; Single Panel and Redundant System Operation, Heat Load = 5600 BTU/hr	24
14	Comparison of Predicted Heat Rejection in Translunar Thermal Cycle; Single Panel and Redundant System Operation	25
15	Comparison of Predicted Outlet Temperatures in Translunar Thermal Cycle; Single Panel and Redundant System Operation	26

LIST OF FIGURES (Cont'd)

<u>Figure Number</u>		<u>Page</u>
16	Comparison of Predicted Pressure Drops in Translunar Thermal Cycle; Single Panel and Redundant System Operation	27
17	Comparison of Predicted Heat Rejection in Earth Orbit . .	28
18	Comparison of Predicted Panel Flow Rates and Outlet Temperature in Earth Orbit	29
19	Comparison of Predicted Pressure Drops in Earth Orbit . .	30
20	Comparison of Predicted Heat Rejection and Outlet Temperatures for a Deep Space Transient	31
21	Run Submission Card Deck Configuration	36

1.0 SUMMARY

This report presents analytical methods, program description, sample results and user's instructions for a digital computer routine for transient space radiator performance predictions. The routine was written specifically for analysis of the Apollo Block II Environmental Control System (ECS) radiators.

Specific equations for a simplified thermal model of the radiators are written directly in the program. All equations necessary to obtain the radiator outlet temperature, heat rejection and pressure drop are contained in the routine. The temperature equations for each node in the thermal model are solved by an implicit finite difference method. All thermal properties are considered to be constant with the exception of the radiator coolant viscosity. The routine includes a characterization of the flow proportioning and bypass valves which are contained in the radiator systems. Provisions are also included for single panel and redundant loop operation.

Radiator absorbed heat data for four vehicle environments (two lunar orbit orientations, a translunar thermal cycle and zero absorbed heat) are contained in the routine. A mission code determines which set of absorbed heat flux data is used. Provisions are also included for inputting absorbed heat data as a function of time. A cyclic repeat of the heat flux data is utilized allowing multiple periods to be analyzed with heat flux data supplied for only one period.

Inputs for the first four missions consist of mission code, print interval, inlet temperature and total flow rate. The input mission requires mission code, print interval, total mission time and time dependent tables of incident heats, inlet temperature and flow rate. Output for all the missions includes heat rejection, pressure drop, low load heater on/off operation, flow rate and outlet fluid temperature printed at the times defined by the print interval and (following completion of the mission) maximum, minimum and average values for heat rejection, pressure drop and fluid outlet temperature and total heat dissipated by the low load heater.

Computer time required to analyze a 4.08 hour lunar orbit mission (two orbits) with a calculation and print interval of .02 hours is 25 seconds on the Univac 1108 computer. This represents a routine run speed of better than 500 times real time.

The Apollo Block II ECS radiator performance predictions obtained by the computer program described herein have been compared to detailed thermal model predictions which have been verified by predicting test results. All of the expected operating modes and environmental conditions of the Block II ECS radiator were considered. All active controls (bypass valve, proportioning valve, isolation valve and low load heater) were exercised. Both the primary and redundant system performance predictions were compared. It has been determined that the computer routine provides adequate Block II ECS radiator performance predictions for a wide variety of conditions with a minimum of computer time required.

2.0

INTRODUCTION

In the design of thermal control flow systems, whether it be that of individual components or overall systems, the ability to determine a design's adequacy is enhanced by the capability to simulate its performance while it is subjected to a variety of mission parameters. The space radiators used in the Apollo Block II Environmental Control System (ECS) are components critical to the operation of the ECS and as such, their response to various combinations of environmental parameters has been established. Computer analysis of these space radiators has required large quantities of computer time for the many missions required for a comprehensive study.

This report summarizes the creation of a program which computes approximate radiator performance rapidly enough to permit a large number of parametric mission analyses with minimum computer time. The work was done under Contract NAS9-6807, which requires modification of the Block II component subroutine described in Reference 1. The thermal model of Reference 1 has been improved and additional analyses have been made to verify the performance predictions of the program. The revised routine also contains the redundant loop, the low load heater and provisions for single panel operation. In addition to the four missions built into the parent subroutine, a fifth mission has been created to permit the card input of time-dependent tables of incident heats, system fluid inlet temperature, and total flow rate. Provision is also made for dynamic printout of heat rejection, pressure drop, flow rate, and outlet fluid temperature at selected intervals for all five missions. A User's Manual for the program thus formed is included.

The stated objective of the computer routine requires the prediction of suitable radiator performance in any environment and with any heat load. In order to evaluate the adequacy of the computer routine predictions it is necessary to establish a baseline to which the predictions can be compared. No flight data are available and test data are available for only a limited number of conditions. The approach taken herein is to use the results of a detailed computer routine and thermal model as a baseline for evaluating the results of the simplified computer routine. The detailed baseline model has been verified by predicting the results of the Qualification Test of the Apollo Block II ECS radiator. Baselines have been established for a wide variety of conditions using the detailed model.

3.0 ANALYTICAL METHODS

This section presents the methods used in the Apollo Block II ECS radiator computer routine (AB2RAD) for heat transfer and fluid flow analysis. The equations used in the temperature and pressure drop calculations are given along with the flow control valve characterizations.

3.1 Heat Transfer Analysis

The temperature of each fluid, tube and structure node in the thermal model is determined from the general temperature equation below:

$$T_j'' = \frac{T_j^{(\tau)} + \sum_k D_{jk} T_k'' + F_j - E_j T_j''^4}{1 + \sum_k D_{jk}}$$

The above equation is derived in Reference 2 and the symbols are defined in the List of Symbols (page 40) and Table 1.

An implicit backward-difference method is used to determine the temperature of each node. First the coefficients in the above equation are determined for each node. These coefficients are detailed in Table 1. All coefficients are a function of the computing time increment. The fluid upstream temperature coefficient is a function of the flow rate and the tube and structure absorbed heat coefficients are a function of the incident heat. All other terms in the coefficients given in Table 1 are determined from the radiator physical characteristics and input as constants in the routine.

The resulting set of non-linear temperature equations are solved simultaneously by a modified point-iterative method known as "successive overrelaxation (SOR)" to yield the temperature of each lump at the end of the computing time increment. The method is as follows:

1. Assume an initial temperature matrix called T.
2. Set matrices T₁ and T₂ to T.
3. Using the values of temperature in T₂, calculate values of temperature from the general temperature equation above, one lump at a time. Call this iterate T' for the particular lump. The T₂ value is determined by the equation $T_2 = T_1 + \phi (T' - T_1)$, ϕ being the overrelaxation parameter ($\phi = 1.3$). This procedure is continued until each lump's equation has been iterated.

TABLE 1 - TEMPERATURE EQUATION COEFFICIENTS

Type Lump j	D_{jk}	T_k	E_j	F_j
Fluid	$\frac{\dot{w}_j \Delta T}{w_f}$	T_{fu}	0	0
	$\frac{h_f A_f \Delta T}{w_f c_f}$	T_t		
Tube	$\frac{h_f A_f \Delta T}{w_t c_t}$	T_f	$\frac{\epsilon_j \sigma A_{ej} \Delta T}{w_t c_t}$	$\frac{\alpha_j Q_{tj} A_{ej} \Delta T}{w_t c_t}$
	$\frac{U_{jk} \Delta T}{w_t c_t}$	T_b		
Structure	$\frac{U_{jk} \Delta T}{w_s c_s}$	T_s	$\frac{\epsilon_j \sigma A_{ej} \Delta T}{w_s c_s}$	$\frac{\alpha_j Q_{sj} A_{ej} \Delta T}{w_s c_s}$

The quantities used in Table 1 are defined as the following:

\dot{w}_j - Flow rate in the tube containing fluid lump j. This is a variable quantity calculated at the beginning of each calculation time increment.

ΔT - calculation time increment. This value is set internally in the routine as 0.02 hour.

w - weight of lump j.

T_{fu} - temperature of fluid lump upstream of lump j.

h_f - heat transfer coefficient.

A_f - internal area for heat transfer.

c - specific heat

T_k - temperature of neighboring lump in communication with lump j.

ϵ_j - emissivity of lump j.

σ - Stefan-Boltzmann constant.

$\alpha_j Q_j$ - incident heat absorbed by lump j.

TABLE 1 - TEMPERATURE EQUATION COEFFICIENTS (Cont'd)

A_{ej} - external area for radiation of lump j.

$U_{jk} = A_c \left[\frac{k}{Y_j + Y_k} \right]$ - conductance between lump j and neighboring lump k.

k - thermal conductivity.

A_c - conduction area between lumps j and k.

Y_j, Y_k - conduction path lengths.

Subscripts

f - fluid

t - tube

s - structure (fin)

All temperature calculations in the subroutines are carried out in degrees Rankine.

4. The new matrix of T_2 thus determined is compared to the T_1 matrix term by term. If $|T_2 - T_1|$ is less than the iteration limit ($\Theta = 0.10$) the iteration of this particular equation is temporarily suspended. If $|T_2 - T_1|$ is less than Θ for each lump, the solution is achieved.
5. The T_1 matrix is set to the T_2 matrix and the process is repeated from step 3. As soon as the last lump satisfies the $|T_2 - T_1| < \Theta$, if all equations were not iterated, the process is again begun for each lump from step 3.

The standard GOR procedure is modified in that those equations which satisfy the $|T_2 - T_1| < \Theta$ are not iterated until all equations have satisfied the relation.

3.2 Pressure Drop

The pressure drop in the radiators and connecting tubes is calculated by the following equation and is restricted to Reynolds numbers less than 2000.

$$\Delta P_j = \sum_i B_i \mu_i \dot{w}_j$$

The constants, B_i , are obtained from the density and geometry of the fluid lump. The dynamic viscosity, μ , of each fluid lump is determined from the fluid lump temperature by table look-up.

The flow rates in the five parallel tubes of each radiator panel are determined by requiring the pressure drop for each of the parallel flow paths to be equal. Therefore defining:

$$AK_j = \Delta P_j / \dot{w}_j$$

then the flow rate in tube j is

$$\dot{w}_j = (AK_1)(\dot{w}_1)/AK_j$$

$$\dot{w}_{total} = \sum_j \dot{w}_j$$

The flow rate in tube 1 is

$$\dot{w}_1 = \frac{\dot{w}_{total}}{1 + \sum_j \frac{AK_1}{AK_j}} \quad j \neq 1$$

3.3 Bypass Valve Characterization

The bypass valve in the Apollo Block II ECS radiator system controls the heat rejection by regulating the flow through the radiator. The bypass line and valve pressure losses are not calculated since they are in parallel with the radiator, and their pressure loss will equal the radiator pressure loss. The valve is programmed to route flow through the radiator such that the mixed temperature is controlled to a nominal 45°F if obtaining this temperature at the end of each computing interval is within the valve response capability. The valve's position (fraction bypass) is determined according to the difference between the mixed outlet temperature after each iteration and the desired temperature (45°F). A deadband is included in the valve logic such that the valve is not activated until the above temperature difference exceeds 0.75°F. The fraction bypass is characterized in AB2RAD by the equation:

$$Z = Z_{\text{previous}} + \Delta Z$$

where Z is the fraction bypassed. The term, ΔZ , is calculated from the valve response characteristics and computing time increment. The usual valve response characteristics were altered to desensitize the valve in order to eliminate problems with the larger time increments used in the routine. The valve rate factor used in AB2RAD is 0.0003 (fraction of full travel per °F per second) and the rate limit is 0.0033 (fraction of full travel per second). The actual valve constants are 0.00258 for the rate factor and 0.0465 for the rate limit.

3.4 Proportioning Valve

The proportioning of flow through the two parallel radiators in the Apollo Block II ECS radiator subsystem is controlled by a proportioning valve located at the junction of the inlets to the radiators. Temperature sensors located near the outlets of the radiators provide signals for valve control. The valve is designed to respond by increasing flow to the radiator with the lower outlet temperature. This arrangement is utilized to provide maximum heat rejection when the two radiator panels operate in significantly different incident heat environments.

The proportioning valve simulation in the computer routine is designed to reproduce the characteristics of the actual valve that will be used in the Apollo Block II subsystem. The basic equation describing the operation of the valve is:

$$X = X_{\text{previous}} + \frac{\Delta T}{t_c} \left[(X_i - X_{\text{previous}}) + G (T_{RT} - T_{LT}) \right]$$

where:

- G = valve gain
- t_c = valve time constant
- ΔT = computing time increment
- X_i = initial valve position (from left)
- X_{previous} = valve position at previous iteration
- X = new valve position
- T_{RT}, T_{LT} = temperature of sensors in right and left tubes

This equation is valid only when the computing time increment is small compared with the valve time constant. The computing time increment used in AB2RAD (0.02 hours) is much greater than the valve time constant (0.000833 hours) and the valve position is obtained by the equation:

$$X = X_1 + G (T_{RT} - T_{LT})$$

After the position, X , is determined, it is used to define the pressure drops in each side of the valve by the relations:

$$\Delta P_{RT} = H \left[\frac{\dot{w}_{RT}}{X_2} \right]^2$$

$$\Delta P_{LT} = H \left[\frac{\dot{w}_{LT}}{X_1} \right]^2$$

where:

H = proportionality factor for valve pressure drop
 $\dot{w}_{RT}, \dot{w}_{LT}$ = right and left side flow rates
 X_1 = valve position from left
 X_2 = valve position from right

The valve pressure drops are considered together with the pressure drops in the remainder of the right and left hand flow paths to determine flow rates that give a pressure balance for both sides. Considering the pressure drop of the radiator to be a linear function of flow rate, such that $\Delta P = K \dot{w}$, the pressure balance in the radiator and valve can be written as:

$$K_{RT} \dot{w}_{RT} + H \left[\frac{\dot{w}_{RT}}{X_2} \right]^2 = K_{LT} \dot{w}_{LT} + H \left[\frac{\dot{w}_{RT}}{X_1} \right]^2$$

where:

K_{RT} = ΔP of radiator right branch/right side flow rate
 K_{LT} = ΔP of radiator left branch/left side flow rate
 $\dot{w}_{RT}, \dot{w}_{LT}$ = right and left flow rates

Substituting $\dot{w}_{RT} = \dot{w}_{TOT} - \dot{w}_{LT}$ and rearranging yields a solution for \dot{w}_{LT} in the standard quadratic form:

$$H \left[\frac{1}{X_1^2} - \frac{1}{X_2^2} \right] \dot{w}_{LT}^2 + \left[K_{LT} + K_{RT} + \frac{2 H \dot{w}_{TOT}}{X_2^2} \right] \dot{w}_{LT} - \left[K_{RT} \dot{w}_{TOT} + \frac{H (\dot{w}_{TOT})^2}{X_2^2} \right] = 0$$

The standard quadratic equation solution is used in the AB2RAD to solve the above equation for the flow rate in the left branch of the radiator system as determined by the proportioning valve position.

4.0 RADIATOR CHARACTERIZATION

The AB2RAD calculates heat rejection and pressure drop as a function of time. This section describes the thermal model used, the orientations available and a comparison of predicted results to a detailed analysis of the Block II ECS radiator.

4.1 Thermal Model

The thermal model used in AB2RAD consists of two radiator panels connected in parallel. Each panel is composed of a five flow tube stagnation panel plus a single flow tube in series downstream of the selective stagnation panel. Figure 1 shows the thermal model for the complete radiator system including valves as analyzed by AB2RAD.

The thermal model is a coarse breakdown of the actual Apollo Block II ECS radiator panels. The primary and redundant flow tubes in the stagnation panel were divided into two equal length isothermal nodes. This coarse division resulted in twelve tube and fluid nodes and eight fin nodes per stagnation panel. Each series panel has four fluid and tube nodes (two each for the primary and redundant loop) and two fin nodes. The inlet and outlet manifold tubes and other connecting tubes shown in Figure 1 were considered adiabatic, and therefore did not affect the thermal balance.

4.2 Absorbed Heat

Heat flux data for two 80 nautical mile ecliptic plane lunar orbits, a 1.0 RPH translunar thermal cycle and a deep space environment are contained in AB2RAD. A description of these thermal environments follows:

1. Lunar Orbit Broadside Orientation - The longitudinal axis of the vehicle is always parallel to the lunar surface and passes through the subsolar point. The vehicle orbits with one radiator panel always facing the Moon while the diametrically opposed panel alternately sees deep space and the solar flux.
2. Lunar Orbit Nose Down Orientation - The longitudinal axis of the vehicle is always perpendicular to the lunar surface and the vehicle passes through the subsolar point. The mid-point of both the leading and lagging radiator panels are in the plane of the orbit.
3. Translunar Thermal Cycle - The longitudinal axis of the vehicle is perpendicular to the sun's rays. The vehicle rolls at a rate of one revolution per hour.
4. Deep Space - The radiator panels are positioned so as to receive zero incident heat.

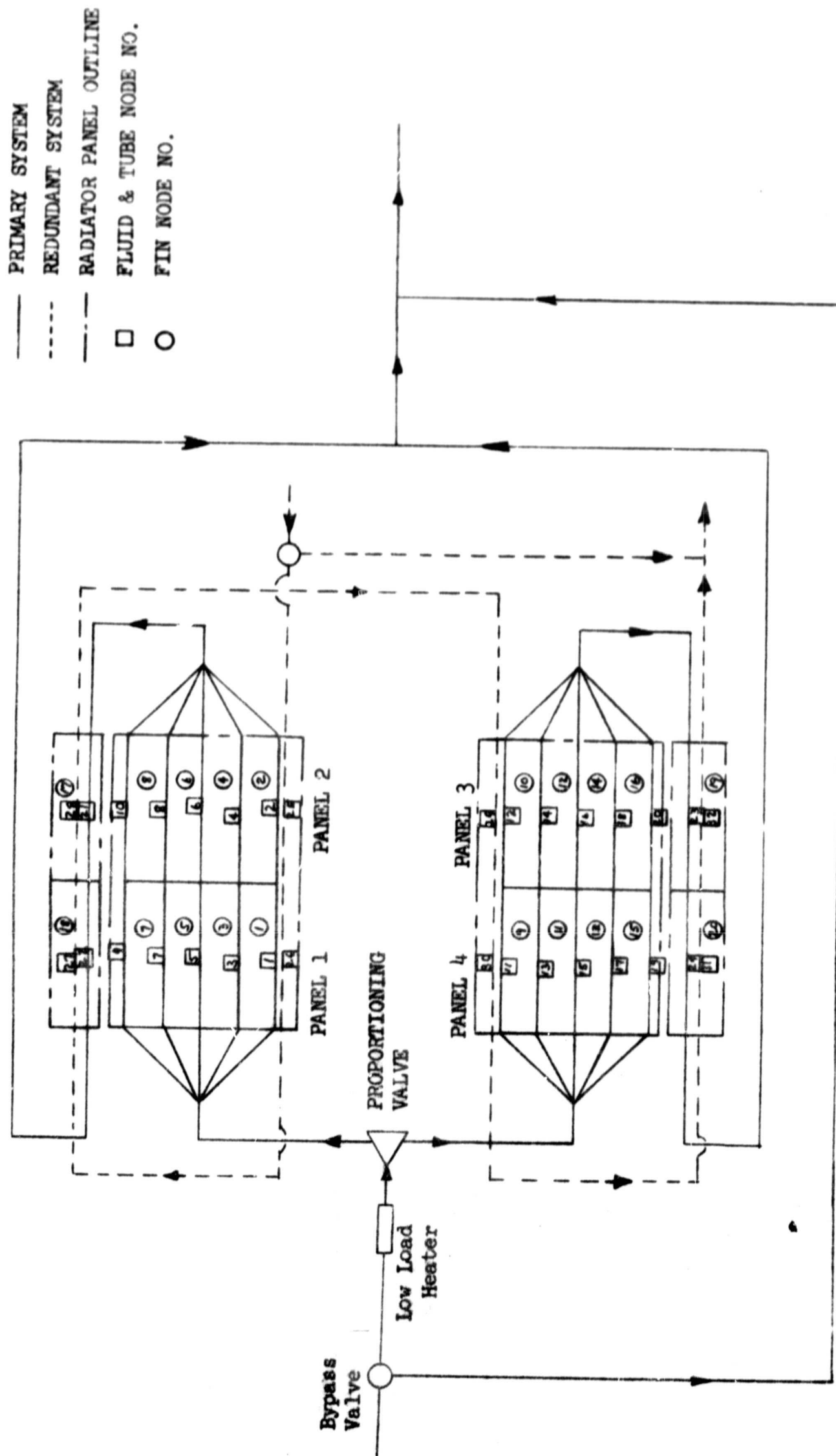


FIGURE 1 APOLLO BLOCK II ECS RADIATOR SIMPLIFIED THERMAL MODEL

Absorbed heat data for the two lunar orbits above were obtained by use of the digital computer routine of Reference 3. The solar absorptivity and emissivity used in this analysis were 0.20 and 0.92 respectively. Figures 2 through 4 show the absorbed heat for the lunar broadside orientation, lunar nose down orientation and a 1.0 RPH thermal cycle respectively. The data shown are for a flat plate inclined at an angle of 30° from the radiator mid point. It was determined that these flat plate values of absorbed heat represented an average value for the curved radiator surface.

The absorbed heat data for each of the above conditions are written into the routine for one orbit for the lunar orbits and one vehicle revolution for the thermal cycle. A cyclic repeat of the curves is provided to obtain data for four orbits (8.1676 hours of mission time) and four vehicle revolutions (4.0 hours of mission time). In addition to the absorbed heat data contained in the routine, AB2RAD provides for the tabular input of any heat flux data desired. Input data requirements and instructions are given in Section 5.0.

4.3 CAPABILITY

The AB2RAD is intended to be suitable for predicting radiator performance in any environment and with any heat load. In order to evaluate the adequacy of the AB2RAD predictions, comparisons have been made to the detailed analysis results presented in References 4 and 5 for the Block II ECS radiator. The baseline model used in Reference 4 has been verified by predicting the results of the Qualification Test of the Block II ECS radiators (Reference 5). It should be noted that the baseline thermal model and Qualification Test hardware assumed adiabatic conditions between the radiator supply and return lines and the Service Module structure. Recent tests of the complete Block II ECS (2TV-1) and preliminary analyses have indicated that a significant amount of heat can be added to the radiator outlet line from the Service Module structure. Since the AB2RAD is based on the thermal model of Reference 4, the effect of the Service Module structure on the supply and return lines is not included in the results. The baseline model has a total of 1141 nodes and yields 2.5 hours of mission time for each hour of computer time as compared to the AB2RAD total of 84 nodes and run speed of better than 500 hours of mission time for each hour of computer time. Both models were run on the Univac 1108 computer. Performance comparisons were made between the baseline analyses and the AB2RAD results for the following conditions:

- (1) Lunar orbit, broadside orientation, 8500 BTU/hr heat load, primary system operating.
- (2) Lunar orbit, broadside orientation, 3470 BTU/hr heat load, primary system operating.
- (3) Lunar orbit, broadside orientation, 5600 BTU/hr heat load, redundant system and one primary system panel operating.
- (4) Translunar thermal cycle, 2.5 RPH, 6880 BTU/hr heat load, redundant system and one primary system panel operating.
- (5) Earth orbit, broadside orientation, 8800 BTU/hr heat load, primary system operating.
- (6) Deep space transient from a heat load of 4415 BTU/hr to 9275 BTU/hr.

A total flow rate of 200 lb/hr was used for both the primary and redundant systems with the inlet temperatures computed from the heat load for each case.

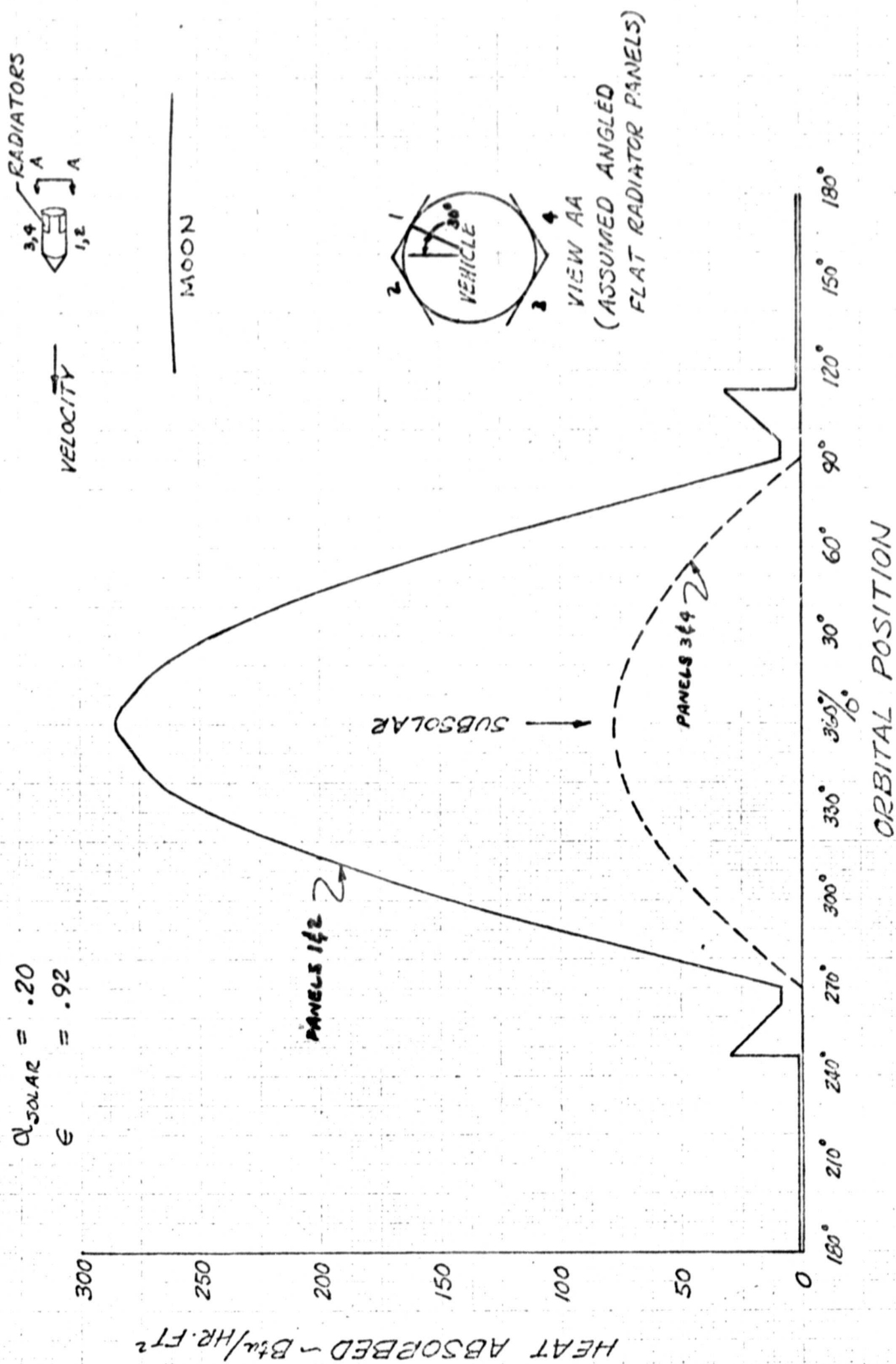


FIGURE 2 ABSORBED HEAT DATA FOR LUNAR ORBIT BROADSIDE ORIENTATION

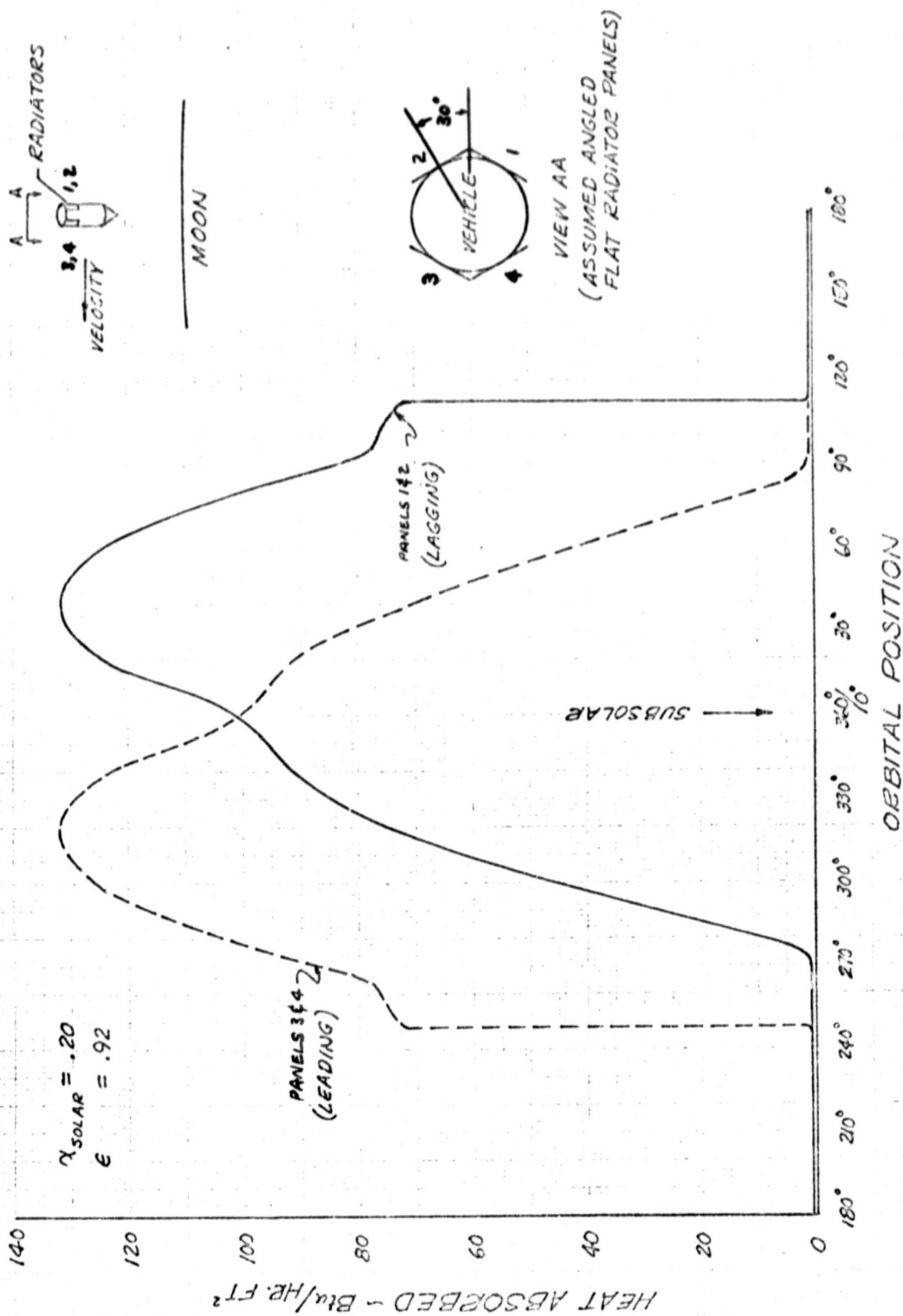


FIGURE 3 ABSORBED HEAT DATA FOR LUNAR ORBIT NOSE DOWN ORIENTATION

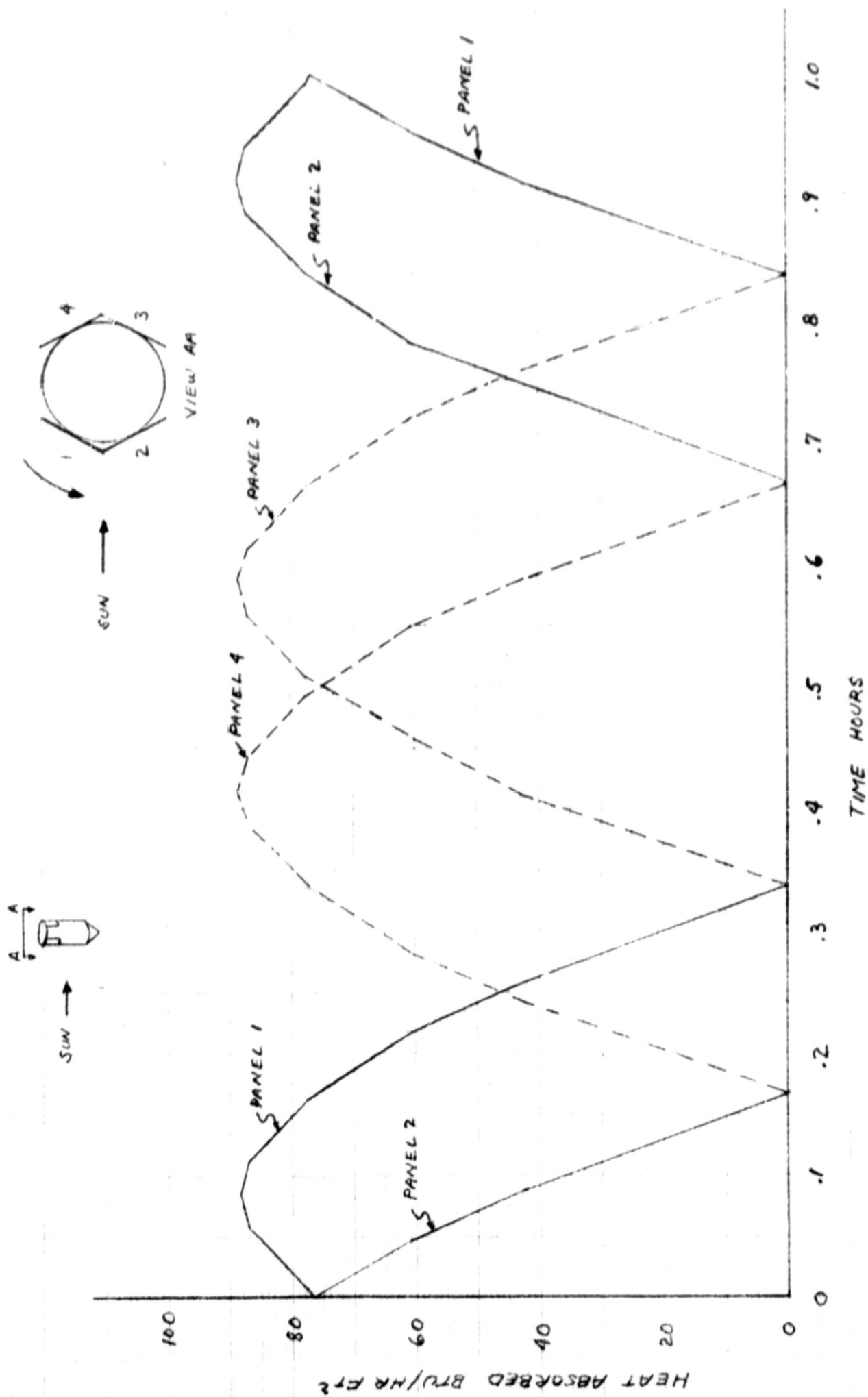


FIGURE 4 ABSORBED HEAT DATA FOR 1.0 RPH THERMAL CYCLE

Figures 5 through 20 present the results of the analysis for each of the above conditions. Comparisons of heat rejected, panel flowrate, radiator outlet temperature, and radiator pressure drop are presented for the first 5 conditions. Only the radiator outlet temperature and heat rejection are compared for the deep space transient since the baseline for this condition was taken directly from reference 5 and the flow rate and pressure drop results were not available. A summary of the results for the six conditions compared is given in Table 2. The average heat rejection errors (Table 2) are based on the integrated heat rejection rates over the period of an orbit, or a single revolution in the case of translunar thermal cycle. Water boiling rates are based on the assumption that the water boiler is "on" when the coolant temperature is above 48°F, and that enough water is boiled to reduce the coolant temperature to 41.5°F. These assumptions yield the maximum amount of water which can be boiled based on hardware specification limits.

The results indicate that the AB2RAD provides adequate performance predictions for a variety of environment and heat load conditions. All predicted performance parameters show a reasonable agreement with the baseline analysis except for the panel outlet temperatures on Figures 6 and 9. The AB2RAD predicts that the outlet temperature of the panel facing the moon, during the sunlight portion of the orbit, gets much hotter than predicted by the baseline analysis. Since the flow rate to the "hot" panel is less than 10 lb/hr (due to the proportioning valve action) the hotter panel outlet has very little effect on overall radiator performance. It should be noted that during the Block II ECS radiator Qualification Test (Reference 5) the "hot" panel outlet temperatures were measured to be approximately 160°F. The baseline thermal model predicted a maximum of 103°F (Reference 5) for the test conditions. This can be attributed to the fact that the explicit finite-difference technique used for temperature calculations in the baseline analysis, in combination with a large computing time increment, does not provide for the proper propagation of a temperature front in a flowing tube. Therefore, the hot fluid in the radiator panel was not propagated to the radiator outlet. In the AB2RAD routine, an implicit backward-difference technique is used to calculate temperatures. This method inherently provides for the propagation of a temperature front. Thus, in this case the AB2RAD prediction may actually better represent the radiator performance.

Figure 12 indicates that the AB2RAD predicted outlet temperatures for the redundant system are higher than the baseline predictions. During the baseline analysis, the low load heater was inadvertently deactivated. When the redundant system outlet falls below 47°F, the heater should be activated to increase the inlet temperature. In the AB2RAD analysis, the heater controls the outlet to 45-47°F, but the baseline predicted outlet reaches a minimum of 37.5°F with no heater. This is also reflected in the heat rejection comparison (Figure 11). The AB2RAD maximum heat rejection is reduced by the heater increasing the redundant system outlet temperature.

During the deep space transient (Figure 20) the panel outlet temperatures falls below -15°F and the low-load heater is activated. However, the AB2RAD predictions react much faster to the low-load heater than the baseline analysis. As the radiator inlet temperature is increased by the

— Reference 4
 AB2RAD

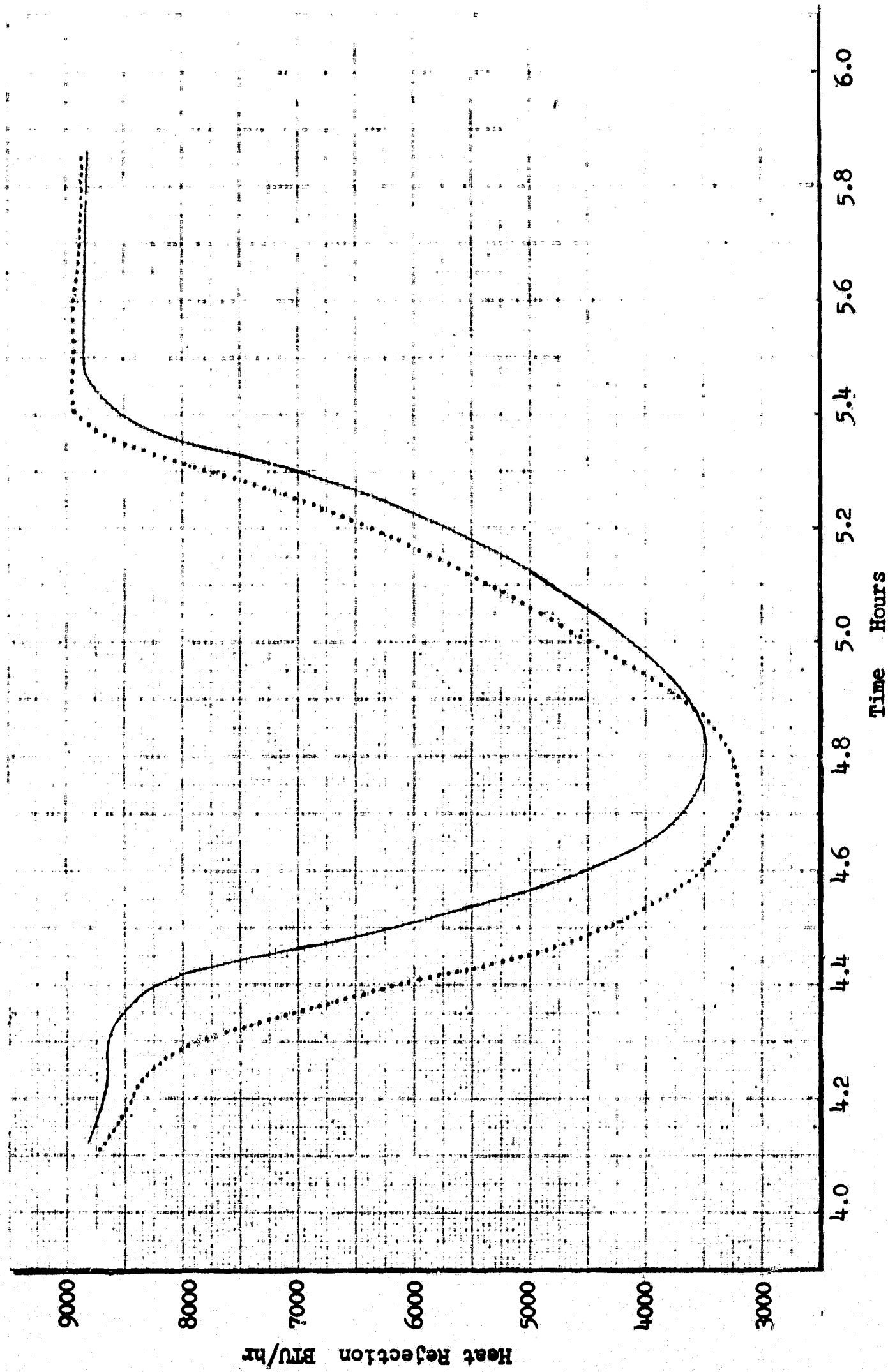


FIGURE 5 COMPARISON OF PREDICTED HEAT REJECTION IN LUNAR ORBIT; HEAT LOAD = 8500 BTU/HR

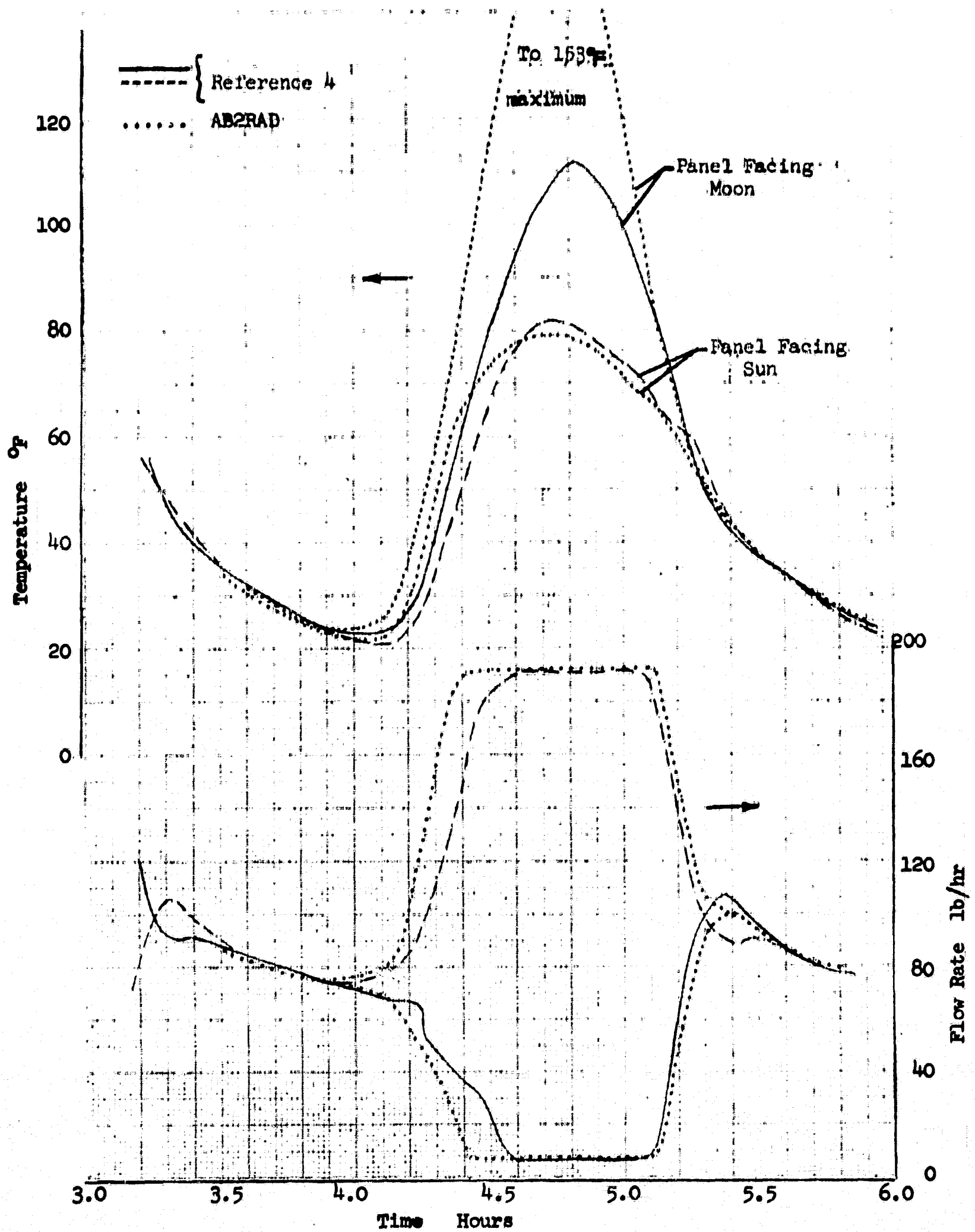


FIGURE 6 COMPARISON OF PREDICTED PANEL FLOW RATES AND OUTLET TEMPERATURES IN LUNAR ORBIT; HEAT LOAD = 8500 BTU/HR

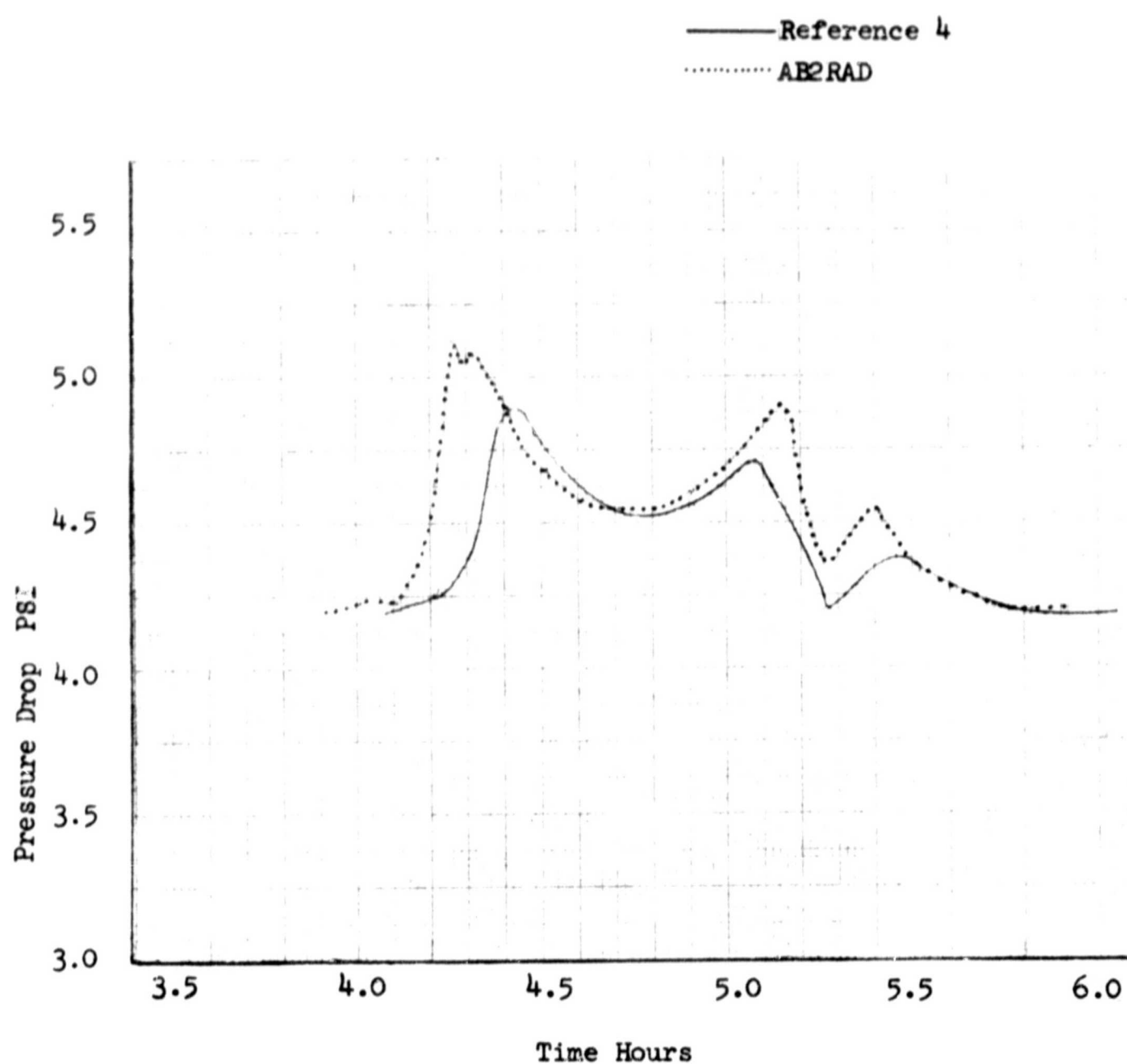


Figure 7 Comparison of Predicted Pressure Drops in Lunar Orbit; Heat Load = 8500 BTU/hr

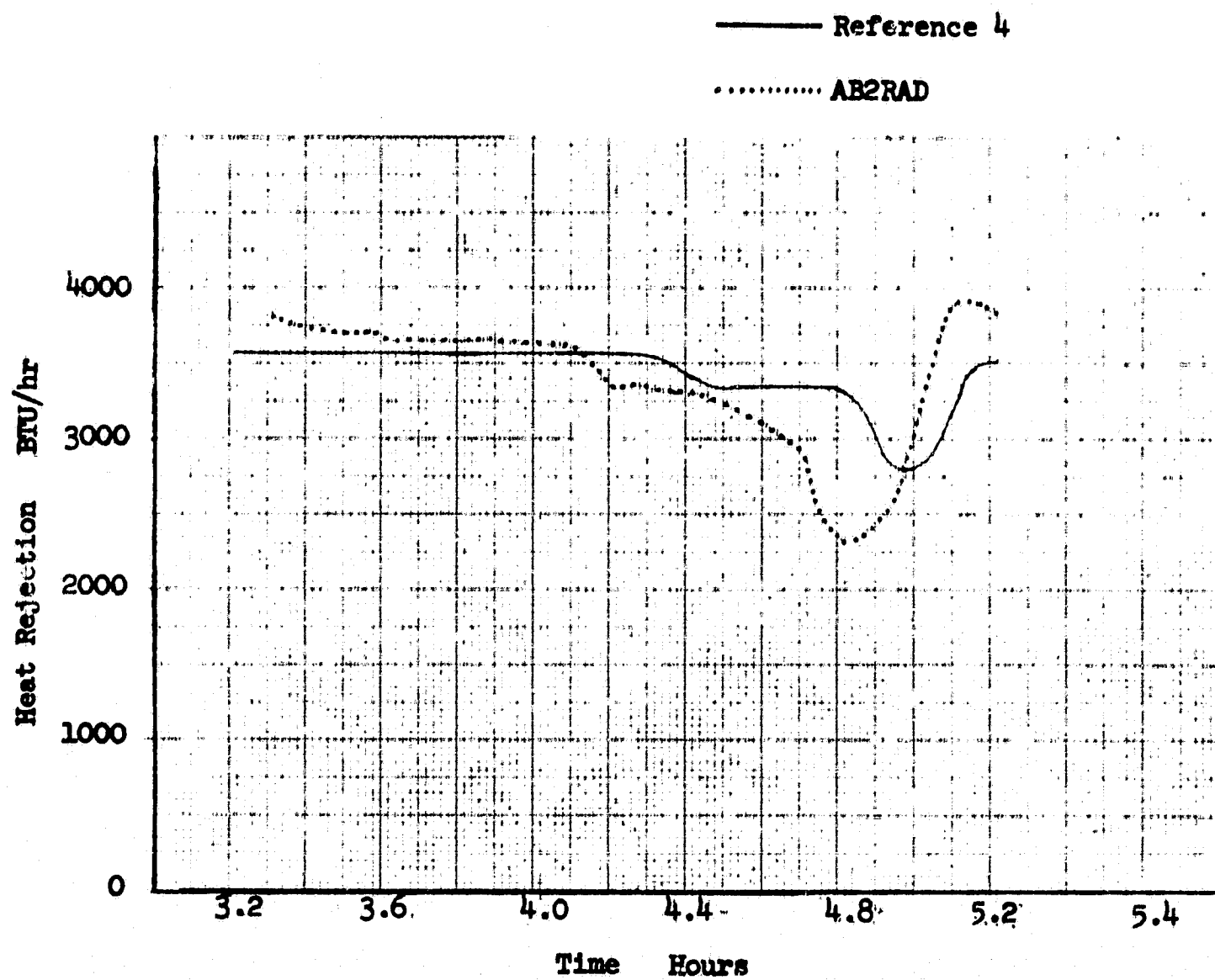


FIGURE 8 COMPARISON OF PREDICTED HEAT REJECTION IN LUNAR ORBIT; HEAT LOAD = 3470 BTU/HR

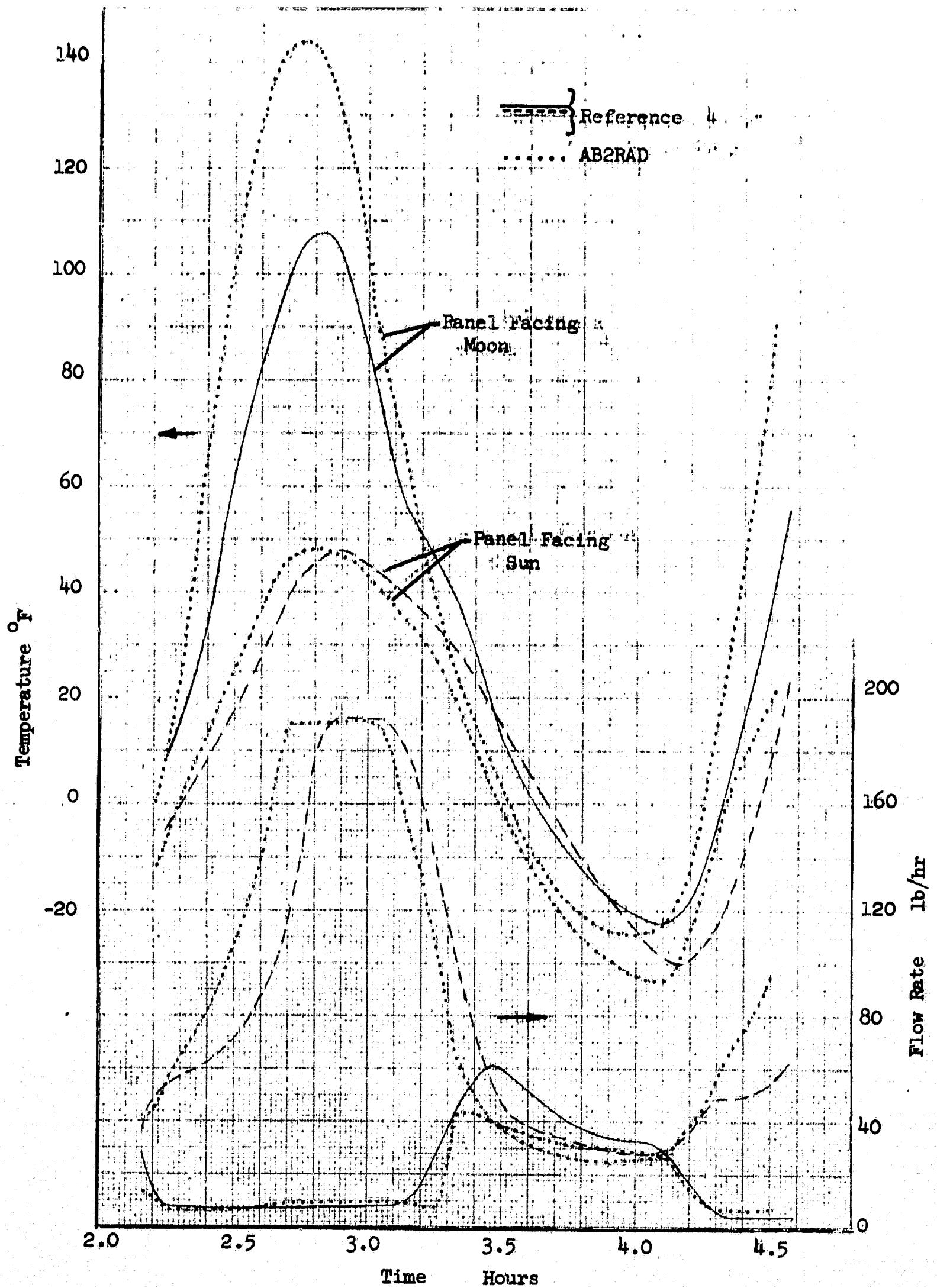


Figure 9 Comparison of Predicted Panel Flow Rates and Outlet Temperatures in Lunar Orbit; Heat Load = 3470 BTU/hr

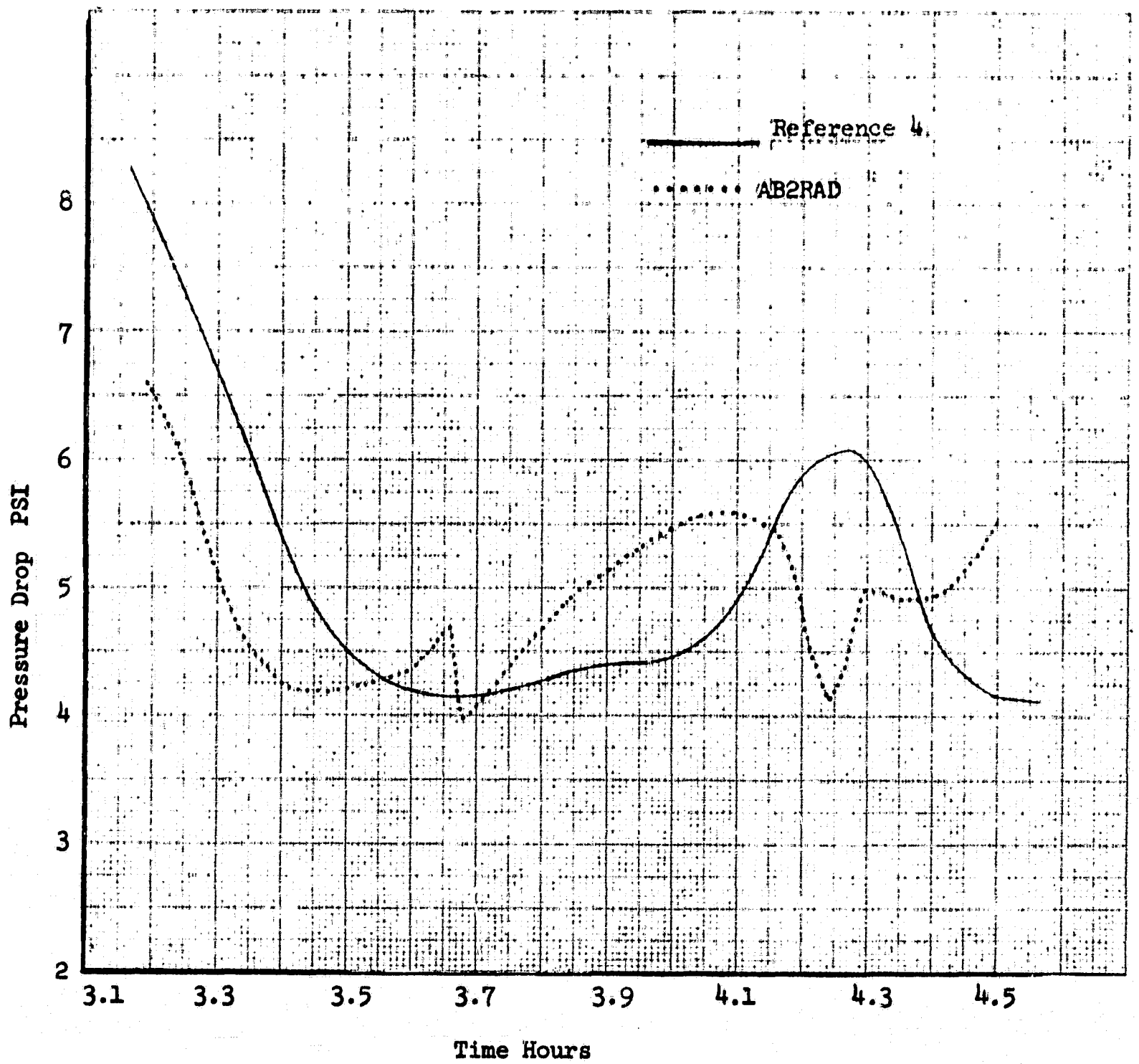


Figure 10 Comparison of Predicted Pressure Drops in Lunar Orbit; Heat Load = 3470 BTU/hr

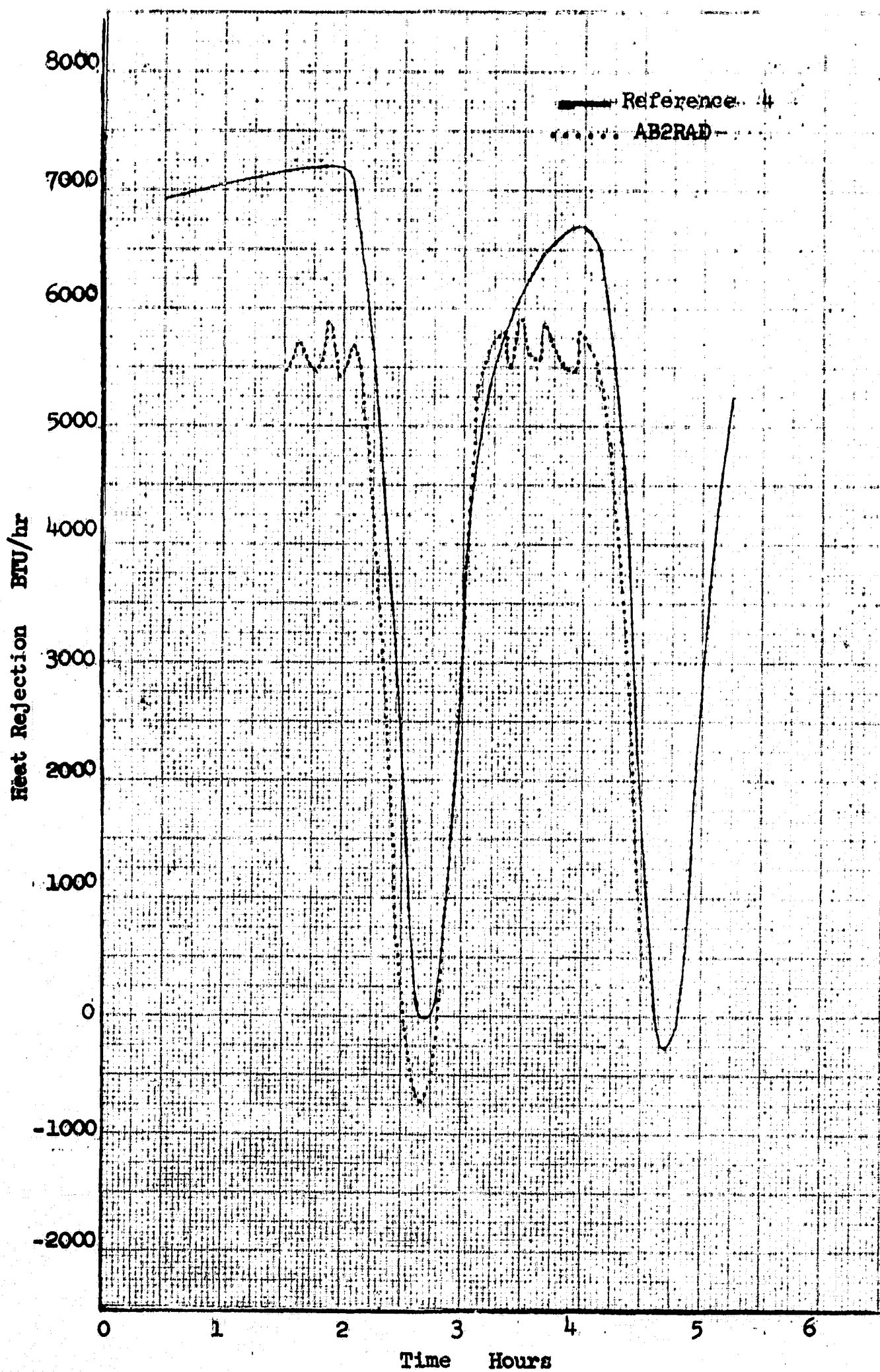


FIGURE 11 COMPARISON OF PREDICTED HEAT REJECTION IN LUNAR ORBIT;
SINGLE PANEL AND REDUNDANT SYSTEM OPERATION, HEAT
LOAD = 5600 BTU/HR

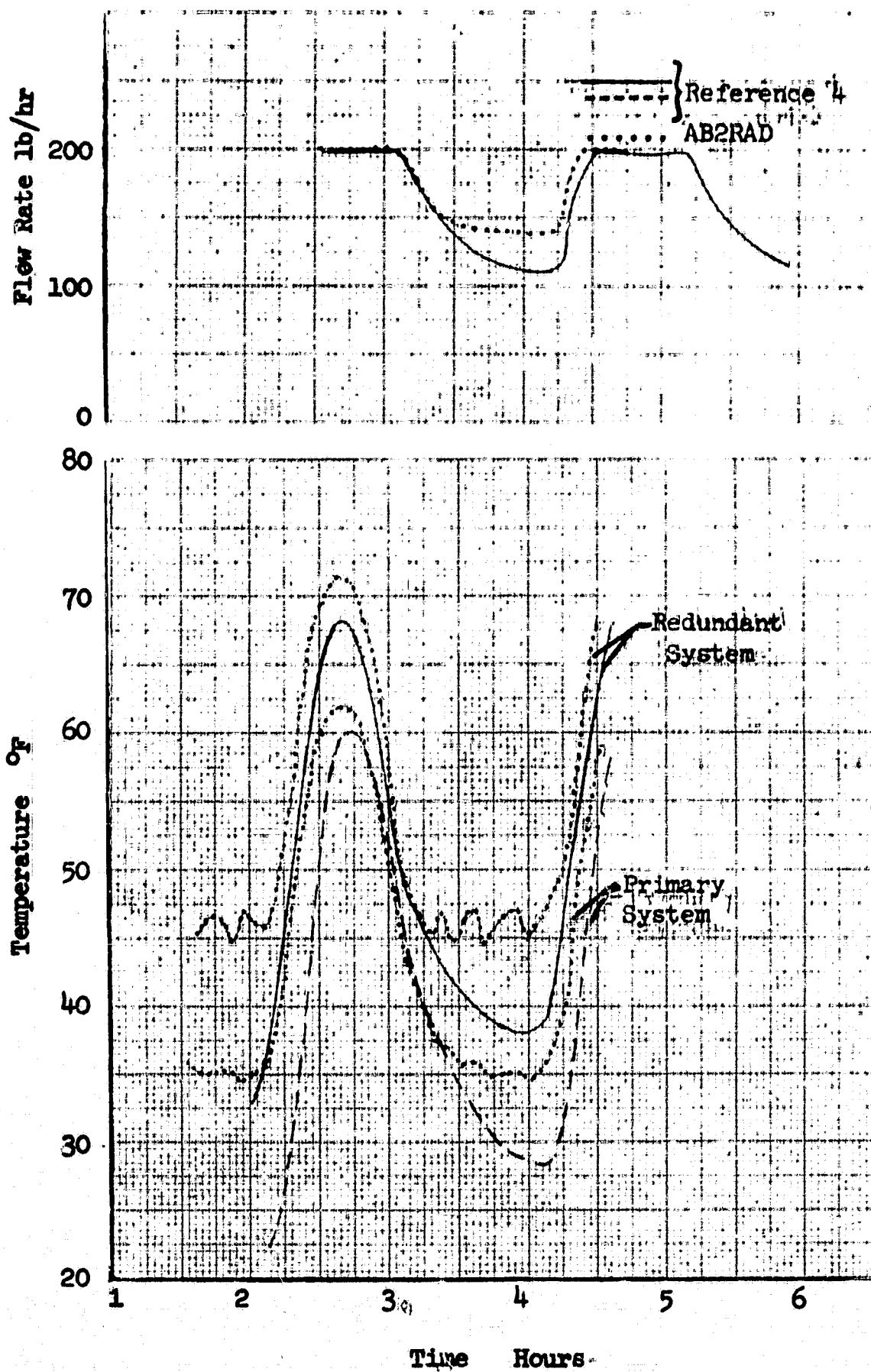


FIGURE 12 COMPARISON OF PREDICTED PANEL FLOW RATES AND OUTLET TEMPERATURES IN LUNAR ORBIT; SINGLE PANEL AND REDUNDANT SYSTEM OPERATION, HEAT LOAD = 5600 BTU/HR

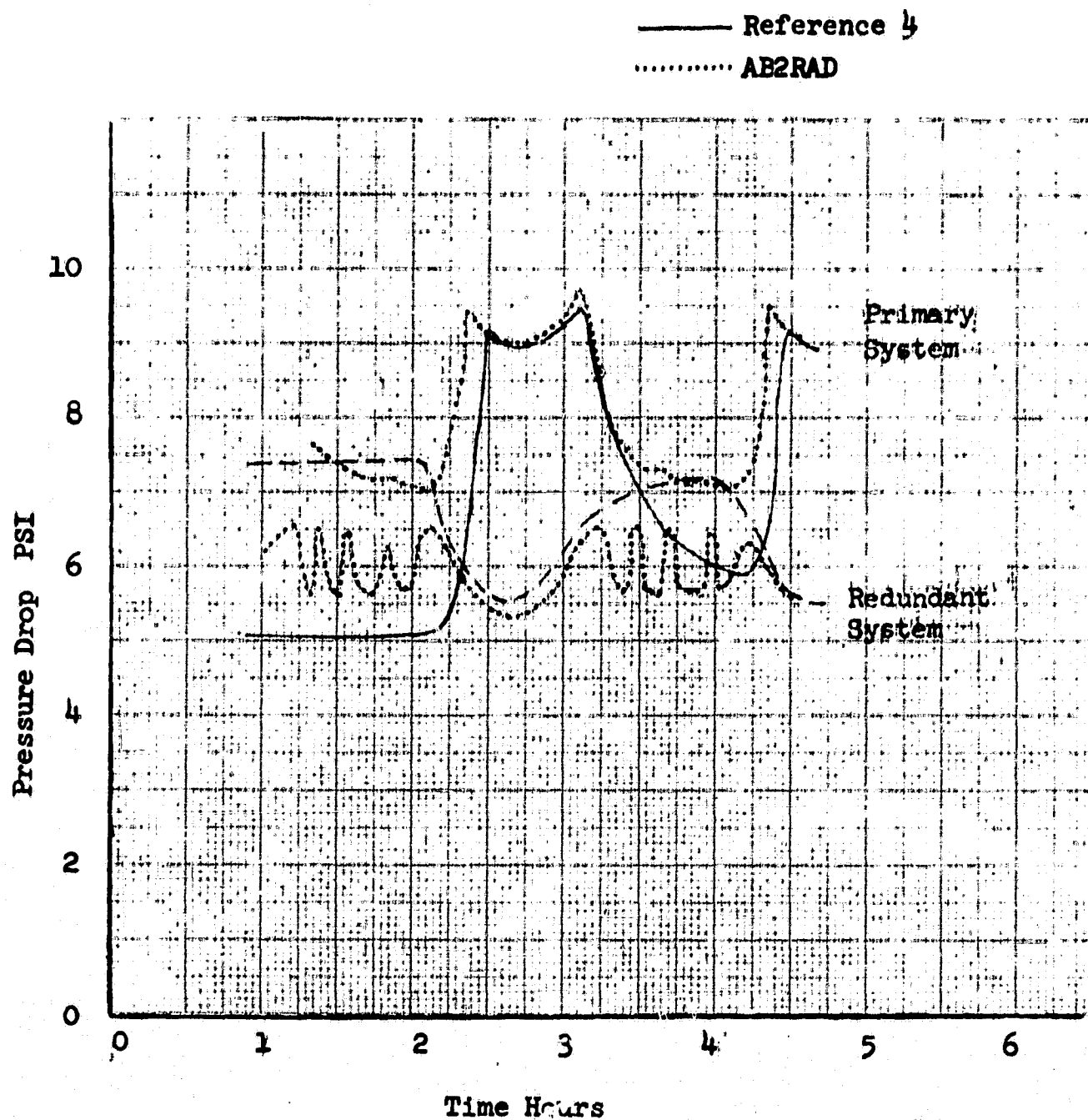


Figure 13 Comparison of Predicted Pressure Drops in Lunar Orbit; Single Panel and Redundant System Operation, Heat Load = 5600 BTU/hr

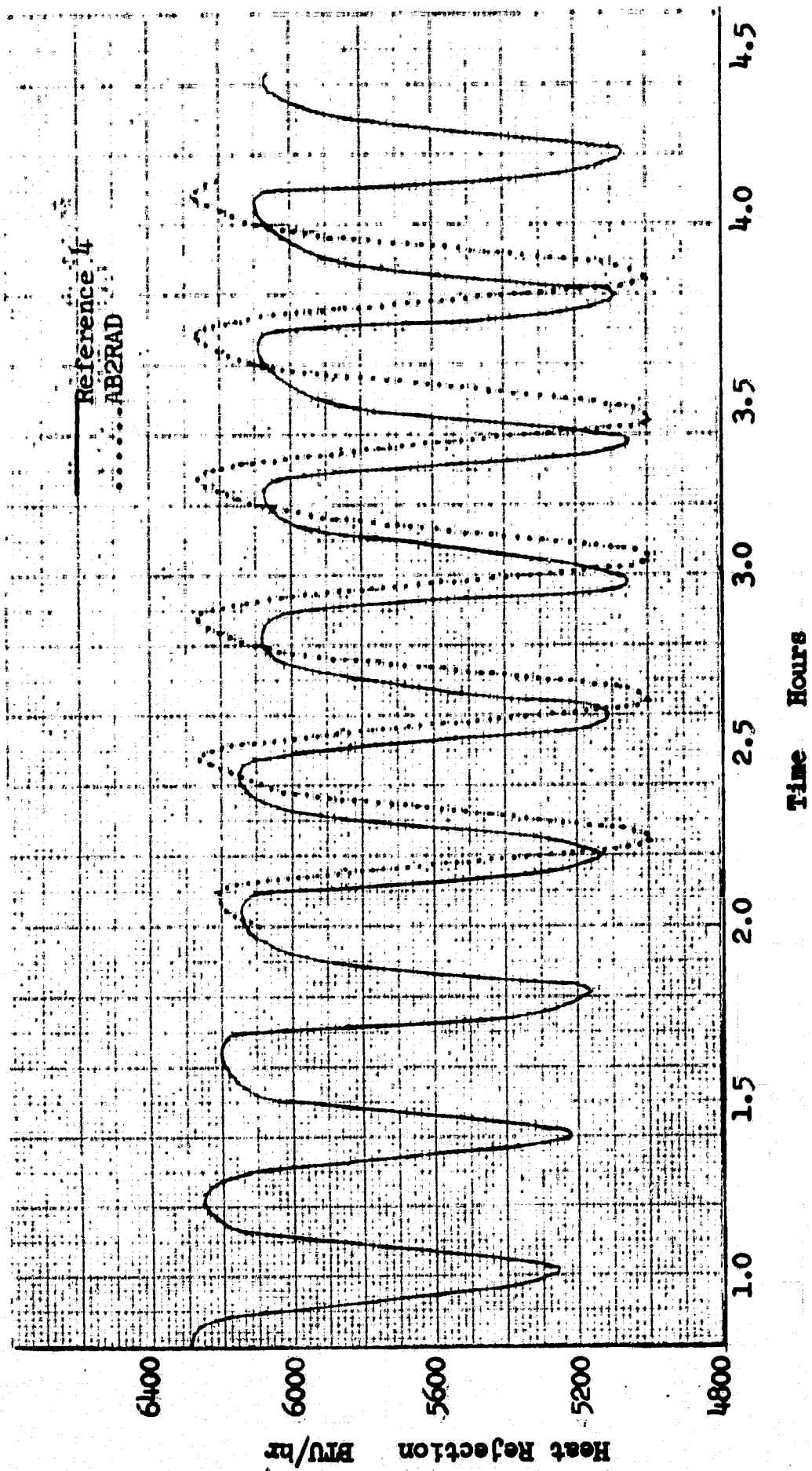


FIGURE 14 COMPARISON OF PREDICTED HEAT REJECTION IN TRANSJUNAR THERMAL CYCLE;
SINGLE PANEL AND REDUNDANT SYSTEM OPERATION

— Reference 4

..... AB2RAD

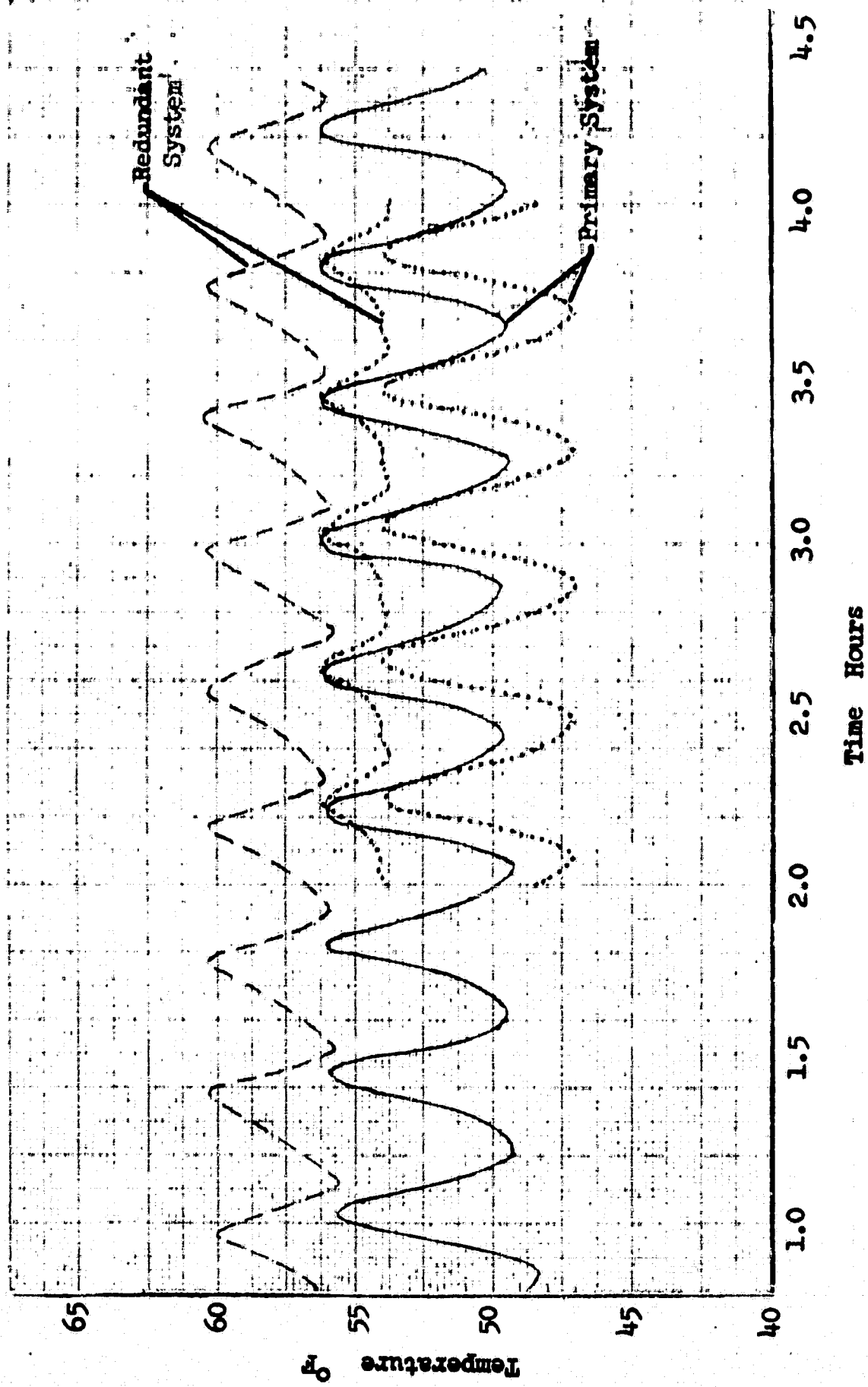


FIGURE 15 COMPARISON OF PREDICTED OUTLET TEMPERATURES IN TRANSILUNAR
THERMAL CYCLE: SINGLE PANEL AND REDUNDANT SYSTEM OPERATION

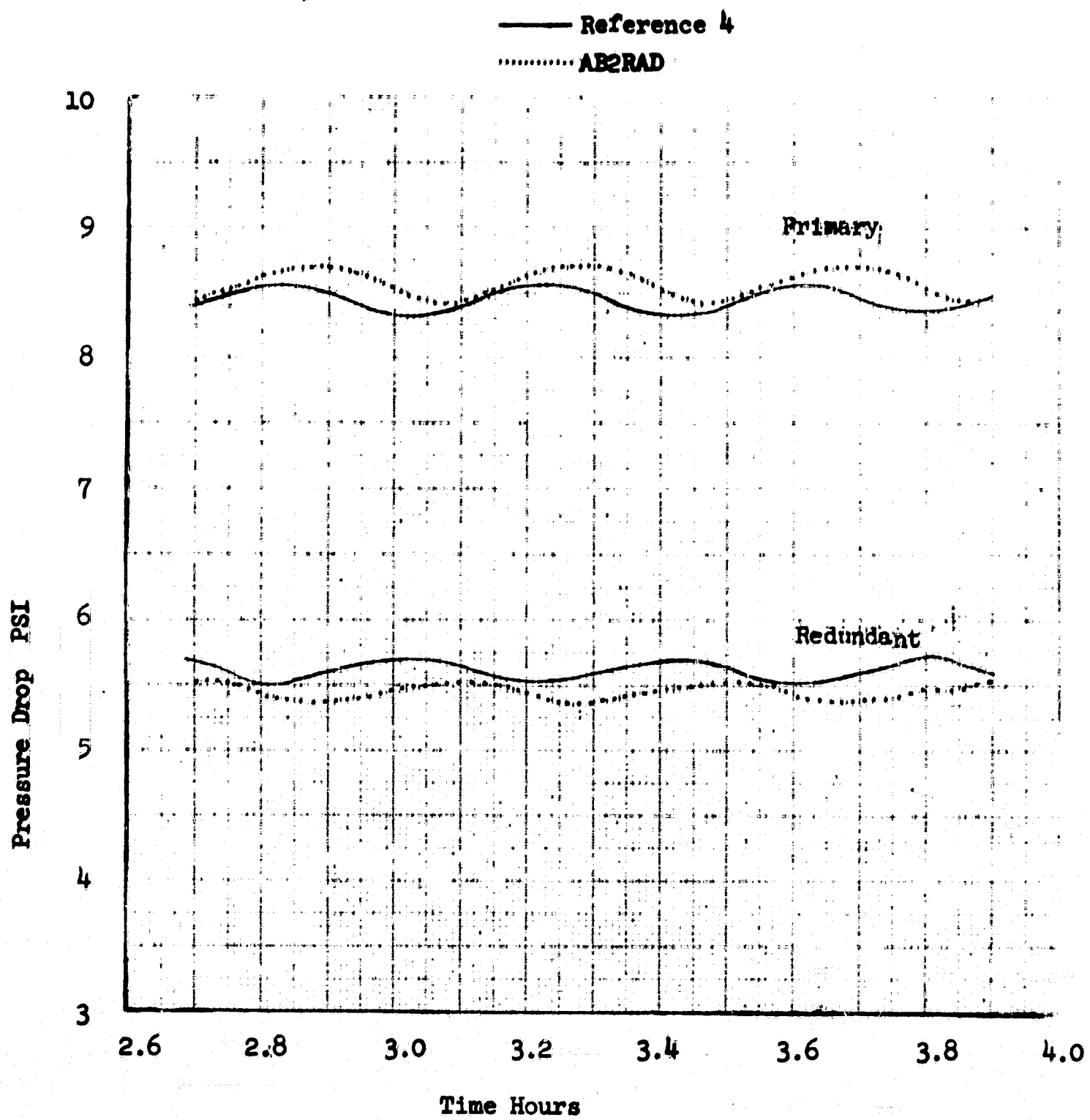


Figure 16 Comparison of Predicted Pressure Drops in Translunar Thermal Cycle; Single Panel and Redundant System Operation

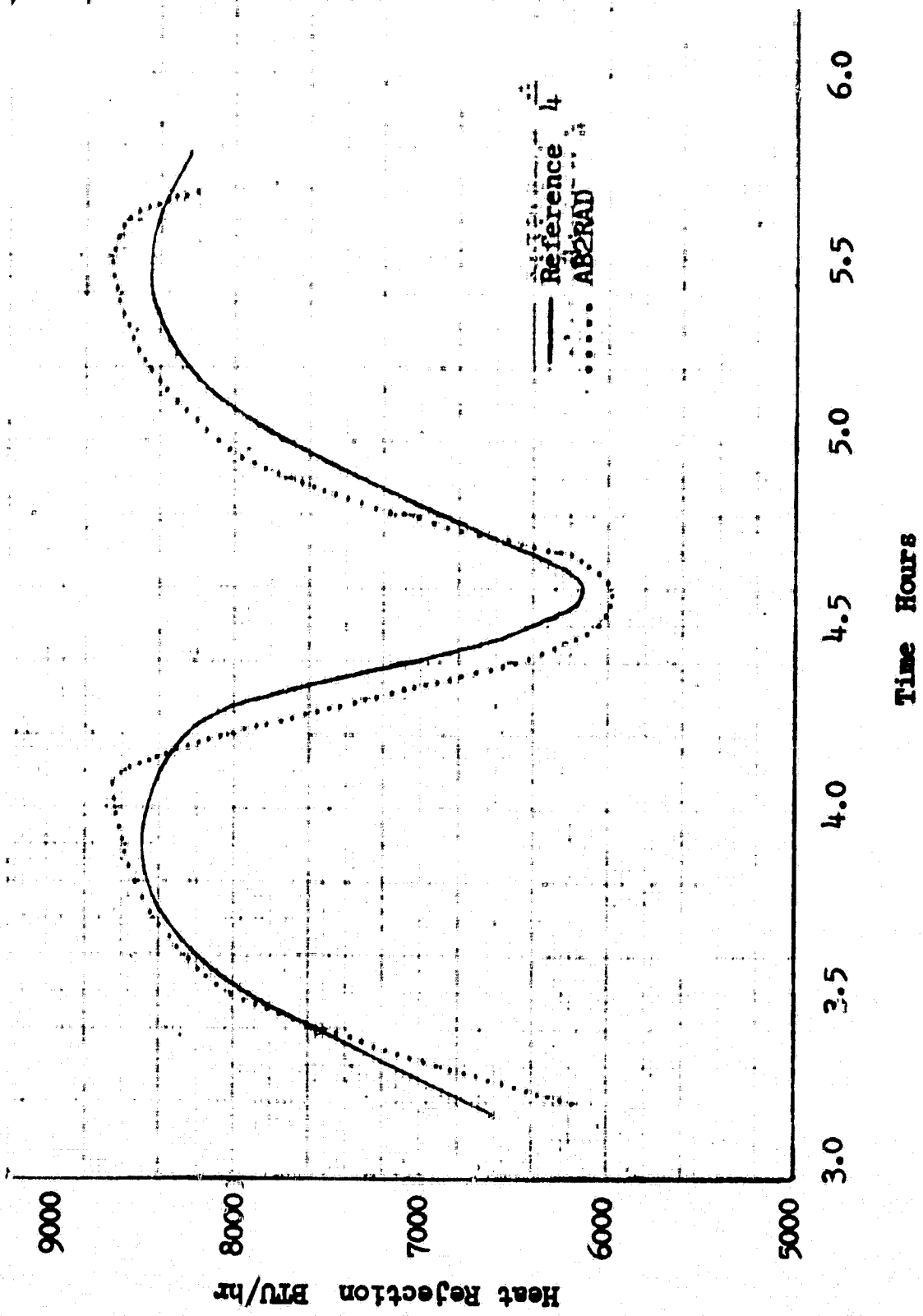


FIGURE 17 COMPARISON OF PREDICTED HEAT REJECTED IN EARTH ORBIT

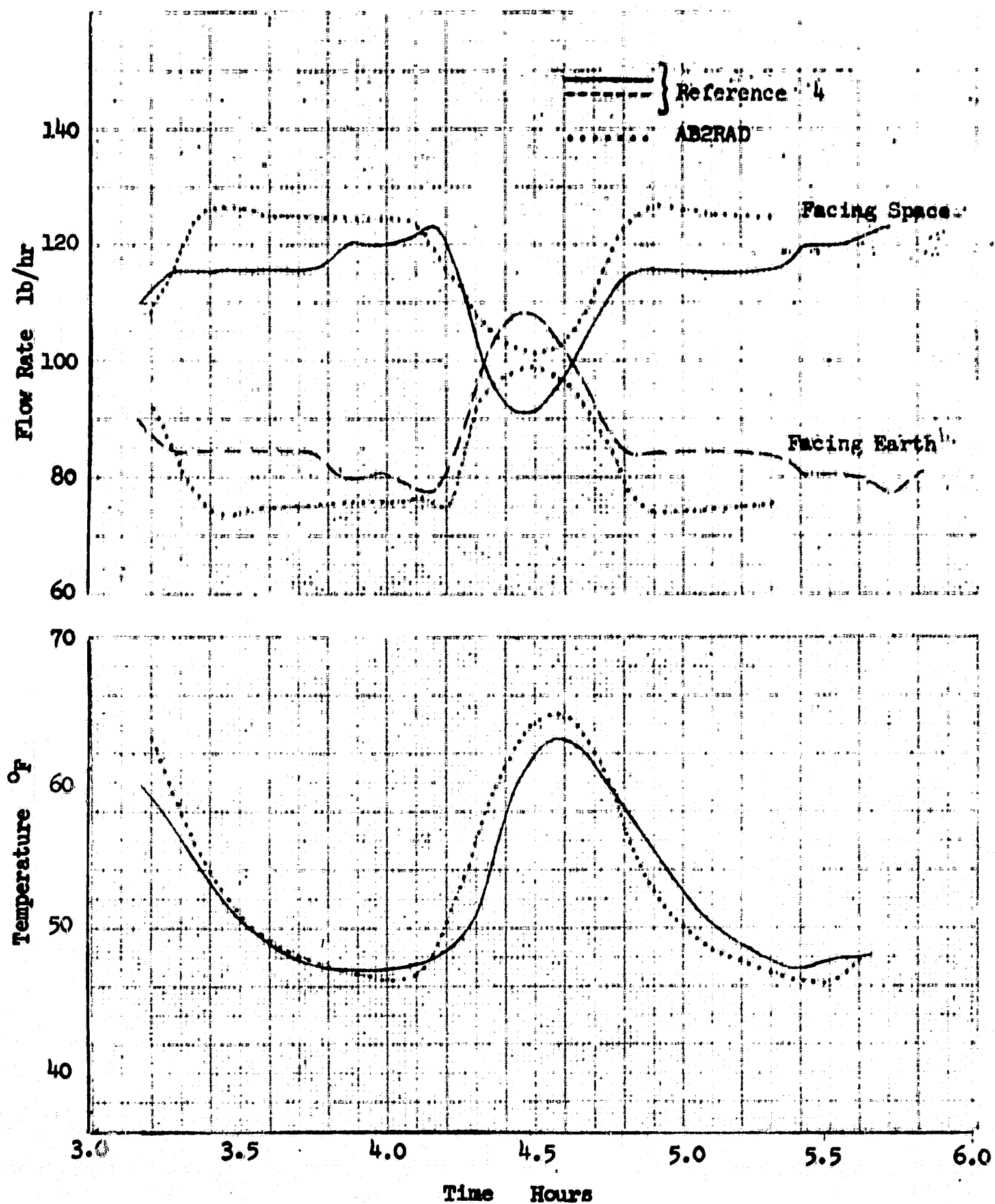


FIGURE 18 COMPARISON OF PREDICTED PANEL FLOW RATES AND
OUTLET TEMPERATURE IN EARTH ORBIT

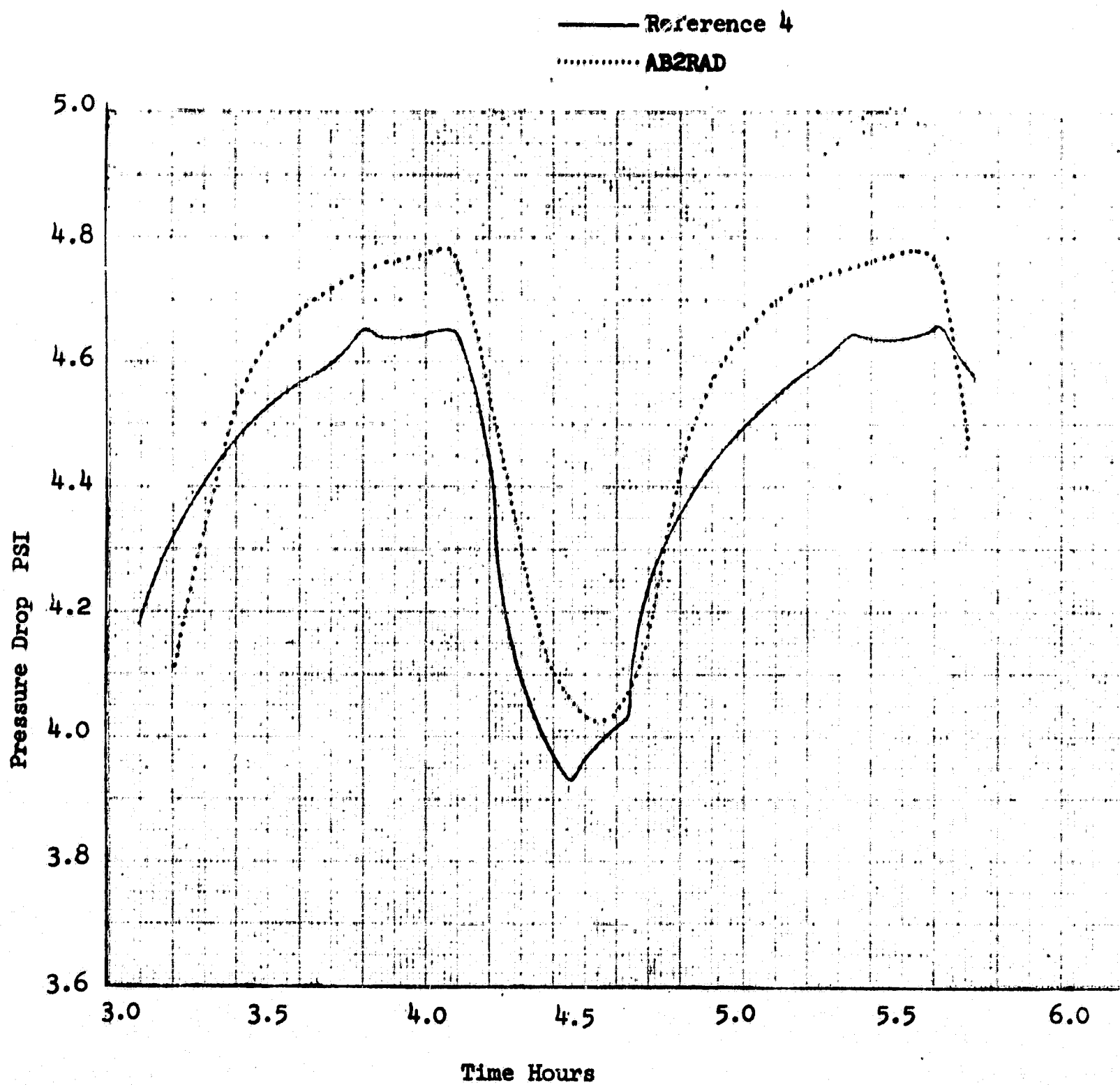


Figure 19 Comparison of Predicted Pressure Drops in Earth Orbit

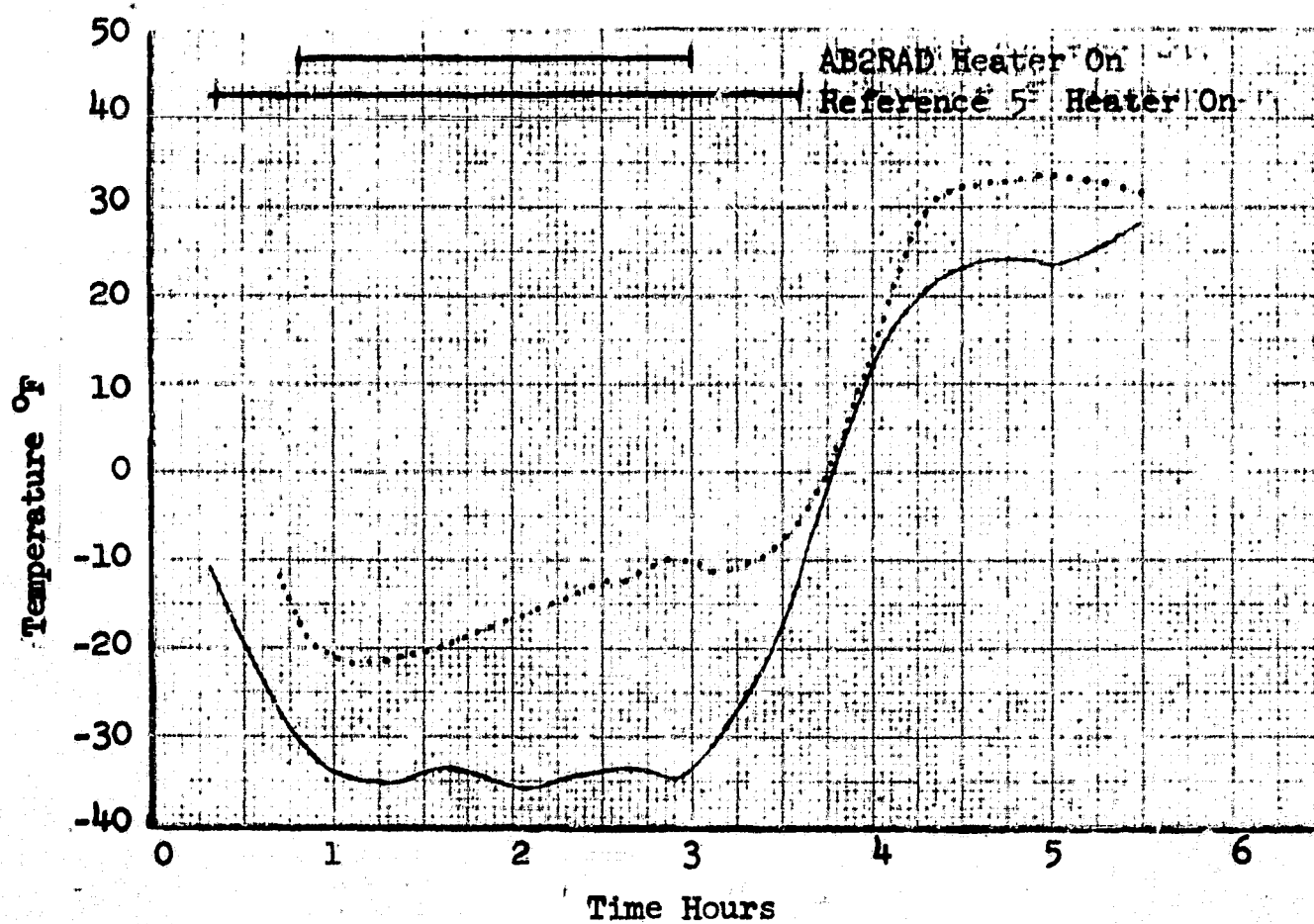
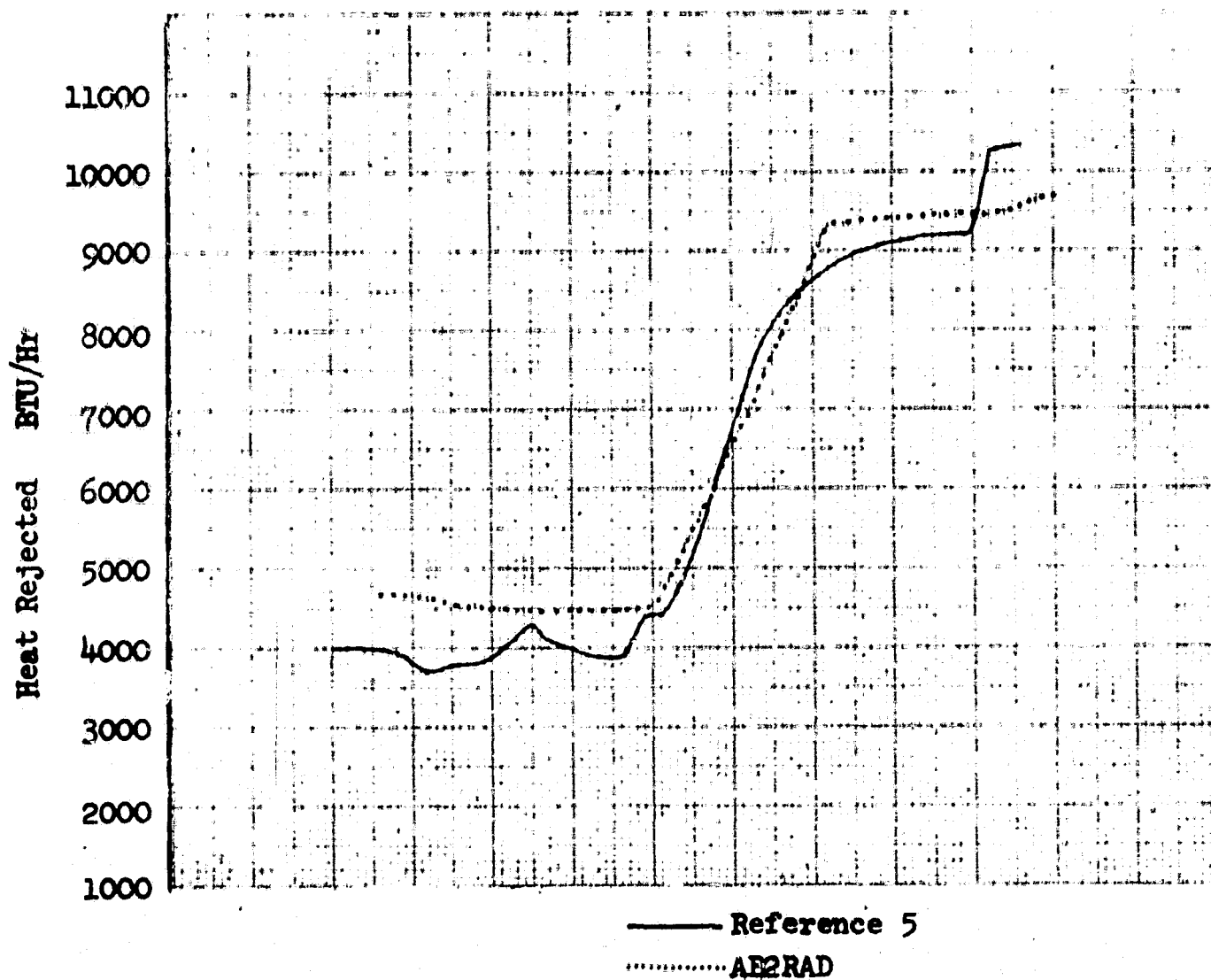


Figure 20 . Comparison of Predicted Heat Rejection and Outlet Temperatures For a Deep Space Transient

Table 2 Comparison of AB2RAD and Baseline Results						
Condition	Lunar Orbit Heat Load = 3470 BTU/hr	Lunar Orbit Heat Load = 8500 BTU/hr	Lunar Orbit Heat Load 5600 BTU/hr	Translunar Heat Load 6880 BTU/hr	Earth Orbit Heat Load = 8800 BTU/hr	Deep Space Transient
Error in Maximum Heat Rejection, %	+10.4	+1.71	-11.9	+2.95	+2.49	-7.0
Error in Minimum Heat Rejection, %	-17.8	-8.05	NA	-1.18	-1.46	+15.4
Error in Average Heat Rejection, %	-3.3	-4.05	-18.8	+ .11	- .02	NA
Water Boiling Rate Baseline AB2RAD error	.24 lbs/orbit .461 lbs/orbit + .221	4.35 lbs/orbit 4.65 lbs/orbit + .30	1.74 lbs/orbit 1.67 lbs/orbit - .07	1.16 lbs/hr 1.065 lbs/hr -.095	2.86 lbs/orbit 2.28 lbs/orbit - .58	0. 0. 0.
Primary System Error in Maximum Pressure Drop, %	-8.2	+3.66	+ .15	+1.16	+2.8	--
Error in Minimum Pressure Drop, %	-3.6	+3.58	+10.2	+1.2	+2.55	--
Error in Average Pressure Drop, %	-2.4	+2.2	+12.6	+1.42	+2.0	--
Redundant System Error in Maximum Pressure Drop, %	NA	NA	-8.33	-2.63	NA	NA
Error in Minimum Pressure Drop, %	NA	NA	-2.73	-3.6	NA	NA
Error in Average Pressure Drop, %	--	--	-10.73	-3.1	--	--
Low Load Heater Power Dissipation, BTU	789 ² 936.35 +147.35	0.0 0.0 0.0	0.0 ¹ 890.3 ~	0.0 0.0 0.0	0.0 0.0 0.0	4919.9 3346.3 -1563.7

¹ Redundant System Heater Only. Not Included in Baseline Analysis.

² Per Orbit

heater the AB2RAD predicts an increase in the outlet temperature; whereas, the baseline predictions show the outlet temperature being maintained at approximately -35°F during the minimum heat load conditions. The sensitivity of the AB2RAD model to the low-load heater could cause errors in the predicted heater power consumption. If the minimum heat load conditions had continued for longer than three hours, the AB2RAD would have predicted that the low-load heater would cycle on and off. As indicated by Table 2, the AB2RAD predicts 31 per cent less heater power consumption than the baseline for a 3.0 hour minimum heat load condition. The degradation and transient to the 9275 BTU/hr heat load condition is adequately predicted by AB2RAD.

The AB2RAD predictions have been compared to detailed thermal model predictions which have been verified by predicting test results (Reference 5). Environmental conditions of deep space, solar heating, and lunar heating with the expected maximum and the minimum heat loads for the Block II ECS radiator have been considered. All active controls (bypass valve, proportioning valve, isolation valve, and low load heater) have been exercised. Both the primary and redundant systems have been operated. In conclusion, the AB2RAD has been shown to provide adequate Block II ECS radiator performance predictions for a wide variety of conditions with a minimum of computer time required. With the limitations discussed above, and summarized in Table 2, the AB2RAD can be utilized to predict the performance of the Apollo Block II ECS radiator under any combination of heat load and environment.

5.0 USER'S INSTRUCTIONS

5.1 Program Description

AB2RAD is written in Fortran V for the Univac 1108 digital computer and requires a total of 24,773 words of core storage, including the required system routines. Table 3 lists the Univac 1108, Fortran V, system subroutines which are used by AB2RAD. The square root routine (Table 3) is required regardless of the system on which the program is run. Input data comprises the majority of the storage requirements. Storage space is reserved for four incident heat tables, two inlet temperature tables and two flow rate tables. Each table is dimensioned for 1000 values of the dependent and independent variables for a total data storage requirement of 16000 words. A complete program listing is given in Appendix A. Appendix B presents a program flow chart. The major Fortran terms used in the routine are given in Appendix C.

5.2 Data Preparation

For submitting runs under the NASA Exec II Processor, the configuration of the card deck is as shown in Figure 21. If the AB2RAD source deck is submitted with the data the deck arrangement is as shown in Figure 21a. Figure 21b shows the deck configuration when the AB2RAD program is read from a magnetic tape.

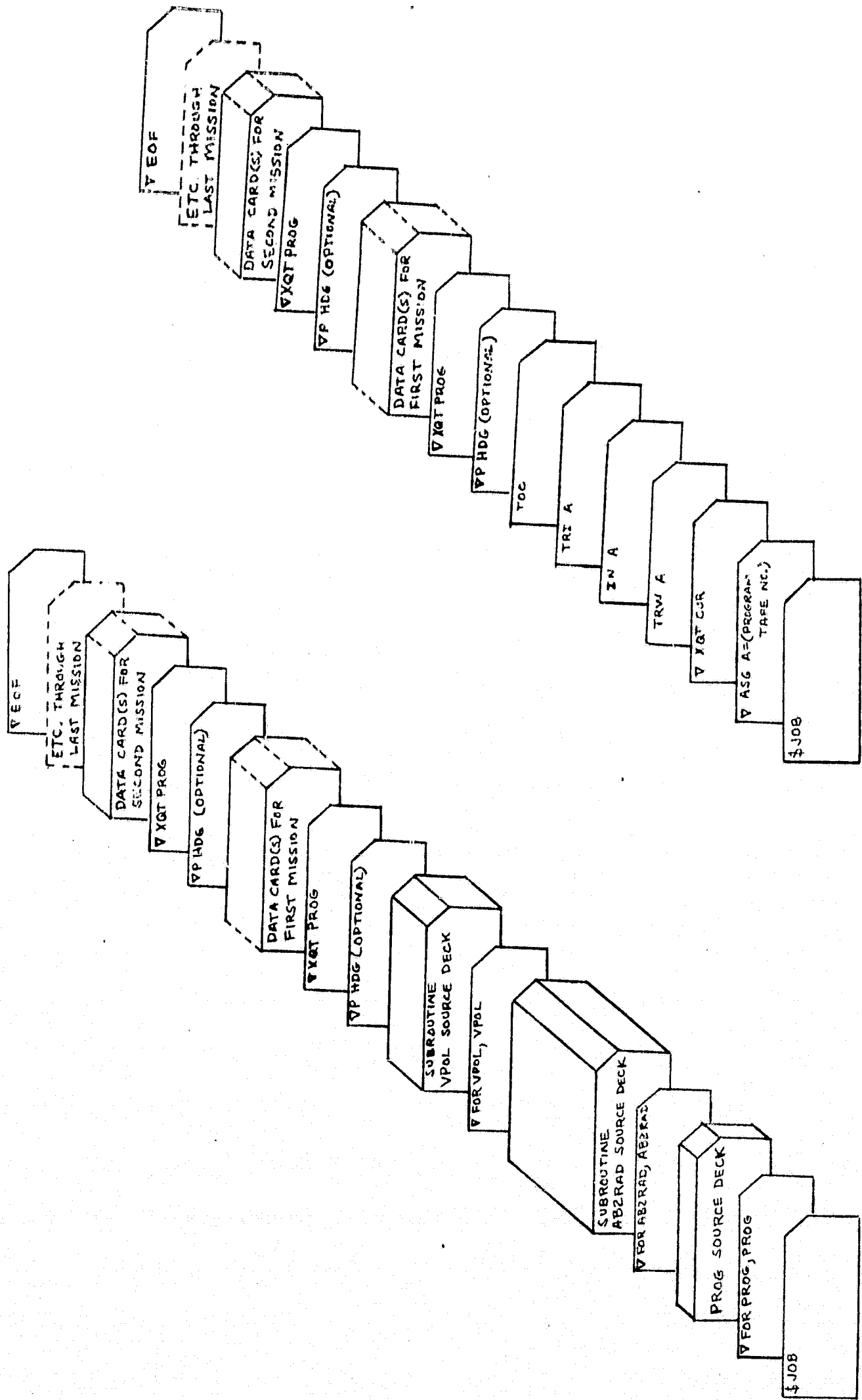
The data input consists of a mission parameter card and curve cards where required. An optional HDG control card may be used for problem identification if desired (see Figure 20). The mission parameter card specifies which heat flux data is to be used or if the heat flux data will be input, the primary and redundant loop operation code, mission time if required, print interval, inlet temperature and flow rate. The curve cards provide for tabular inputs of absorbed heat, flow rates and inlet temperatures as a function of time.

5.2.1 Mission Parameter Card

<u>Columns</u>	<u>Fortran Nomen- clature</u>	<u>Format</u>	<u>Description</u>
<u>Card 1</u>			
1-5	MSSION	I5	Mission Code = 1, lunar-oriented broadside = 2, lunar-oriented nose down = 3, translunar broadside - 1 RPH Thermal Cycle = 4, zero incident heat (Steady state only) = 5, mission defined by time dependent curves

Table 3 System Subroutines Used

1. EXIT
2. NERR2\$
3. NRDU\$
4. NIO1\$
5. NIO2\$
6. NWDU\$
7. SQRT
8. NERR3\$
9. NSTOP\$
10. NFTV\$
11. WOTIN\$
12. FPACK\$
13. DEPTH
14. NERR\$
15. NIOIN\$
16. NINPT\$
17. FLOATX
18. NEXP\$X
19. NTAB\$
20. CONVIX
21. NININ\$



(a) Source Deck Configuration

(b) Program Tape Configuration

FIGURE 21 RUN SUBMISSION CARD DECK CONFIGURATION

<u>Columns</u>	<u>Fortran Nomen- clature</u>	<u>Format</u>	<u>Description</u>
6-8	KODE	I5	Primary System Code = 0, both primary panels on = -1, side 3,4 only on = 1, side 1,2 only on = 2, primary system off
9-10	KODSEC	I5	Redundant System Code = 0, redundant system off = 1, redundant system on
11-20	P3	F10.0	Total mission time, not required for missions 1 through 4, so may be left blank for these missions.
21-30	PERIOD	F10.0	Cyclic period for which the time dependent curves will repeat until total mission time is reached. Required only for Mission 5. May be left blank if non-cyclic curves are supplied for entire mission time. Not required for Missions 1 - 4, so may be left blank for these conditions.
31-40	PRINT	F10.0	Print interval, hrs. Must be integral multiple of calculation interval (.02 hr). For printing every iteration, any value $\leq .02$ (including zero or blank) will suffice. For Mission 4, a value of 6.0 will print only steady state values.
41-50	TINLT	F10.0	Primary system inlet fluid temperature, °F; not required for Mission 5 or if KODE = 2, so may be left blank for these conditions.
51-60	WDOTS	F10.0	Primary system total flow rate, lb/hr; not required for Mission 5 or if KODE = 2, so may be left blank for these conditions.
61-70	TINSEC	F10.0	Redundant system inlet fluid temperature, °F; not required for Mission 5 or if KODSEC = 0, so may be left blank for these conditions.
71-80	WDTSEC	F10.0	Redundant system total flow rate, lb/hr; not required for Mission 5 or if KODSEC = 0, so may be left blank for these conditions.

Card 1 constitutes the entire data set for Missions 1 through 4; for Mission 5, however, curves must be supplied as described below.

5.2.2 Curve Cards

The following curve sets (curve header card followed by curve data cards) must be supplied in the order given:

1. Absorbed heat curve for Panel 1, $\text{BTU/hr-ft}^2 = f(\text{hours})$
2. Absorbed heat curve for Panel 2, $\text{BTU/hr-ft}^2 = f(\text{hours})$
3. Absorbed heat curve for Panel 3, $\text{BTU/hr-ft}^2 = f(\text{hours})$
4. Absorbed heat curve for Panel 4, $\text{BTU/hr-ft}^2 = f(\text{hours})$
5. Primary system total flow rate, $\text{lb/hr} = f(\text{hours})$
6. Primary system inlet fluid temperature, $^{\circ}\text{F} = f(\text{hours})$
7. Redundant system total flow rate, $\text{lb/hr} = f(\text{hours})$
8. Redundant system inlet fluid temperature, $^{\circ}\text{F} = f(\text{hours})$

The first six curves must be supplied for all problems. If the redundant system is not used ($\text{KODSEC} = 0$), curves 7 and 8 must not be included. For conditions when the primary system is not operating a dummy curve 5 and 6 must be supplied. A minimum of two points must be used for each curve and the last time point must be equal to or exceed the problem mission time.

<u>Columns</u>	<u>Fortran Nomen- clature</u>	<u>Format</u>	<u>Description</u>
<u>Card 1 (Curve Header Card)</u>			
1-5	NPTS	I5	Number of points on curve, $2 \leq \text{NPTS} \leq 1000$
6-72	ALPHA	11A6,A1	Title
<u>Cards 2 through 2NPTS/7 (Curve Data Cards)</u>			
1-10	TIME1	E10.3	Initial time point = 0.
11-20	TIME2	E10.3	Second time point, hr.
21-30	TIME3	E10.3	Third time point, hr.
Etc. through last time point, which must be equal to or greater than total mission time, P3. Then beginning in next ten-column field,			
	Q, WDTTOT, or TIN	E10.3	Initial dependent variable value, BTU/hr-ft^2 , lb/hr , or $^{\circ}\text{F}$, respectively.
Etc. through last dependent variable value.			

5.3 Output

During simulation of a mission, current values of heat rejection, pressure drop, flow rate, and fluid outlet temperature will be printed at

times specified by the input print interval. The print interval must be an integral multiple of the calculation interval (.02 hr) or irregular print intervals will result. Upon completion of Mission 4 the steady state values of the output parameters are printed. At the end of all other missions the maximum, minimum, and average values of the output parameters are printed. For Missions 1 through 3, these will represent values encountered during the last orbit and for Mission 5, values over the entire mission including the initial conditions.

5.4 Error Diagnostics

If certain errors occur during execution of the program, diagnostic messages will be printed before execution is terminated. As an aid to error tracing, these messages are listed below with explanatory remarks.

1. INTERPOLATION IMPOSSIBLE

MERR = x TIME = x.xx
EXECUTION TERMINATED BY PROGRAMMED HALT

The problem time has exceeded the times supplied with one of the curves, the specific curve being indicated by the value of MERR as follows:

<u>Value</u>	<u>Curve</u>
1	Absorbed heat for Panel 1
2	Absorbed heat for Panel 2
3	Absorbed heat for Panel 3
4	Absorbed heat for Panel 4
5	Primary system total flow rate
6	Primary system inlet fluid temperature
7	Redundant system total flow rate
8	Redundant system inlet fluid temperature

2. THE PROBLEM COULD NOT BE SOLVED IN 500 ITERATIONS

Five hundred relaxation passes through the temperature equations failed to produce a solution accurate to within 0.002°F.

3. THE TEMPERATURE USED IN FINDING THE VISCOSITY IS GREATER THAN THE HIGHEST VALUE ON THE CURVE

Fluid temperature has become greater than 300°F at some point in the system.

4. THE TEMPERATURE USED IN FINDING THE VISCOSITY IS LESS THAN THE LOWEST TEMPERATURE ON THE CURVE

Fluid temperature has become less than -300°F at some point in the system.

5. THREE HUNDRED ITERATIONS HAVE FAILED TO PRODUCE A STEADY STATE SOLUTION

A Mission 4 problem has failed to reach steady state in 300 iterations.

6.0	LIST OF SYMBOLS
A_c	Conduction area
A_e	External area for radiation
A_f	Internal area for heat transfer
AK	Ratio of pressure drop in a tube to flow rate in that tube
B	Pressure drop constant = $\frac{2(\text{tube length})(\text{Wetted Perimeter})^2}{(\text{Flow area})^3 (\text{Fluid density})}$
c	Specific heat
D_h	Hydraulic diameter
D,E,F,	Temperature equation coefficients
G	Proportioning valve gain
H	Factor for proportioning valve pressure drop
f	Friction factor
h_f	Heat transfer coefficient
k	Thermal conductivity
K	Ratio of pressure drop to flow rate
ΔP	Pressure drop
Pr	Prandtl Number
Q	Incident heat
Re	Reynolds number
T	Temperature
t_c	Proportioning valve time constant
U	Thermal conductance
w	Weight of lump j
\dot{w}	Flow rate
Y	Conduction path length
X	Proportioning valve position

Z	Fraction bypassed
α	Incident heat absorptivity
ϵ	Emissivity
σ	Stefan-Boltzmann constant
μ	Dynamic viscosity
$\Delta\tau$	Calculation time increment
θ	Iteration limit
ϕ	Overrelaxation parameter

Subscripts

i, j, k	Indices
LT	Left side
f	Fluid
fu	Upstream fluid lump
RT	Right side
s	Structure (fin)
t	Tube

Superscripts

τ	Conditions at time, τ
"	Conditions at time $\tau + \Delta\tau$

REFERENCES

1. Hixon, C. W., "Simplified Transient Computer Subroutines For Apollo Block I and Block II Environmental Control System Radiators," LTV Astronautics Report No. 00.822, 18 July 1966
2. Gaddis, J. L., "Implicit Finite-Difference Generalized Heat Transfer Program (LVVM22)," LTV Astronautics Report No. 00.809, 12 July 1966
3. Finch, H. L., et al, "Orbiting Satellite Surface Temperature Prediction and Analysis," Midwest Research Institute Project No. 2669-E (Contract No. NAS9-1059), 3 February 1964
4. Hixon, C. W., et al, "Apollo Block II Command Module Thermal Simulator," LTV Missiles and Space Division Report No. 350.2 Volume I, 28 July 1967
5. Summerhays, R. M., and Whitten, W. A., "Test Report for Qualification Test of an Apollo Block II ECS Radiator Subsystem," LTV Missiles and Space Division Report No. 332.62, 7 April 1967

APPENDIX A

PROGRAM LISTING

ULTRA-FAST MISSION ANALYSIS ROUTINE
FOR APOLLO BLOCK 2 ENVIRONMENTAL
CONTROL SYSTEM RADIATORS

DEVELOPED BY

MISSILES AND SPACE DIVISION - TEXAS
LTV AEROSPACE CORPORATION
P. O. BOX 6267 - DALLAS, TEXAS 75222

FOR

NATIONAL AERONAUTICS AND SPACE ADMINISTRATION
MANNED SPACECRAFT CENTER - HOUSTON, TEXAS
UNDER CONTRACT NAS9-6807

PROG READS THE DATA AND CALLS SUBROUTINE AB2RAD WHICH DOES
THE ACTUAL CALCULATIONS

```

1  FORMAT(15,13,12,7F10.0)
50  READ(5,1) MSSION,KODE,KODSEC,P3,PERIOD,PRINT,TINLT,WDOTS,
    1TINSEC,WDTSEC
    CALL AB2RAD(MSSION,KODE,KODSEC,P3,PERIOD,PRINT,TINLT,WDOTS,
    1TINSEC,WDTSEC)
    GO TO 50
    END
    SUBROUTINE AB2RAD(MSSION,KODE,KODSEC,P3,PERIOD,PRINT,TINLT,
    1WDOTS,TINSEC,WDTSEC)
    DATA ON/' ON '/,OFF/' OFF '/
    DIMENSION AK(12),BMU(37),TF(32),TFIN(32),IPPF(32),TPF(32),NFCODE
    1(32),TT(32),TTIN(32),TPP(32),TPT(32),NTCODE(32),TS(20),TSIN(20),
    2TPPS(20),TPS(20),NSCODE(20),QABS(4),Q(4000),TIME(8000),T(33),TMU
    3(33),WDOT(10),DPT(10),E(5),ALPHA(12),NPTS(8),WDTTOT(1000),TIN
    4(1000),MSTART(8),WDOTSC(1000),TINLSC(1000),H(10),F(5)
    COMMON T,TMU
503  FORMAT(//50H THE PROBLEM COULD NOT BE SOLVED IN 500 ITERATIONS//)
    SAVE1 = PRINT
    IQUIT=0
2508 AA1 = .034
    A2 = 32.1
    AA2 = AA1
    AA3 = 2.355
    A3A = 1.984
    A5 = .842
    AA5 = 2.55
    A5A = 2.158
    AA6 = AA5
    A7 = .087
    A8 = .0519
    A9 = 1.32E-10
    A10 = .79E-10
    A11 = .41
    A12 = .0911
    A13 = 1.39E-10
    A14 = 1.56
    A15 = .468
    A16 = 1.06
    A17 = .557
    A18 = .0792
    A19 = 1.224E-10

```

BLK20001
BLK20002
BLK20003
BLK20004
BLK20005
BLK20006
BLK20007
BLK20008
BLK20009
BLK20010
BLK20011
BLK20012
BLK20013
BLK20014
BLK20015
BLK20016
BLK20017
BLK20018
BLK20019
BLK20020
BLK20021
BLK20022
BLK20023
BLK20024
BLK20025
BLK20026
BLK20027
BLK20028
BLK20029
BLK20030
BLK20031
BLK20032
BLK20033
BLK20034
BLK20035
BLK20036
BLK20037
BLK20038
BLK20039
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BLK20041
BLK20042
BLK20043
BLK20044
BLK20045
BLK20046
BLK20047
BLK20048
BLK20049
BLK20050
BLK20051
BLK20052
BLK20053
BLK20054
BLK20055
BLK20056
BLK20057
BLK20058
BLK20059
BLK20060

A20 = 1.565
 A21 = .1091
 A22 = .0166E-8
 A23 = .719
 B1 = AA1
 B3 = A3A
 B4 = A5A
 E(1) = .000987407532
 E(2) = .002965444665
 E(3) = .0039182373
 F(4) = .004551234485
 E(5) = .004730380465
 F(1) = .000126203
 F(2) = .000126421111
 F(3) = .000139544253
 F(4) = .000156136337
 F(5) = .000178387671
 WDOTT=WDOTS
 SETPT=504.69
 DBAND=.75
 FLOWMX=1.0
 FLOWMN=.01
 RTFCTR=.0003
 RLIMIT=.0033
 POSIN = 16.5
 VLVGAN=1.155
 POSMIN=.854
 POSMAX=32.146
 FULOPN = 33.
 GFACT=30.
 PPARA=2.0
 VTOL= .001
 COUNT=0.
 SQREJ=0.
 STOUT=0.0
 SDP=0.
 TMX=0.0
 TMN=1000000.0
 TMAX=0.
 TMIN=1000000.
 DPMAX=0.0
 DPMIN=500.0
 PCMAX=0.0
 PCMIN=1000000.0
 IF(KODSEC.EQ.0) GO TO 5060
 SDP2=0.
 DPMX=0.
 DPMN=500.
 TMX2=0.
 TMN2=1000000.
 STOUT2=0.
 5060 XX1=POSIN
 XX2=XX1
 TFA21 = 529.69
 TFA23 = 529.69
 TOUTP = 529.69
 M=0
 MM=0
 MMM=0
 SUM1=0

BLK20061
 BLK20062
 BLK20063
 BLK20064
 BLK20065
 BLK20066
 BLK20067
 BLK20068
 BLK20069
 BLK20070
 BLK20071
 BLK20072
 BLK20073
 BLK20074
 BLK20075
 BLK20076
 BLK20077
 BLK20078
 BLK20079
 BLK20080
 BLK20081
 BLK20082
 BLK20083
 BLK20084
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 BLK20100
 BLK20101
 BLK20102
 BLK20103
 BLK20104
 BLK20105
 BLK20106
 BLK20107
 BLK20108
 BLK20109
 BLK20110
 BLK20111
 BLK20112
 BLK20113
 BLK20114
 BLK20115
 BLK20116
 BLK20117
 BLK20118
 BLK20119
 BLK20120

SUM2=0

NP5=33

T(1)=159.69

T(2)=354.69

T(3)=369.69

T(4)=375.69

T(5)=376.69

T(6)=377.69

T(7)=378.69

T(8)=379.69

T(9)=381.69

T(10)=383.69

T(11)=385.69

T(12)=387.69

T(13)=389.69

T(14)=394.69

T(15)=399.69

T(16)=409.69

T(17)=419.69

T(18)=429.69

T(19)=439.69

T(20)=449.69

T(21)=459.69

T(22)=469.69

T(23)=479.69

T(24)=489.69

T(25)=499.69

T(26)=509.69

T(27)=519.69

T(28)=529.69

T(29)=539.69

T(30)=559.69

T(31)=609.69

T(32)=659.69

T(33)=759.69

TMU(1)=125000000.0

TMU(2)=125000000.0

TMU(3)=240000.0

TMU(4)=63000.0

TMU(5)=25000.0

TMU(6)=11750.0

TMU(7)=6600.0

TMU(8)=3900.0

TMU(9)=1850.0

TMU(10)=1420.0

TMU(11)=1190.0

TMU(12)=1000.0

TMU(13)=870.0

TMU(14)=610.0

TMU(15)=425.0

TMU(16)=245.0

TMU(17)=135.0

TMU(18)=80.0

TMU(19)=51.9

TMU(20)=34.0

TMU(21)=24.5

TMU(22)=16.5

TMU(23)=12.2

TMU(24)=9.3

TMU(25)=7.3

BLK20121

BLK20122

BLK20123

BLK20124

BLK20125

BLK20126

BLK20127

BLK20128

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TMU(26)=5.75
 TMU(27)=4.65
 TMU(28)=3.75
 TMU(29)=3.05
 TMU(30)=2.08
 TMU(31)=1.11
 TMU(32)=0.625
 TMU(33)=0.269
 GO TO (1212,1313,1414,1515,1616),MISSION

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* * * * * BROADSIDE * * * * *

12121 MSTART(1)=1
 MSTART(2)=19
 MSTART(3)=37
 MSTART(4)=47
 PERIOD=2.0419
 P3=8.1676
 NP1=18
 NP2=36
 NP3=46
 NP4=56
 Q(1)=284.19
 Q(2)=275.61
 Q(3)=267.04
 Q(4)=217.70
 Q(5)=142.07
 Q(6)=8.31
 Q(7)=8.06
 Q(8)=31.18
 Q(9)=1.37
 Q(10)=1.37
 Q(11)=31.18
 Q(12)=8.06
 Q(13)=8.31
 Q(14)=142.07
 Q(15)=217.70
 Q(16)=267.04
 Q(17)=275.61
 Q(18)=284.19
 Q(19)=284.19
 Q(20)=275.61
 Q(21)=267.04
 Q(22)=217.70
 Q(23)=142.07
 Q(24)=8.31
 Q(25)=8.06
 Q(26)=31.18
 Q(27)=1.37
 Q(28)=1.37
 Q(29)=31.18
 Q(30)=8.06
 Q(31)=8.31
 Q(32)=142.07
 Q(33)=217.70
 Q(34)=267.04
 Q(35)=275.61
 Q(36)=284.19

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 BLK20232
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 BLK20240

Q(37)=77.48
Q(38)=72.81
Q(39)=59.38
Q(40)=38.72
Q(41)=0.0
Q(42)=0.0
Q(43)=38.72
Q(44)=59.38
Q(45)=72.81
Q(46)=77.48
Q(47)=77.48
Q(48)=72.81
Q(49)=59.38
Q(50)=38.72
Q(51)=0.0
Q(52)=0.0
Q(53)=38.72
Q(54)=59.38
Q(55)=72.81
Q(56)=77.48
TIME(1)=0.0
TIME(2)=0.0567
TIME(3)=0.1135
TIME(4)=0.2268
TIME(5)=0.3403
TIME(6)=0.5105
TIME(7)=0.5388
TIME(8)=0.6401
TIME(9)=0.6411
TIME(10)=1.4020
TIME(11)=1.4030
TIME(12)=1.5031
TIME(13)=1.5314
TIME(14)=1.7016
TIME(15)=1.8151
TIME(16)=1.9284
TIME(17)=1.9853
TIME(18)=2.0419
TIME(19)=0.0
TIME(20)=0.0567
TIME(21)=0.1135
TIME(22)=0.2268
TIME(23)=0.3403
TIME(24)=0.5105
TIME(25)=0.5388
TIME(26)=0.6401
TIME(27)=0.6411
TIME(28)=1.4020
TIME(29)=1.4030
TIME(30)=1.5031
TIME(31)=1.5314
TIME(32)=1.7016
TIME(33)=1.8151
TIME(34)=1.9284
TIME(35)=1.9853
TIME(36)=2.0419
TIME(37)=0.0
TIME(38)=0.1135
TIME(39)=0.2268
TIME(40)=0.3403

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BLK20297
BLK20298
BLK20299
BLK20300

TIME(41)=0.5105
 TIME(42)=1.5314
 TIME(43)=1.7016
 TIME(44)=1.8151
 TIME(45)=1.9284
 TIME(46)=2.0419
 TIME(47)=0.0
 TIME(48)=0.1135
 TIME(49)=0.2268
 TIME(50)=0.3403
 TIME(51)=0.5105
 TIME(52)=1.5314
 TIME(53)=1.7016
 TIME(54)=1.8151
 TIME(55)=1.9284
 TIME(56)=2.0419
 GO TO 1111

C
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 C * * * * * LUNAR DIRECT - NOSE DOWN * * * * *
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1313 MSTART(1)=1
 MSTART(2)=15
 MSTART(3)=29
 MSTART(4)=43
 PERIOD=2.0419
 P3=8.1676
 NP1=14
 NP2=28
 NP3=42
 NP4=56
 Q(1)=100.43
 Q(2)=123.63
 Q(3)=129.74
 Q(4)=131.92
 Q(5)=130.11
 Q(6)=124.30
 Q(7)=108.87
 Q(8)=77.11
 Q(9)=71.18
 Q(10)=0.49
 Q(11)=0.49
 Q(12)=71.25
 Q(13)=91.33
 Q(14)=100.43
 Q(15)=100.43
 Q(16)=123.63
 Q(17)=129.74
 Q(18)=131.92
 Q(19)=130.11
 Q(20)=124.30
 Q(21)=108.87
 Q(22)=77.11
 Q(23)=71.18
 Q(24)=0.49
 Q(25)=0.49
 Q(26)=71.25
 Q(27)=91.33
 Q(28)=100.43

BLK20301
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 BLK20360

Q(29)=100.43
 Q(30)=91.33
 Q(31)=71.25
 Q(32)=0.49
 Q(33)=0.49
 Q(34)=71.18
 Q(35)=77.11
 Q(36)=108.87
 Q(37)=124.30
 Q(38)=130.11
 Q(39)=131.92
 Q(40)=129.74
 Q(41)=123.63
 Q(42)=100.43
 Q(43)=100.43
 Q(44)=91.33
 Q(45)=71.25
 Q(46)=0.49
 Q(47)=0.49
 Q(48)=71.18
 Q(49)=77.11
 Q(50)=108.87
 Q(51)=124.30
 Q(52)=130.11
 Q(53)=131.92
 Q(54)=129.74
 Q(55)=123.63
 Q(56)=100.43
 TIME(1)=0.0
 TIME(2)=0.1135
 TIME(3)=0.1702
 TIME(4)=0.2268
 TIME(5)=0.2837
 TIME(6)=0.3403
 TIME(7)=0.4253
 TIME(8)=0.5388
 TIME(9)=0.6401
 TIME(10)=0.6411
 TIME(11)=1.5598
 TIME(12)=1.8151
 TIME(13)=1.9284
 TIME(14)=2.0419
 TIME(15)=0.0
 TIME(16)=0.1135
 TIME(17)=0.1702
 TIME(18)=0.2268
 TIME(19)=0.2837
 TIME(20)=0.3403
 TIME(21)=0.4253
 TIME(22)=0.5388
 TIME(23)=0.6401
 TIME(24)=0.6411
 TIME(25)=1.5598
 TIME(26)=1.8151
 TIME(27)=1.9284
 TIME(28)=2.0419
 TIME(29)=0.0
 TIME(30)=0.1135
 TIME(31)=0.2268
 TIME(32)=0.4822

BLK20361
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 BLK20419
 BLK20420

TIME(33)=1.4020
 TIME(34)=1.4030
 TIME(35)=1.5031
 TIME(36)=1.6164
 TIME(37)=1.7016
 TIME(38)=1.7583
 TIME(39)=1.8151
 TIME(40)=1.8718
 TIME(41)=1.9284
 TIME(42)=2.0419
 TIME(43)=0.0
 TIME(44)=0.1135
 TIME(45)=0.2260
 TIME(46)=0.4822
 TIME(47)=1.4020
 TIME(48)=1.4030
 TIME(49)=1.5031
 TIME(50)=1.6164
 TIME(51)=1.7016
 TIME(52)=1.7583
 TIME(53)=1.8151
 TIME(54)=1.8718
 TIME(55)=1.9284
 TIME(56)=2.0419
 GO TO 1111

BLK20421
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 BLK20480

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1414 MSTART(1)=1
 MSTART(2)=14
 MSTART(3)=27
 MSTART(4)=40
 PERIOD=1.0
 P3=4.0
 NP1=13
 NP2=26
 NP3=39
 NP4=52
 Q(1)=76.5
 Q(2)=77.6
 Q(3)=87.0
 Q(4)=88.6
 Q(5)=87.0
 Q(6)=77.6
 Q(7)=60.6
 Q(8)=42.5
 Q(9)= 0.0
 Q(10)= 0.0
 Q(11)=42.5
 Q(12)=60.6
 Q(13)=76.5
 Q(14)=76.5
 Q(15)=60.6
 Q(16)=42.5
 Q(17)= 0.0
 Q(18)= 0.0
 Q(19)=42.5
 Q(20)=60.6
 Q(21)=77.6

Q(22)=87.0
 Q(23)=88.6
 Q(24)=87.0
 Q(25)=77.6
 Q(26)=76.5
 Q(27)= 0.0
 Q(28)= 0.0
 Q(29)=42.5
 Q(30)=60.6
 Q(31)=77.6
 Q(32)=87.0
 Q(33)=88.6
 Q(34)=87.0
 Q(35)=77.6
 Q(36)=60.6
 Q(37)=42.5
 Q(38)= 0.0
 Q(39)= 0.0
 Q(40)= 0.0
 Q(41)= 0.0
 Q(42)=42.5
 Q(44)=77.6
 Q(45)=87.0
 Q(46)=88.6
 Q(47)=87.0
 Q(48)=77.6
 Q(49)=60.6
 Q(50)=42.5
 Q(51)= 0.0
 Q(52)= 0.0
 TIME(1)=0.0
 TIME(2)=0.0078
 TIME(3)=0.0578
 TIME(4)=0.0856
 TIME(5)=0.1134
 TIME(6)=0.1638
 TIME(7)= .219
 TIME(8)=0.258
 TIME(9) = .333
 TIME(10)=0.8360
 TIME(11)=0.911
 TIME(12)=0.952
 TIME(13)=1.0
 TIME(14)=0.0
 TIME(15)=0.0478
 TIME(16)=0.0884
 TIME(17)=0.1660
 TIME(18)=0.666
 TIME(19) = .747
 TIME(20)=0.781
 TIME(21)=0.837
 TIME(22)=0.888
 TIME(23)=0.914
 TIME(24)=0.942
 TIME(25)=0.991
 TIME(26)=1.0
 TIME(27)=0.0
 TIME(28) = .333
 TIME(29) = .418
 TIME(30) = .459

BLK20481
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 BLK20540

TIME(31)=0.507
 TIME(32)=0.557
 TIME(33)=0.586
 TIME(34)=0.613
 TIME(35)=0.664
 TIME(36)=0.719
 TIME(37)=0.758
 TIME(38)=0.835
 TIME(39)=1.0
 TIME(40)=0.0
 TIME(41)=0.1642
 TIME(42)=0.242
 TIME(43)=0.281
 TIME(44)=0.337
 TIME(45)=0.387
 TIME(46)=0.415
 TIME(47)=0.442
 TIME(48)=0.493
 TIME(49)=0.548
 TIME(50)=0.587
 TIME(51)=0.664
 TIME(52)=1.0
 GO TO 1111

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1515 MSTART(1)=1
 MSTART(2)=3
 MSTART(3)=5
 MSTART(4)=7
 PERIOD=1.0
 P3=6.
 QREJ1=1000000.
 NP1=2
 NP2=4
 NP3=6
 NP4=8
 Q(1)=0.0
 Q(2)=0.0
 Q(3)=0.0
 Q(4)=0.0
 Q(5)=0.0
 Q(6)=0.0
 Q(7)=0.0
 Q(8)=0.0
 TIME(1)=0.0
 TIME(2)=1.0
 TIME(3)=0.0
 TIME(4)=1.0
 TIME(5)=0.0
 TIME(6)=1.0
 TIME(7)=0.0
 TIME(8)=1.0
 GO TO 1111
 1616 NEXT=1
 LAST=0
 DO 1629 I=1,8
 READ(5,1620) NPTS(I),ALPHA

1620	FORMAT(15,11A6,A1)	BLK20601
	NOPTS=NPTS(1)	BLK20602
	WRITE(6,1621) NPTS(1),ALPHA	BLK20603
1621	FORMAT(1H0/15,11A6,A1/)	BLK20604
	LAST=LAST+NPTS(1)	BLK20605
	GO TO (1622,1622,1622,1622,1625,1626,1630,1631), 1	BLK20606
1622	READ(5,1623) (TIME(11),11=NEXT,LAST),(Q(11),11=NEXT,LAST)	BLK20607
1623	FORMAT(7E10.3)	BLK20608
	WRITE(6,1624) (TIME(11),11=NEXT,LAST),(Q(11),11=NEXT,LAST)	BLK20609
1624	FORMAT(10X1P7G10.4)	BLK20610
	GO TO 1628	BLK20611
1625	READ(5,1623) (TIME(11),11=NEXT,LAST),(WDTTOT(11),11=1,NOPTS)	BLK20612
	WRITE(6,1624) (TIME(11),11=NEXT,LAST),(WDTTOT(11),11=1,NOPTS)	BLK20613
	GO TO 1628	BLK20614
1626	READ(5,1623) (TIME(11),11=NEXT,LAST),(TIN(11),11=1,NOPTS)	BLK20615
	WRITE(6,1624) (TIME(11),11=NEXT,LAST),(TIN(11),11=1,NOPTS)	BLK20616
	DO 1627 11=1,NOPTS	BLK20617
1627	TIN(11)=TIN(11)+459.69	BLK20618
	IF (KODSEC.EQ.0) GO TO 1633	BLK20619
	GO TO 1628	BLK20620
1630	READ(5,1623) (TIME(11),11=NEXT,LAST),(WDOTSC(11),11=1,NOPTS)	BLK20621
	WRITE(6,1624) (TIME(11),11=NEXT,LAST),(WDOTSC(11),11=1,NOPTS)	BLK20622
	GO TO 1628	BLK20623
1631	READ(5,1623) (TIME(11),11=NEXT,LAST),(TINLSC(11),11=1,NOPTS)	BLK20624
	WRITE(6,1624) (TIME(11),11=NEXT,LAST),(TINLSC(11),11=1,NOPTS)	BLK20625
	DO 1632 11=1,NOPTS	BLK20626
1632	TINLSC(11)=TINLSC(11)+459.69	BLK20627
1628	NEXT=NEXT+NPTS(1)	BLK20628
1629	CONTINUE	BLK20629
1633	IF(.NOT.PERIOD.GT.0.) PERIOD=P3	BLK20630
	TINLT=TIN(1)	BLK20631
	WDOTS=WDTTOT(1)	BLK20632
	WDOTT=WDOTS	BLK20633
	MSTART(1)=1	BLK20634
	NP1=NPTS(1)	BLK20635
	MSTART(2)=NP1+1	BLK20636
	NP2=NP1+NPTS(2)	BLK20637
	MSTART(3)=NP2+1	BLK20638
	NP3=NP2+NPTS(3)	BLK20639
	MSTART(4)=NP3+1	BLK20640
	NP4=NP3+NPTS(4)	BLK20641
	MSTART(5)=NP4+1	BLK20642
	MW=1	BLK20643
	NP5B=NPTS(5)	BLK20644
	NP5A=NP4+NP5B	BLK20645
	MSTART(6)=NP5A+1	BLK20646
	MT=1	BLK20647
	NP6B=NPTS(6)	BLK20648
	NP6A=NP5A+NP6B	BLK20649
	IF(KODSEC.EQ.0) GO TO 1112	BLK20650
	TINSEC=TINLSC(1)	BLK20651
	WDTSEC=WDOTSC(1)	BLK20652
	MSTART(7)=NP6A+1	BLK20653
	MWSEC=1	BLK20654
	NP7=NP6A+NPTS(7)	BLK20655
	MSTART(8)=NP7+1	BLK20656
	MTSEC=1	BLK20657
	NP8=NP7+NPTS(7)	BLK20658
	GO TO 1112	BLK20659
1111	TINLT=TINLT+459.69	BLK20660

TINSEC=TINSEC+459.69	BLK20661
TLINP = TINLT	BLK20662
TLINS = TINSEC	BLK20663
1112 DO 605 I=1,20	BLK20664
TFIN(I)=529.69	BLK20665
TTIN(I)=529.69	BLK20666
605 TSIN(I)=529.69	BLK20667
DO 606 I=21,32	BLK20668
TFIN(I)=529.69	BLK20669
606 TTIN(I)=529.69	BLK20670
WDOTT1=WDOTS/2.	BLK20671
WDOTT2=WDOTT1	BLK20672
FLOWPC=1.	BLK20673
WRITE(6,436)	BLK20674
436 FORMAT('1'//16X'***----- - PRIMARY SYSTEM-----**	BLK20675
1***-----REDUNDANT SYSTEM-----** **TOTAL**//	BLK20676
218X'HEAT'5X'PRESSURE'4X'FLOW'5X'OUTLET'4X'INLINE'3X'HEAT'5X'PRESSURE'4X'FLOW'5X'OUTLET'6X'INLINE'6X'HEAT'7X'TIME'3X'REJECTION'4X'DROP'6X'RATE'3X'TEMPERATURE'3X'HEATER'4X'REJECTION'107X'STAGE STAGE'109X'1'5X'2'//)	BLK20677
3RE'4X'FLOW'5X'OUTLET'6X'INLINE'6X'HEAT'7X'TIME'3X'REJECTION'4X'DROP'6X'RATE'3X'TEMPERATURE'3X'HEATER'4X'REJECTION'107X'STAGE STAGE'109X'1'5X'2'//)	BLK20678
40P'6X'RATE'3X'TEMPERATURE HEATER REJECTION'4X'DROP'6X'RATE'3X'TEMPERATURE'3X'HEATER'4X'REJECTION'107X'STAGE STAGE'109X'1'5X'2'//)	BLK20679
TAU = 0.02	BLK20680
SAVE = 0.02	BLK20681
425 IF(TAU-PERIOD) 431,431,426	BLK20682
426 TAU=TAU-PERIOD	BLK20683
GO TO (427,428,429,430,6008),MSSION	BLK20684
427 MSTART(1)=1	BLK20685
MSTART(2)=19	BLK20686
MSTART(3)=37	BLK20687
MSTART(4)=47	BLK20688
GO TO 431	BLK20689
428 MSTART(1)=1	BLK20690
MSTART(2)=15	BLK20691
MSTART(3)=29	BLK20692
MSTART(4)=43	BLK20693
GO TO 431	BLK20694
429 MSTART(1)=1	BLK20695
MSTART(2)=14	BLK20696
MSTART(3)=27	BLK20697
MSTART(4)=40	BLK20698
GO TO 431	BLK20699
430 MSTART(1)=1	BLK20700
MSTART(2)=3	BLK20701
MSTART(3)=5	BLK20702
MSTART(4)=7	BLK20703
GO TO 431	BLK20704
MSTART(1)=1	BLK20705
MSTART(2)=NP1+1	BLK20706
MSTART(3)=NP2+1	BLK20707
MSTART(4)=NP3+1	BLK20708
MSTART(5)=NP4+1	BLK20709
MSTART(6)=NP5A+1	BLK20710
IF(KODSEC.EQ.0) GO TO 431	BLK20711
MSTART(7)=NP6A+1	BLK20712
MSTART(8)=NP7+1	BLK20713
431 DO 35 K=1,4	BLK20714
GO TO (432,433,434,435),K	BLK20715
432 KK=MSTART(1)	BLK20716
JJ=NP1	BLK20717
GO TO 32	BLK20718
433 KK=MSTART(2)	BLK20719
	BLK20720

JJ=NP2	BLK2072.
GO TO 32	BLK20722
434 KK=MSTART(3)	BLK20723
JJ=1 3	BLK20724
GO TO 32	BLK20725
435 KK=MSTART(4)	BLK20726
JJ=NP4	BLK20727
32 DO 40 I=KK,JJ	BLK20728
J=I	BLK20729
IF(TAU-TIME(I)) 36,37,40	BLK20730
40 CONTINUE	BLK20731
MERR=K	BLK20732
GO TO 2000	BLK20733
36 TEMP=TAU-TIME(J-1)	BLK20734
TINT=TIME(J)-TIME(J-1)	BLK20735
TEMPQ=Q(J)-Q(J-1)	BLK20736
QABS(K)=Q(J-1)+TEMP/TINT*TEMPQ	BLK20737
MSTART(K)=J-1	BLK20738
GO TO 35	BLK20739
37 QABS(K)=Q(J)	BLK20740
MSTART(K)=J	BLK20741
35 CONTINUE	BLK20742
IF(MSSION .EQ. 5) GO TO 5012	BLK20743
IF(KODSEC .EQ. 0) GO TO 2507	BLK20744
GO TO 5013	BLK20745
5012 KK = MSTART(5)	BLK20746
DO 1650 I=KK, NP5A	BLK20747
J=I	BLK20748
IF(TAU-TIME(I)) 1651,1652,1660	BLK20749
1660 MW=MW+1	BLK20750
1650 CONTINUE	BLK20751
MERR=5	BLK20752
GO TO 2000	BLK20753
1651 TEMP=TAU-TIME(J-1)	BLK20754
TINT=TIME(J)-TIME(J-1)	BLK20755
TEMPW=WDTTOT(MW)-WDTTOT(MW-1)	BLK20756
WDOTS=WDTTOT(MW-1)+TEMP/TINT*TEMPW	BLK20757
MW=MW-1	BLK20758
MSTART(5)=J-1	BLK20759
GO TO 1653	BLK20760
1652 WDOTS=WDTTOT(MW)	BLK20761
MSTART(5)=J	BLK20762
1653 KK=MSTART(6)	BLK20763
DO 1654 I=KK, NP6A	BLK20764
J=I	BLK20765
IF(TAU-TIME(I)) 1655,1656,1664	BLK20766
1664 MT=MT+1	BLK20767
1654 CONTINUE	BLK20768
MERR=6	BLK20769
2000 WRITE(6,2001) MERR,TAU	BLK20770
2001 FORMAT(1H010X24HINTERPOLATION IMPOSSIBLE/15X5HMERR=15/	BLK20771
110X5HTIME=F10.2///10X40HEXECUTION TERMINATED BY PROGRAMMED HALT.)	BLK20772
CALL EXIT	BLK20773
1655 TEMP=TAU-TIME(J-1)	BLK20774
TINT=TIME(J)-TIME(J-1)	BLK20775
TEMPT=TIN(MT)-TIN(MT-1)	BLK20776
TINLT=TIN(MT-1)+TEMP/TINT*TEMPT	BLK20777
MT=MT-1	BLK20778
MSTART(6)=J-1	BLK20779
GO TO 38	BLK20780

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1656 TINLT=TIN(MT)
      MSTART(6)=J
38  TLINP = TINLT
      IF(KODSEC .EQ. 0) GO TO 2507
      KK=MSTART(7)
      DO 2500 I=KK,NP7
      J=1
      IF(TAU-TIME(I)) 2501,2502,
      MWSEC=MWSEC+1
2500 CONTINUE
      MFRR=7
      GO TO 2000
2501 TEMP=TAU-TIME(J-1)
      TINT=TIME(J)-TIME(J-1)
      TEMPWS=WDOTSC(MWSEC)-WDOTSC(MWSEC-1)
      WDTSEC=WDOTSC(MWSEC-1)+TEMP/TINT*TEMPWS
      MWSEC = MWSEC - 1
      MSTART(7)=J-1
      GO TO 2503
2502 WDTSEC=WDOTSC(MWSEC)
      MSTART(7)=J
2503 KK=MSTART(8)
      DO 2504 I=KK,NP8
      J=1
      IF(TAU-TIME(I)) 2505,2506,
      MTSEC=MTSEC+1
2504 CONTINUE
      MERR=8
      GO TO 2000
2505 TEMP=TAU-TIME(J-1)
      TINT=TIME(J)-TIME(J-1)
      TEMPTS=TINLSC(MTSEC)-TINLSC(MTSEC-1)
      TINSEC=TINLSC(MTSEC-1)+TEMP/TINT*TEMPTS
      MTSEC=MTSEC-1
      MSTART(8)=J-1
      GO TO 5015
2506 TINSEC=TINLSC(MTSEC)
      MSTART(8)=J
5015 TLINS = TINSEC
5013 IF(M .EQ. 1) GO TO 5002
      IF(TFIN(32) - 504.69) 5000,5000,2507
5002 IF(TFIN(32).LT.506.69) GO TO 5003
      TINSEC=TLINS
      M=0
      MM=0
      GO TO 2507
5000 M=1
5003 IF(MM.EQ.1) GO TO 5004
      IF(TFIN(32).LT.503.69) GO TO 5007
5006 TINSEC=TLINS+2130./WDTSEC
      SUM2=SUM2+30.7
      GO TO 2507
5004 IF(TFIN(32)-505.69) 5007,5007,5006
5007 TINSEC=TLINS+4260./WDTSEC
      SUM2=SUM2+61.4
2507 IF(MMM .EQ. 1) GO TO 5008
      IF(TOUTP-444.69) 5009,5009,5010
5008 IF(TOUTP.LT.449.69) GO TO 5011
      TINLT=TLINP
      MMM=0

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BLK20781
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BLK20839
BLK20840

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GO TO 5010
5009 MMM=1
5011 TINLT=TLINP+2130./WDOT
SUM1=SUM1+30.7
5010 DO 701 I=1,32
NFCODE(I)=0
701 NTCODE(I)=0
DO 94 I=1,20
94 NSCODE(I)=0
DO 50 I=1,10
50 CALL VPOL(TFIN(I),BMU(I),NP5)
CALL VPOL(TINLT,BMU(33),NP5)
DO 51 I=1,5
51 AK(I)=E(I)*BMU(33)+.000201*BMU(2*I-1)+F(I) *BMU(2*I)
DO 636 I=1,20
636 CALL VPOL(TFIN(I),BMU(I),NP5)
DO 637 I=6,10
637 AK(I)=E(I-5)*BMU(33)+.000201*BMU(2*I-1)+F(I-5) *BMU(2*I)
WDOT(1)=WDOTT1/(AK(1)/AK(2)+AK(1)/AK(3)+AK(1)/AK(4)+AK(1)/AK(5)+
11.)
DO 52 I=2,5
52 WDOT(I)=WDOT(1)*AK(I)/AK(1)
WDOT(6)=WDOTT2/(AK(6)/AK(7)+AK(6)/AK(8)+AK(6)/AK(9)+AK(6)/AK(10)+
11.)
DO 53 I=7,10
53 WDOT(I)=WDOT(6)*AK(I)/AK(1)
DO 54 I=1,10
54 DPT(I)=AK(I)*WDOT(I)
CALL VPOL(TFA21,BMU(34),NP5)
CALL VPOL(TFA23,BMU(35),NP5)
CALL VPOL(TFIN(21),BMU(21),NP5)
CALL VPOL(TFIN(22),BMU(22),NP5)
CALL VPOL(TFIN(23),BMU(23),NP5)
CALL VPOL(TFIN(24),BMU(24),NP5)
AKIL = .002675*BMU(33)
AKIS=.001843*BMU(33)
AKS1=.0000350*BMU(34)
AKS2=.0000350*BMU(35)
AKT21=.000201*BMU(21)
AKT22=.000201*BMU(22)
AKT23=.000201*BMU(23)
AKT24=.000201*BMU(24)
AK(11)=BMU(22)*.00017987
AK(12)=BMU(24)*.00027965
DPTS1=WDOTT1*(AKT21+AKT22+AKS1+AK(11)+AKIS)+DPT(1)
DPTS2=WDOTT2*(AKT23+AKT24+AKS2+AK(12)+AKIL)+DPT(6)
IF (KODSEC.EQ.0) GO TO 1634
DPKAPS = 0.
DO 2026 I=25,32
CALL VPOL(TFIN(I),BMU(I),NP5)
DPKAP = .000201*BMU(I)
DPKAPS = DPKAPS + DPKAP
2026 CONTINUE
CALL VPOL (TINSEC,BMU(37),NP5)
DPTOT = (DPKAPS+.0000254*BMU(26)+.001231*BMU(28)+.00002535*BMU(30)
1+.00003863*BMU(32)+.003889*BMU(37))*WDTSEC
1634 DO 1 I=1,32
TPPF(I)=TFIN(I)
1 TPPT(I)=TTIN(I)
DO 91 I=1,20

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BLK2084
BLK20842
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BLK20899
BLK20900

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91   TPPS(I)=TSIN(I)
    DO 305 ITER=1,500
      LTER=ITER
      MCODE=1
      DO 821 I=1,32
        TF(I)=TPPF(I)
821  TT(I)=TPPT(I)
      DO 93 I=1,20
        93  TS(I)=TPPS(I)
      DO 2 I=1,32
        TPT(I)=TT(I)
        2   TPF(I)=TF(I)
      DO 92 I=1,20
        92  TPS(I)=TS(I)

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C   FLUID LUMP TEMPERATURE EQUATIONS
C
      DO 101 I=1,9,2
        J=(I+1)/2
        IF (NFCODE(I).NE.0) GO TO 101
        TF(I) = (TFIN(I)+AA1*WDOT(J)*TINLT+AA3*TT(I))
        1 / (1.+AA3+AA1*WDOT(J))
        TPPF(I) = TF(I)
101  CONTINUE
      DO 102 I=11,19,2
        J=(I+1)/2
        IF (NFCODE(I).NE.0) GO TO 102
        TF(I) = (TFIN(I)+AA2*WDOT(J)*TINLT+AA3*TT(I))
        1 / (1.+AA3+AA2*WDOT(J))
        TPPF(I) = TF(I)
102  CONTINUE
      DO 103 I=2,10,2
        J=I/2
        IF (NFCODE(I).NE.0) GO TO 103
        TF(I) = (TFIN(I)+AA1*WDOT(J)*TF(I-1)+A3A*TT(I))
        1 / (1.+A3A+AA1*WDOT(J))
        TPPF(I) = TF(I)
103  CONTINUE
      DO 104 I=12,20,2
        J=I/2
        IF (NFCODE(I).NE.0) GO TO 104
        TF(I) = (TFIN(I)+AA2*WDOT(J)*TF(I-1)+A3A*TT(I))
        1 / (1.+A3A+AA2*WDOT(J))
        TPPF(I) = TF(I)
104  CONTINUE
      IF (NFCODE(21).NE.0) GO TO 2020
      DO 2021 I=1,5
2021  H(I) = -83.39+.232*TF(2*I)+.000486*TF(2*I)**2
        H21=WDOT(1)*H(1)+WDOT(2)*H(2)+WDOT(3)*H(3)+WDOT(4)*H(4)+
        1 WDOT(5)*H(5)
        TFA21 = 247.43+1.879*H21/WDOTT1-.00155*H21*H21/WDOTT1/WDOTT1
        TF(21) = (TFIN(21)+AA1*WDOTT1*TFA21+A3A*TT(21))/
        1 (1. + A3A + AA1*WDOTT1)
        TPPF(21) = TF(21)
2020  IF (NFCODE(22).NE.0) GO TO 2002
        TF(22) = (TFIN(22)+AA1*WDOTT1*TF(21)+A3A*TT(22))/
        1 (1. + A3A + AA1*WDOTT1)
        TPPF(22) = TF(22)
2002  IF (NFCODE(23).NE.0) GO TO 2003
        DO 2004 I=6,10

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BLK20901
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BLK20960

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2004 H(1) = -B3.39+.232*TF(2*1)+.000486*TF(2*1)**2
      H23 = WDOT(6)*H(6)+WDOT(7)*H(7)+WDOT(8)*H(8)+WDOT(9)*H(9)+
      1 WDOT(10)*H(10)
      TFA23 = 247.43+1.879*H23/WDOTT2- .00155*H23*H23/WDOTT2/WDOTT2
      TF(23) = (TFIN(23)+AA2*WDOTT2*TFA23+A3A*TT(23))/
      1 (1. + A3A + AA2*WDOTT2)
      TPPF(23) = TF(23)
2003 IF (NFCODE(24).NE.0) GO TO 2005
      TF(24) = (TFIN(24)+AA2*WDOTT2*TF(23)+A3A*TT(24))/
      1 (1. + A3A + AA2*WDOTT2)
      TPPF(24) = TF(24)
2005 IF (NFCODE(25).NE.0) GO TO 2006
      TF(25) = (TFIN(25)+B1*WDTSEC*TTINSEC+B3*TT(25))/(1.+B3+B1*WDTSEC)
      TPPF(25) = TF(25)
2006 DO 2007 I=26,32
      IF (NFCODE(I).NE.0) GO TO 2007
      TF(I) = (TFIN(I)+B1*WDTSEC*TF(I-1)+B3*TT(I))/(1.+B1*WDTSEC+B3)
      TPPF(I) = TF(I)
2007 CONTINUE

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TUBE LUMP TEMPERATURE EQUATIONS

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109 IF(NTCODE( 1))111,112,111
112 TT(1)=(TTIN(1)+A14*TF(1)+A15*TS(1)+A2*TT(26)+A7*QABS(1)-
      1 A9*TT(1)**4)/(1.+A14+A15+A2)
      TPPT( 1)=TT( 1)
111 IF(NTCODE( 2))113,114,113
114 TT(2)=(TTIN(2)+A16*TF(2)+A15*TS(2)+A2*TT(25) +A7*QABS(2)-
      1 A9*TT(2)**4)/(1.+A16+A15+A2)
      TPPT( 2)=TT( 2)
113 IF(NTCODE( 3))115,116,115
116 TT(3)=(TTIN(3)+AA5 *TF(3)+A5 *TS(3)+A5 *TS(1)+A8 *QABS(1)-
      1A10 *TT(3)**4)/(1.+AA5+2.*A5)
      TPPT( 3)=TT( 3)
115 IF(NTCODE( 4))117,118,117
118 TT(4)=(TTIN(4)+A5A *TF(4)+A5 *TS(2)+A5 *TS(4)+ A8 *QABS(2)-
      1A10 *TT(4)**4)/(1.+A5A+2.*A5)
      TPPT( 4)=TT( 4)
117 IF(NTCODE( 5))119,120,119
120 TT(5)=(TTIN(5)+AA5 *TF(5)+A5 *TS(3)+A5 *TS(5)+A8 *QABS(1)-
      1A10 *TT(5)**4)/(1.+AA5+2.*A5)
      TPPT( 5)=TT( 5)
119 IF(NTCODE( 6))121,122,121
122 TT(6)=(TTIN(6)+A5A *TF(6)+A5 *TS(4)+A5 *TS(6)+A8 *QABS(2)-
      1 A10 *TT(6)**4)/(1.+A5A+2.*A5)
      TPPT( 6)=TT( 6)
121 IF(NTCODE( 7))123,124,123
124 TT(7)=(TTIN(7)+AA5 *TF(7)+A5 *TS(5)+A5 *TS(7)+A8 *QABS(1)-
      1A10 *TT(7)**4)/(1.+AA5+2.*A5)
      TPPT( 7)=TT( 7)
123 IF(NTCODE( 8))125,126,125
126 TT(8)=(TTIN(8)+A5A *TF(8)+A5 *TS(6)+A5 *TS(8)+A8 *QABS(2)-
      1A10 *TT(8)**4)/(1.+A5A+2.*A5)
      TPPT( 8)=TT( 8)
125 IF(NTCODE( 9))127,128,127
128 TT(9)=(TTIN(9)+A14 *TF(9)+A15 *TS(7)+A7 *QABS(1)-A9*
      1TT(9)**4)/(1.+A14+A15)
      TPPT( 9)=TT( 9)
127 IF(NTCODE(10))129,130,129
130 TT(10)=(TTIN(10)+A16 *TF(10)+A15 *TS(8)+A7 *QABS(2)-

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1A9	$*TT(10)**4)/(1.+A16+A15)$	BLK21021
	TPPT(10)=TT(10)	BLK21022
129	IF(NTCODE(11))131,132,131	BLK21023
132	TT(11)=(TTIN(11)+A14*TF(11)+A15*TS(9)+A2*TT(30)+A7*QABS(4)-	BLK21024
	1 A9*TT(11)**4)/(1.+A14+A15+A2)	BLK21025
	TPPT(11)=TT(11)	BLK21026
131	IF(NTCODE(12))133,134,133	BLK21027
134	TT(12)=(TTIN(12)+A16*TF(12)+A15*TS(10)+A2*TT(29)+A7*QABS(3)-	BLK21028
	1 A9*TT(12)**4)/(1.+A16+A15+A2)	BLK21029
	TPPT(12)=TT(12)	BLK21030
133	IF(NTCODE(13))135,136,135	BLK21031
136	TT(13)=(TTIN(13)+AA6*TF(13)+A5*TS(9)+A5*TS(11)+A8*QABS(4)-	BLK21032
	1-A10*TT(13)**4)/(1.+AA6+2.*A5)	BLK21033
	TPPT(13)=TT(13)	BLK21034
135	IF(NTCODE(14))137,138,137	BLK21035
138	TT(14)=(TTIN(14)+A5A*TF(14)+A5*TS(10)+A5*TS(12)+A8*QABS(3)-A10	BLK21036
	*TT(14)**4)/(1.+A5A+2.*A5)	BLK21037
	TPPT(14)=TT(14)	BLK21038
137	IF(NTCODE(15))139,140,139	BLK21039
140	TT(15)=(TTIN(15)+AA6*TF(15)+A5*TS(11)+A5*TS(13)+A8*QABS(4)-A10	BLK21040
	*TT(15)**4)/(1.+AA6+2.*A5)	BLK21041
	TPPT(15)=TT(15)	BLK21042
139	IF(NTCODE(16))141,142,141	BLK21043
142	TT(16)=(TTIN(16)+A5A*TF(16)+A5*TS(12)+A5*TS(14)+A8*QABS(3)-A10	BLK21044
	*TT(16)**4)/(1.+A5A+2.*A5)	BLK21045
	TPPT(16)=TT(16)	BLK21046
141	IF(NTCODE(17))143,144,143	BLK21047
144	TT(17)=(TTIN(17)+AA6*TF(17)+A5*TS(13)+A5*TS(15)+A8*QABS(4)-A10	BLK21048
	*TT(17)**4)/(1.+AA6+2.*A5)	BLK21049
	TPPT(17)=TT(17)	BLK21050
143	IF(NTCODE(18))145,146,145	BLK21051
146	TT(18)=(TTIN(18)+A5A*TF(18)+A5*TS(14)+A5*TS(16)+A8*QABS(3)-A10	BLK21052
	*TT(18)**4)/(1.+A5A+2.*A5)	BLK21053
	TPPT(18)=TT(18)	BLK21054
145	IF(NTCODE(19))147,148,147	BLK21055
148	TT(19)=(TTIN(19)+A14*TF(19)+A15*TS(15)+A7*QABS(4)-	BLK21056
	1A9*TT(19)**4)/(1.+A14+A15)	BLK21057
	TPPT(19)=TT(19)	BLK21058
147	IF(NTCODE(20))149,150,149	BLK21059
150	TT(20)=(TTIN(20)+A16*TF(20)+A15*TS(16)+A7*QABS(3)-	BLK21060
	1A9*TT(20)**4)/(1.+A16+A15)	BLK21061
	TPPT(20)=TT(20)	BLK21062
149	IF(NTCODE(21).NE.0) GO TO 2008	BLK21063
	TT(21)=(TTIN(21)+A5A*TF(21)+A20*TS(17)+A2*TT(28)+A8*QABS(2)-	BLK21064
	1 A10*TT(21)**4)/(1.+A5A+A20+A2)	BLK21065
	TPPT(21)=TT(21)	BLK21066
2008	IF(NTCODE(22).NE.0) GO TO 2009	BLK21067
	TT(22)=(TTIN(22)+A5A*TF(22)+A20*TS(18)+A2*TT(27)+A8*QABS(1)-	BLK21068
	1 A10*TT(22)**4)/(1.+A5A+A20+A2)	BLK21069
	TPPT(22)=TT(22)	BLK21070
2009	IF(NTCODE(23).NE.0) GO TO 2010	BLK21071
	TT(23)=(TTIN(23)+A5A*TF(23)+A20*TS(19)+A2*TT(32)+A8*QABS(3)-	BLK21072
	1 A10*TT(23)**4)/(1.+A5A+A20+A2)	BLK21073
	TPPT(23)=TT(23)	BLK21074
2010	IF(NTCODE(24).NE.0) GO TO 2011	BLK21075
	TT(24)=(TTIN(24)+A5A*TF(24)+A20*TS(20)+A2*TT(31)+A8*QABS(4)-	BLK21076
	1 A10*TT(24)**4)/(1.+A5A+A20+A2)	BLK21077
	TPPT(24)=TT(24)	BLK21078
2011	IF(NTCODE(25).NE.0) GO TO 2012	BLK21079
	TT(25)=(TTIN(25)+B4*TF(25)+A2*TT(21))/(1.+B4+A2)	BLK21080

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      TPPT(25) = TT(25)
2012 IF (NTCODE(26).NE.0) GO TO 2013
      TT(26) = (TTIN(26)+B4*TF(26)+A2*TT(1))/(1.+B4+A2)
      TPPT(26) = TT(26)
2013 IF (NTCODE(27).NE.0) GO TO 2014
      TT(27) = (TTIN(27)+B4*TF(27)+A2*TT(22)+A20*TS(18)+A8*QABS(1)-
1 A10*TT(27)**4)/(1.+B4+A2+A20)
      TPPT(27) = TT(27)
2014 IF (NTCODE(28).NE.0) GO TO 2015
      TT(28) = (TTIN(28)+A5A*TT(28)+A2*TT(21)+A20*TS(17)+A8*QABS(2)-
1 A10*TT(28)**4)/(1.+A5A+A2+A20)
      TPPT(28) = TT(28)
2015 IF (NTCODE(29).NE.0) GO TO 2016
      TT(29) = (TTIN(29)+B4*TF(29)+A2*TT(12))/(1.+B4+A2)
      TPPT(29) = TT(29)
2016 IF (NTCODE(30).NE.0) GO TO 2017
      TT(30) = (TTIN(30)+B4*TF(30)+A2*TT(11))/(1.+B4+A2)
      TPPT(30) = TT(30)
2017 IF (NTCODE(31).NE.0) GO TO 2018
      TT(31) = (TTIN(31)+B4*TF(31)+A2*TT(24)+A20*TS(20)+A8*QABS(4)-
1 A10*TT(31)**4)/(1.+B4+A2+A20)
      TPPT(31) = TT(31)
2018 IF (NTCODE(32).NE.0) GO TO 2019
      TT(32) = (TTIN(32)+B4*TF(32)+A2*TT(23)+A20*TS(19)+A8*QABS(3)-
1 A10*TT(32)**4)/(1.+B4+A2+A20)
      TPPT(32) = TT(32)

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STRUCTURAL LUMP TEMPERATURE EQUATIONS

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2019 DO 155 I=3,7,2
      IF(NSCODE(I)) 155,156,155
156 TS(I)=(TSIN(I)+A11 *TT(I)+A11 *TT(I+2)+A12 *QABS(1)-A13*
1 TS(I)**4)/(1.+2.*A11)
      TPPS(I)=TS(I)
155 CONTINUE
      DO 255 I=11,15,2
      IF(NSCODE(I)) 255,256,255
256 TS(I)=(TSIN(I)+A11 *TT(I+2)+A11 *TT(I+4)+A12 *QABS(4)-
1 A13 *TS(I)**4)/(1.+2.*A11)
      TPPS(I)=TS(I)
255 CONTINUE
      DO 157 I=4,8,2
      IF(NSCODE(I)) 157,158,157
158 TS(I)=(TSIN(I)+A11 *TT(I)+A11 *TT(I+2)+A12 *QABS(2)-A13*
1 TS(I)**4)/(1.+2.*A11)
      TPPS(I)=TS(I)
157 CONTINUE
      DO 257 I=12,16,2
      IF(NSCODE(I)) 257,258,257
258 TS(I)=(TSIN(I)+A11 *TT(I+2)+A11 *TT(I+4)+A12 *QABS(3)-
1 A13 *TS(I)**4)/(1.+2.*A11)
      TPPS(I)=TS(I)
257 CONTINUE
      DO 356 I=1,2
      IF (NSCODE(I)) 356,355,356
355 TS(I)=(TSIN(I)+A17*TT(I)+A17*TT(I+2)+A18*QABS(1)-A19*TS(I)**4)
1/(1.+2.*A17)
      TPPS(I)=TS(I)
356 CONTINUE
      IF (NSCODE(9).NE.0) GO TO 357

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BLK21109
BLK21110
BLK21111
BLK21112
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BLK21140

TS(9)=(TSIN(9)+A17*TT(11)+A17*TT(13)+A18*QABS(4)-A19*TS(9)**4)	BLK21141
1/(1.+2.*A17)	BLK21142
TPPS(9)=TS(9)	BLK21143
357 IF (NSCODE(10).NE.0) GO TO 358	BLK21144
TS(10)=(TSIN(10)+A17*TT(12)+A17*TT(14)+A18*QABS(3)-A19*TS(10)**4)	BLK21145
1/(1.+2.*A17)	BLK21146
TPPS(10)=TS(10)	BLK21147
358 CONTINUE	BLK21148
IF (NSCODE(17).NE.0) GO TO 2022	BLK21149
TS(17) = (TSIN(17)+A23*TT(21)+A23*TT(24)+A21*QABS(2)-	BLK21150
1 A22*TS(17)**4)/(1.+2.*A23)	BLK21151
TPPS(17) = TS(17)	BLK21152
2022 IF (NSCODE(18).NE.0) GO TO 2023	BLK21153
TS(18) = (TSIN(18)+A23*TT(22)+A23*TT(27)+A21*QABS(1)-	BLK21154
1 A22*TS(18)**4)/(1.+2.*A23)	BLK21155
Q(10)=0.49	BLK21156
TPPS(18) = TS(18)	BLK21157
2023 IF (NSCODE(19).NE.0) GO TO 2024	BLK21158
TS(19) = (TSIN(19)+A23*TT(23)+A23*TT(32)+A21*QABS(3)-	BLK21159
1 A22*TS(19)**4)/(1.+2.*A23)	BLK21160
TPPS(19) = TS(19)	BLK21161
2024 IF (NSCODE(20).NE.0) GO TO 2025	BLK21162
TS(20) = (TSIN(20)+A23*TT(24)+A23*TT(31)+A21*QABS(4)-	BLK21163
1 A22*TS(20)**4)/(1.+2.*A23)	BLK21164
TPPS(20) = TS(20)	BLK21165
2025 CONTINUE	BLK21166
DO 820 I=1,32	BLK21167
TPPF(I)=TPF(I)+1.3*(TPPF(I)-TPF(I))	BLK21168
820 TPPT(I)=TPT(I)+1.3*(TPPT(I)-TPT(I))	BLK21169
DO 95 I=1,20	BLK21170
95 TPPS(I)=TPS(I)+1.3*(TPPS(I)-TPS(I))	BLK21171
DO 14 I=1,32	BLK21172
IF (NFCODE(I).NE.0) GO TO 1300	BLK21173
IF (ABS(TPPF(I)-TPF(I))-0.1) 20,20,21	BLK21174
20 NFCODE(I)=ITER	BLK21175
GO TO 22	BLK21176
21 NFCODE(I)=0	BLK21177
MCODE=0	BLK21178
1300 IF (NTCODE(I).NE.0) GO TO 14	BLK21179
22 IF (ABS(TPPT(I)-TPT(I))-0.1) 23,23,24	BLK21180
23 NTCODE(I)=ITER	BLK21181
GO TO 14	BLK21182
24 NTCODE(I)=0	BLK21183
MCODE=0	BLK21184
14 CONTINUE	BLK21185
DO 96 I=1,20	BLK21186
1301 IF (NSCODE(I).NE.0) GO TO 96	BLK21187
IF (ABS(TPPS(I)-TPS(I))-0.1) 26,26,27	BLK21188
26 NSCODE(I)=ITER	BLK21189
GO TO 96	BLK21190
27 NSCODE(I)=0	BLK21191
MCODE=0	BLK21192
96 CONTINUE	BLK21193
IF (MCODE.EQ.0) GO TO 305	BLK21194
DO 306 I=1,20	BLK21195
IF (NFCODE(I).NE.ITER) GO TO 308	BLK21196
IF (NTCODE(I).NE.ITER) GO TO 308	BLK21197
IF (NSCODE(I).NE.ITER) GO TO 308	BLK21198
306 CONTINUE	BLK21199
DO 307 I=1,32	BLK21200

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      IF (NFCODE(1).NE.ITER) GO TO 308
      IF (NFCODE(1).NE.ITER) GO TO 308
307  CONTINUE
      GO TO 28
308  DO 309 I=1,20
      NFCODE(I)=0
      NTCODE(I)=0
      NSCODE(I)=0
309  CONTINUE
      DO 310 I=21,32
      NFCODE(I)=0
      NTCODE(I)=0
310  CONTINUE
305  CONTINUE
      WRITE(6,503)
      GO TO 399
28   IF (KODSEC.EQ.0) GO TO 1635
      HIN=-83.39+.232*TLINS+.000486*TLINS**2
      HOUT=-83.39+.232*TF(32)+.000486*TF(32)**2
      QRJTSC = WDTSEC * (HIN - HOUT )
1635 H22 = -83.39+.232*TF(22)+.000486*TF(22)**2
      H24 = -83.39+.232*TF(24)+.000486*TF(24)**2
      HINLT=-83.39+.232*TLINP+.000486*TLINP**2
      HPRI = (WDOTT1*H22 + WDOTT2*H24)/WDOTT
      QRJT = WDOTT * (HINLT - HPRI)
      TOUT = 247.43+1.879*HPRI-.00155*HPRI*HPRI
      HMIX = (WDOTT*HPRI + (WDOTS-WDOTT)*HINLT)/WDOTS
      TOUTP=TOUT
      TMIX = 247.43+1.879*HMIX-.00155*HMIX*HMIX
      CALL VPOL (TOUT,BMU(36),NP5)
      RATE1=WDOTT1
      RATE2=WDOTT2
399  CONTINUE
      DO 66 I=1,32
      TFIN(I)=TPPF(I)
66   TTIN(I)=TPPT(I)
      DO 98 I=1,20
98   TSIN(I)=TPPS(I)
      TOUTSC = TFIN(32)
      SLTEMP = TMIX
      DELTAT=ABS(SLTEMP-SETPT)-DBAND
      ARG2=SLTEMP-SETPT
      IF(DELAT)452,452,453
453  DELTAT=SIGN(DELAT,ARG2)
      DELTAP=RTFCTR*DELTAT
      IF(ABS(DELTAP)-RLIMIT)454,454,455
455  DELTAP=SIGN(RLIMIT,ARG2)
454  FLOWPC=FLOWPC+DELTAP*3600.**.02
      IF(FLOWPC-FLOWMX)456,452,457
457  FLOWPC=FLOWMX
      GO TO 452
456  IF(FLOWPC-FLOWMN)458,452,452
458  FLOWPC=FLOWMN
452  WDOTT=WDOTS*FLOWPC
      IF(KODE) 732,,731
      DTEMP=TF(22)-TF(24)
      DX= (POSIN-XX1+VLVGAN*DTEMP)
      XX1=XX1+DX
      IF(XX1-POSMIN) 81,82,82
81  XX1=POSMIN

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BLK21260

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GO TO 84	BLK21261
82 IF (XX1-POSMAX) 84,84,83	BLK21262
83 XX1=POSMAX	BLK21263
84 XX2=FULOPN-XX1	BLK21264
BB=DPTS2/WDOTT2+DPTS1/WDOTT1+PPARA*(WDOTT/(GFACT*XX2**2))	BLK21265
CC=DPTS1/WDOTT1*WDOTT+(WDOTT/XX2)**2/GFACT	BLK21266
AA=(1./XX2**2-1./XX1**2)/GFACT	BLK21267
IF (ABS(XX1-XX2)-VTOL) 85,85,86	BLK21268
85 WDOTT2=CC/BB	BLK21269
GO TO 87	BLK21270
86 WDOTT2=(BB-SQRT(BB**2-4.*AA*CC))/2./AA	BLK21271
87 WDOTT1=WDOTT-WDOTT2	BLK21272
FLOW=WDOTT1	BLK21273
GO TO 2510	BLK21274
731 IF (KODE.EQ.2) GO TO 733	BLK21275
WDOTT1=WDOTT	BLK21276
WDOTT2=0.	BLK21277
FLOW=WDOTT1	BLK21278
GO TO 2510	BLK21279
733 WDOTT1=0.	BLK21280
WDOTT2=0.	BLK21281
FLOW=0.	BLK21282
GO TO 2510	BLK21283
732 WDOTT1=0.	BLK21284
WDOTT2=WDOTT	BLK21285
FLOW=WDOTT2	BLK21286
2510 IF (DPTS1.LT.DPTS2) DPTS1=DPTS2	BLK21287
DPPRNT=DPTS1+(.005912*BMU(33)+.00079542*BMU(36))*WDOTT	BLK21288
TOTREJ=QRJT+QRJTSC	BLK21289
IF (MSSION.EQ.4.OR.(MISSION.LT.4.AND.SAVE+PERIOD.LT.P3)) GO TO 940	BLK21290
914 COUNT=COUNT+1.	BLK21291
SDP=SDP+DPPRNT	BLK21292
STOUT=STOUT+TOUT	BLK21293
SQREJ=SQREJ+TOTREJ	BLK21294
IF (TOTREJ.GT.TMAX) TMAX=TOTREJ	BLK21295
IF (TOTREJ.LT.TMIN) TMIN=TOTREJ	BLK21296
IF (TOUT.GT.TMX) TMX=TOUT	BLK21297
IF (TOUT.LT.TMN) TMN=TOUT	BLK21298
TEMP=1.-FLOWPC	BLK21299
IF (TEMP.GT.PCMAX) PCMAX=TEMP	BLK21300
IF (TEMP.LT.PCMIN) PCMIN=TEMP	BLK21301
IF (DPPRNT.GT.DPMAX) DPMAX=DPPRNT	BLK21302
IF (DPPRNT.LT.DPMIN) DPMIN=DPPRNT	BLK21303
IF (KODSEC.EQ.0) GO TO 940	BLK21304
SDP2=SDP2+DPTOT	BLK21305
STOUT2=STOUT2+TOUTSC	BLK21306
IF (TOUTSC.GT.TMX2) TMX2=TOUTSC	BLK21307
IF (TOUTSC.LT.TMN2) TMN2=TOUTSC	BLK21308
IF (DPTOT.GT.DPMX) DPMX=DPTOT	BLK21309
IF (DPTOT.LT.DPMN) DPMN=DPTOT	BLK21310
940 ITEST=0	BLK21311
IF (SAVE1.GT.SAVE+.001) GO TO 918	BLK21312
SAVE1=SAVE1+PRINT	BLK21313
788 TOUT=TOUT-459.69	BLK21314
TOUTSC = TOUTSC - 459.69	BLK21315
TF(22)=TF(22)-459.69	BLK21316
TF(24)=TF(24)-459.69	BLK21317
ITEST=1	BLK21318
IF (MMM.EQ.0) GO TO 5051	BLK21319
PRHTR=ON	BLK21320

GO TO 5052	BLK21321
5051 PRHTR=OFF	BLK21322
5052 IF(M.EQ.0) GO TO 5053	BLK21323
SCHTR1=ON	BLK21324
IF(MM.EQ.0) GO TO 5054	BLK21325
SCHTR2=ON	BLK21326
GO TO 5055	BLK21327
5053 SCHTR1=OFF	BLK21328
5054 SCHTR2=OFF	BLK21329
5055 WRITE(6,1670) SAVE,QRJT,DPPRNT,WDOTS,TOUT,PRHTR,QRJTSC,DPTOT,	BLK21330
1 WDTSEC,TOUTSC,SCHTR1,SCHTR2,TOTREJ	BLK21331
1670 FORMAT(7X1PG9.3,G11.5,G10.5,G11.5,G10.4,A4,G14.5,G10.5,G11.5,	BLK21332
1 G10.4,A4,2XA4,G14.5)	BLK21333
WRITE(6,2600) RATE1,TF(22),RATE2,TF(24)	BLK21334
2600 FORMAT(27X'SIDE 1,2 '1PG11.5,G10.4/27X'SIDE 3,4 'G11.5,G10.4/)	BLK21335
IF(IQUIT.EQ.0.AND.MSSION.EQ.4) GO TO 787	BLK21336
IF(MSSION.EQ.4) RETURN	BLK21337
IF(IQUIT.EQ.1) GO TO 789	BLK21338
GO TO 920	BLK21339
918 IF(MSSION.NE.4) GO TO 920	BLK21340
787 IF(ABS(TOTREJ-QREJ1).LT..5) GO TO 790	BLK21341
QREJ1=TOTREJ	BLK21342
GO TO 920	BLK21343
790 WRITE(6,784)	BLK21344
784 FORMAT(15X'STEADY STATE ATTAINED.'//)	BLK21345
GO TO 791	BLK21346
920 IF(.NOT.SAVE.LT.P3-.01) GO TO 9001	BLK21347
SAVE = SAVE + .02	BLK21348
TAU=TAU+0.02	BLK21349
GO TO 425	BLK21350
9001 IF(MSSION.NE.4) GO TO 791	BLK21351
WRITE(6,1097)	BLK21352
1097 FORMAT(15X'THREE HUNDRED ITERATIONS HAVE FAILED TO PRODUCE A STEADY	BLK21353
1Y STATE SOLUTION.'//15X'LAST VALUES OBTAINED'//)	BLK21354
791 IQUIT=1	BLK21355
IF(MSSION.NE.4) GO TO 786	BLK21356
IF(ITEST.EQ.1) GO TO 5055	BLK21357
GO TO 788	BLK21358
786 IF(ITEST.EQ.0) GO TO 788	BLK21359
789 IF(KODSEC.EQ.0) GO TO 400	BLK21360
TAVG2=STOUT2/COUNT	BLK21361
TMX2=TMX2-459.69	BLK21362
TMN2=TMN2-459.69	BLK21363
TAVG2=TAVG2-459.69	BLK21364
DPAVG2=SDP2/COUNT	BLK21365
400 HAVG=SQREJ/COUNT	BLK21366
DPAVG=SDP/COUNT	BLK21367
TAVG=STOUT/COUNT	BLK21368
TMX=TMX-459.69	BLK21369
TMN=TMN-459.69	BLK21370
TAVG=TAVG-459.69	BLK21371
5061 PCMAX=100.*PCMAX	BLK21372
PCMIN=100.*PCMIN	BLK21373
WRITE(6,6001)	BLK21374
6001 FORMAT('1'52X'MAXIMUM'5X'MINIMUM'5X'AVERAGE'//)	BLK21375
WRITE(6,6002) TMAX,TMIN,HAVG	BLK21376
6002 FORMAT('0'9X'TOTAL HEAT REJECTION RATE, BTU/HR'8X1P3G12.5/)	BLK21377
WRITE(6,6003) DPMAX,DPMIN,DPAVG,DPMX,DPMN,DPAVG2	BLK21378
6003 FORMAT('0'9X'PRESSURE DROP, PSI'//15X'PRIMARY SYSTEM'22X1P3G12.5/	BLK21379
1 15X'REDUNDANT SYSTEM'20X3G12.5/)	BLK21380

	WRITE(6,6004) TMX, TMN, TAVG, TMX2, TMN2, TAVG2	BLK21381
6004	FORMAT('0'9X'OUTLET TEMPERATURE, DEG. F'/15X'PRIMARY SYSTEM'22X	BLK21382
1	P3G12.4/15X'REDUNDANT SYSTEM'20X3G12.4/)	BLK21383
	WRITE(6,6005) PCMAX, PCMIN	BLK21384
6005	FORMAT('0'9X'PRIMARY SYSTEM BYPASS, PERCENT'11X1P2G12.5/)	BLK21385
	WRITE(6,6006) SUM1, SUM2	BLK21386
6006	FORMAT('0'9X'TOTAL INLINE HEATER POWER DISSIPATION, BTU'/15X	BLK21387
1	'PRIMARY SYSTEM'22X1PG12.5/15X'REDUNDANT SYSTEM'20XG12.5/)	BLK21388
	RETURN	BLK21389
	END	BLK21390
	SUBROUTINE VPOL(TVIS, BMU, NP3)	BLK21391
C	THIS SUBROUTINE FIND A FLUID VISCOSITY FOR A GIVEN TEMPERATURE	BLK21392
C	FROM A CURVE OF TEMPERATURE VS. VISCOSITY	BLK21393
	DIMENSION T(33), TMU(33)	BLK21394
	COMMON T, TMU	BLK21395
	J=1	BLK21396
	IF(TVIS-T(1)) 6,4,7	BLK21397
7	IF(TVIS-T(NP3)) 1,9,8	BLK21398
1	DO 2 I=2, NP3	BLK21399
	J=1	BLK21400
	IF(TVIS-T(I)) 3,4,2	BLK21401
2	CONTINUE	BLK21402
3	TEMP=TVIS-T(J-1)	BLK21403
	TINT=T(J)-T(J-1)	BLK21404
	TEMPMU=TMU(J)-TMU(J-1)	BLK21405
	BMU=TMU(J-1)+(TEMP/TINT)*TEMPMU	BLK21406
	RETURN	BLK21407
4	BMU=TMU(J)	BLK21408
	RETURN	BLK21409
6	WRITE(6,200)	BLK21410
200	FORMAT(////99H THE TEMPERATURE USED IN FINDING THE VISCOSITY IS	BLK21411
1	LESS THAN THE LOWEST TEMPERATURE ON THE CURVE.////)	BLK21412
	CALL EXIT	BLK21413
8	WRITE(6,201)	BLK21414
201	FORMAT(////97H THE TEMPERATURE USED IN FINDING THE VISCOSITY IS	BLK21415
1	GREATER THAN THE HIGHEST VALUE ON THE CURVE.////)	BLK21416
	CALL EXIT	BLK21417
9	BMU=TMU(NP3)	BLK21418
	RETURN	BLK21419
	END	BLK21420
01420		

APPENDIX B
PROGRAM FLOW CHART

**ULTRA-FAST MISSION ANALYSIS ROUTINE
FOR APOLLO BLOCK 2 ENVIRONMENTAL
CONTROL SYSTEM RADIATORS**

DEVELOPED BY

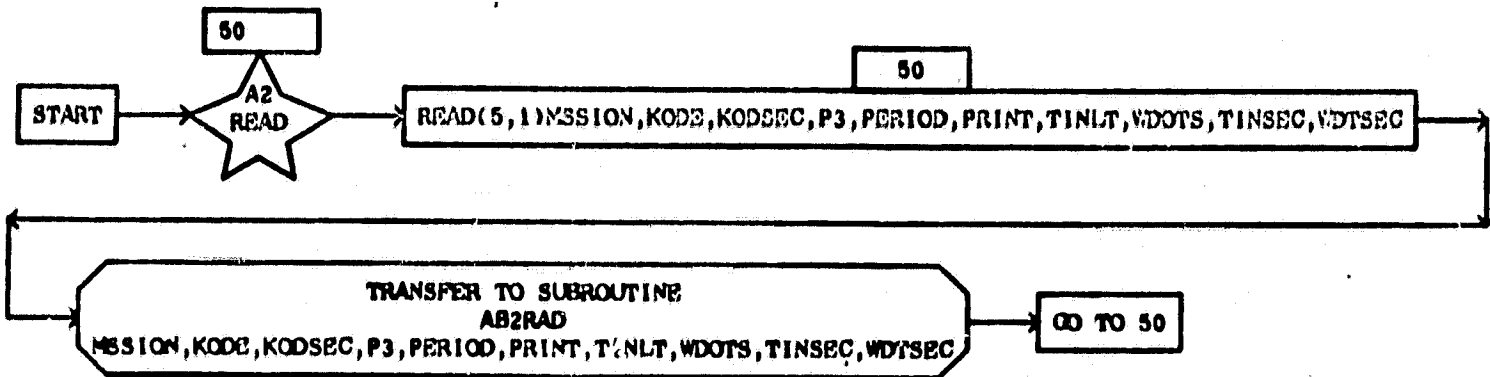
**MISSILES AND SPACE DIVISION - TEXAS
LTV AEROSPACE CORPORATION
P. O. BOX 6267 - DALLAS, TEXAS 75222**

FOR

**NATIONAL AERONAUTICS AND SPACE ADMINISTRATION
MANNED SPACECRAFT CENTER - HOUSTON, TEXAS
UNDER CONTRACT NAS9-6807**

**PROG READS THE DATA AND CALLS SUBROUTINE AB2RAD WHICH DOES
THE ACTUAL CALCULATIONS**

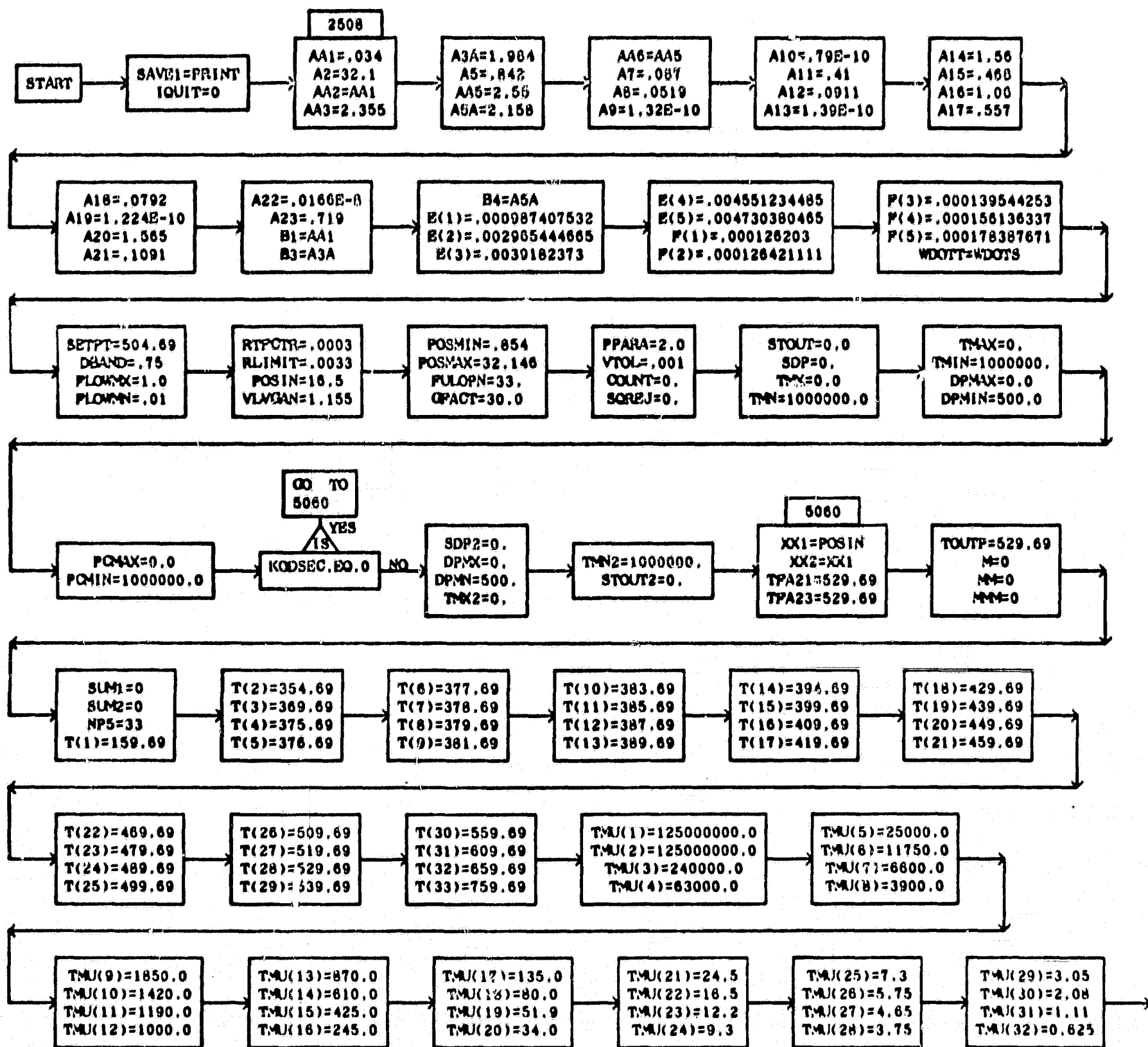
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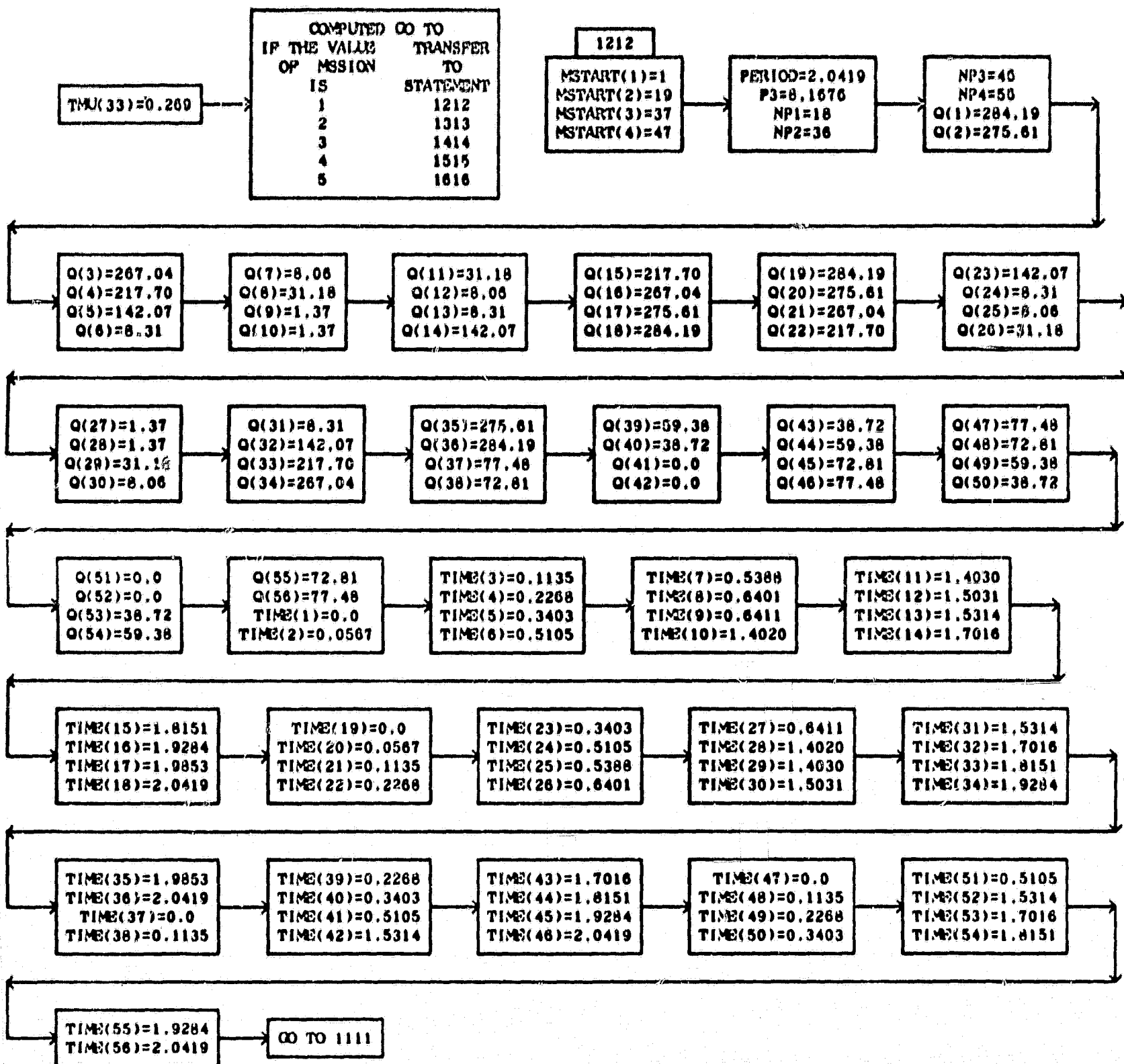


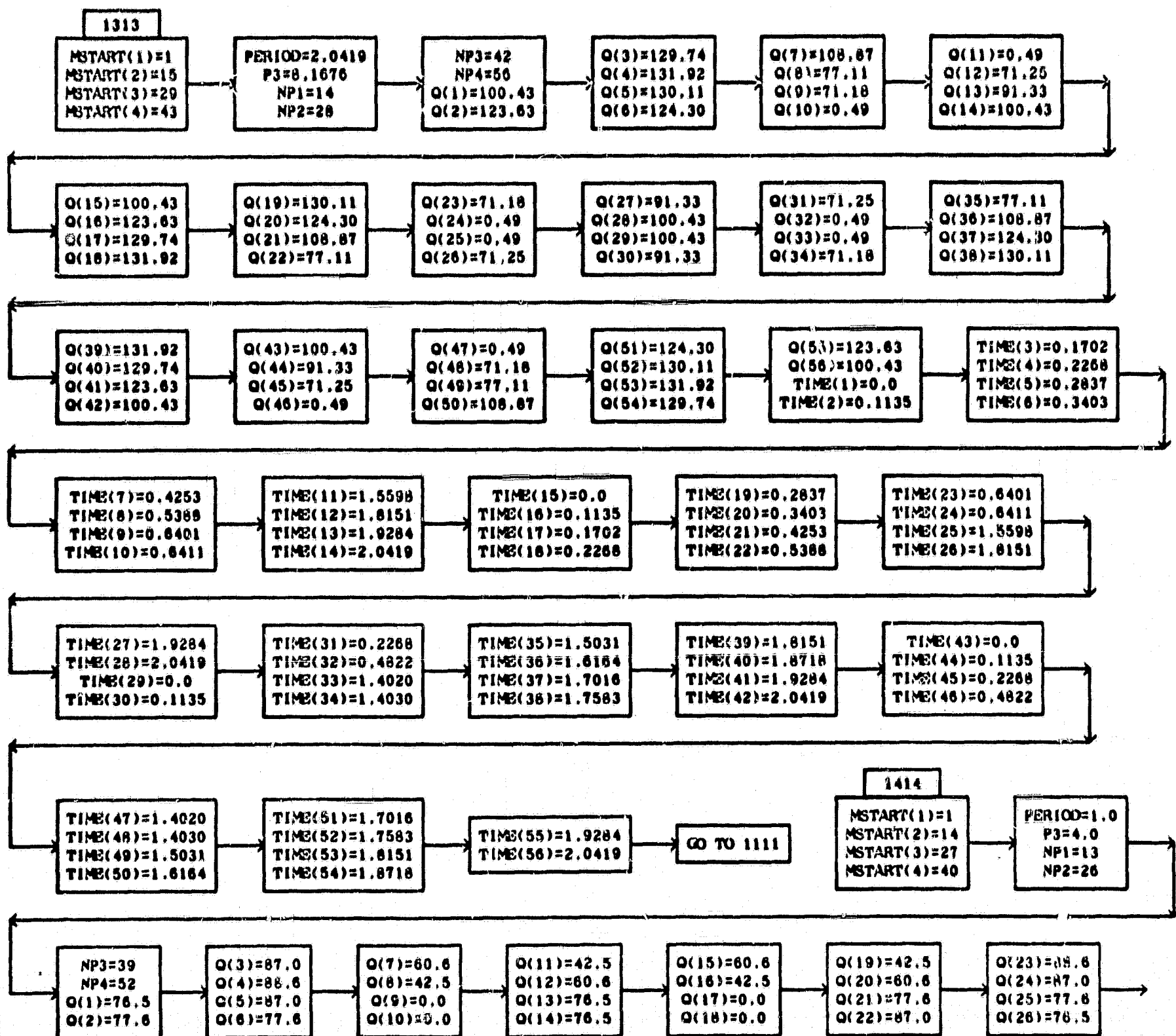
DIMENSIONED VARIABLES

SYMBOL	STORAGES	SYMBOL	STORAGES	SYMBOL	STORAGES	SYMBOL	STORAGES	SYMBOL	STORAGES
AK	12	BMU	37	TP	32	TPIN	32	TPPP	32
TPP	32	NFOODE	32	TT	32	TTIN	32	TPPT	32
TPT	32	NTCODE	32	TS	20	TSIN	20	TPPS	20
TPS	20	NSCODE	20	QABS	4	Q	4000	TIME	6000
T	33	TMU	33	WDOT	10	DPT	10	B	5
ALPHA	12	NPTS	8	WDTTOT	1000	TIN	1000	MSTART	8
WDOTSC	1000	TINLSC	100	H	10	P	5		

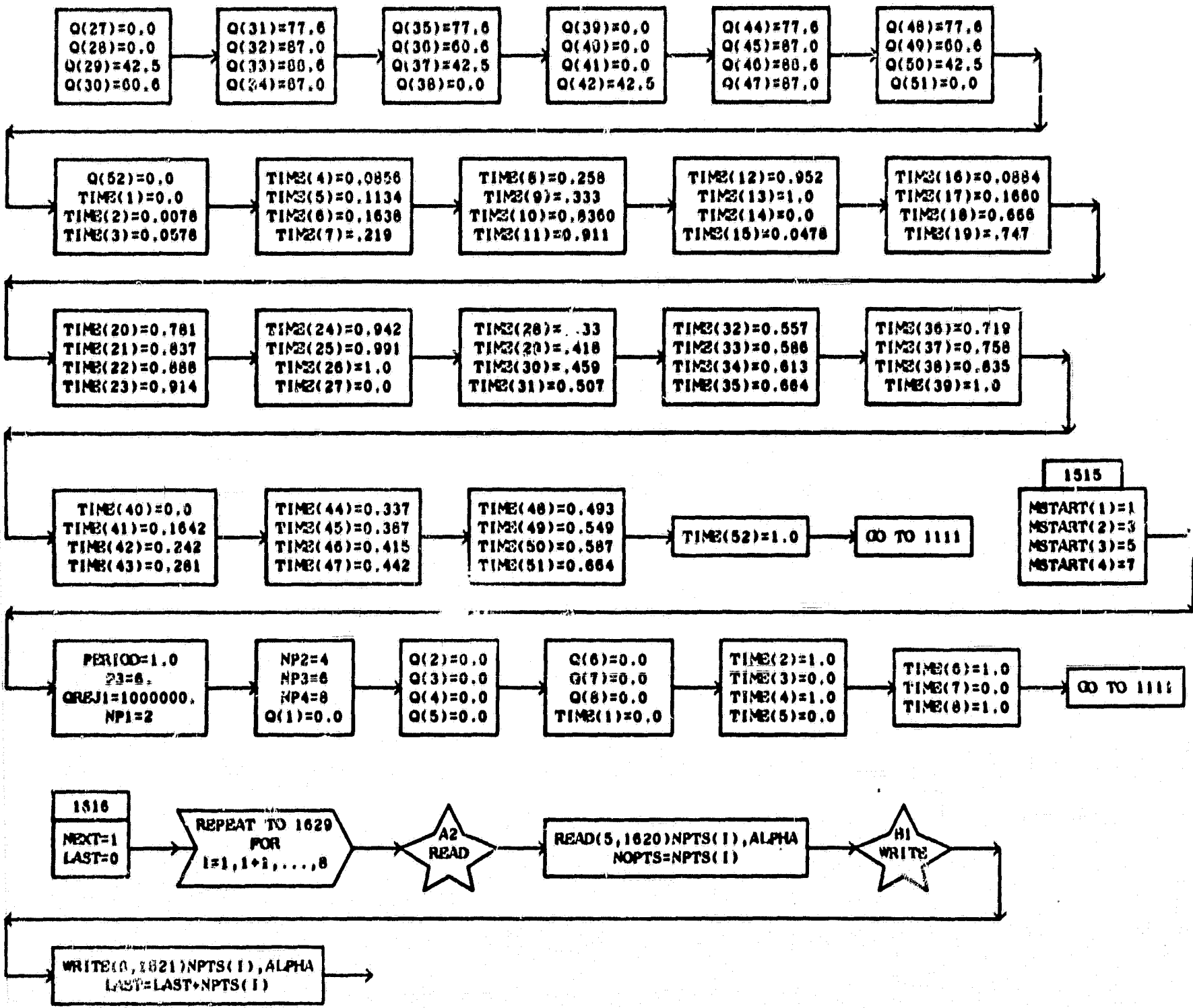
SUBROUTINE AB2RAD (MISSION,KODE,KODSEC,P3,PERIOD,PRINT,TINLT,



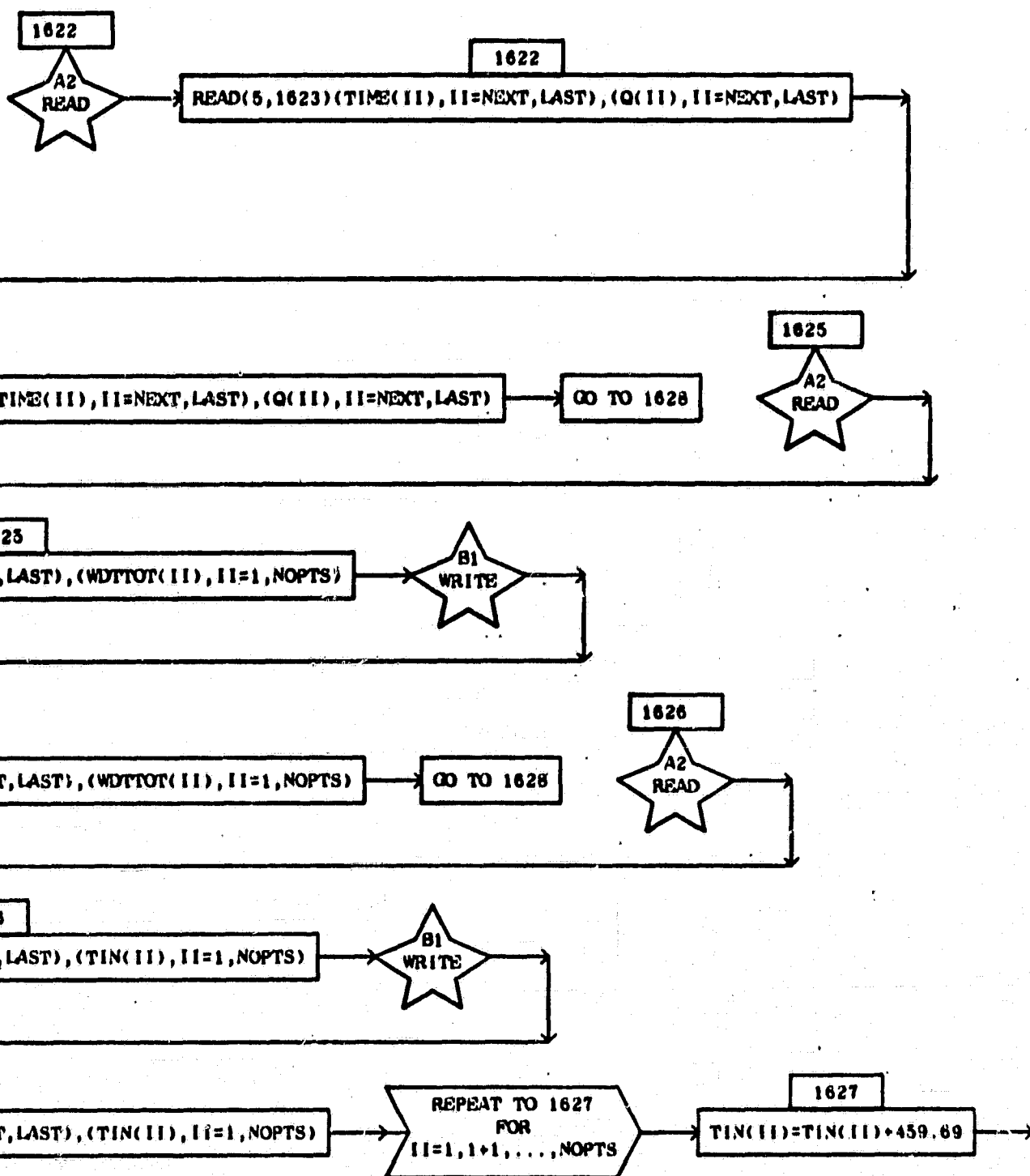


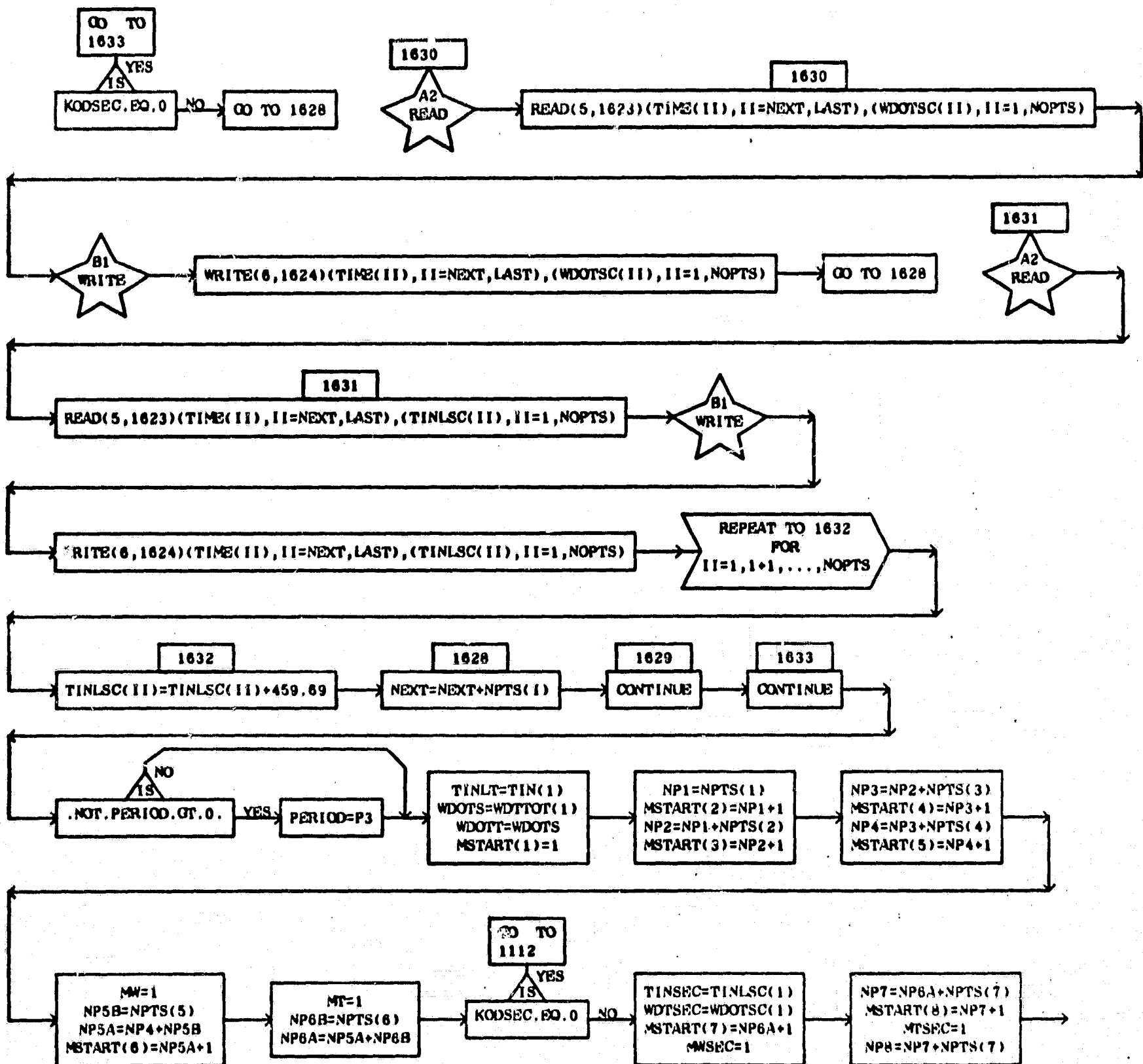


III



COMPUTED GO TO IF THE VALUE OF I		
IS	TRANSFER TO STATEMENT	
1	1622	
2	1622	
3	1622	
4	1622	
5	1625	
6	1626	
7	1630	
8	1631	





COMPUTED GO TO IF THE VALUE OF K IS			TRANSFER TO STATEMENT	
1			432	
2			433	
3			434	
4			435	

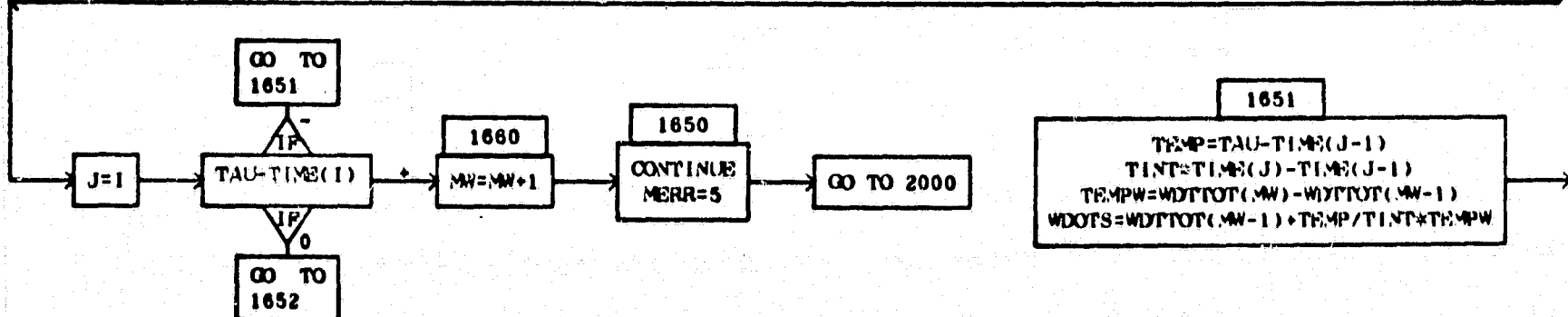
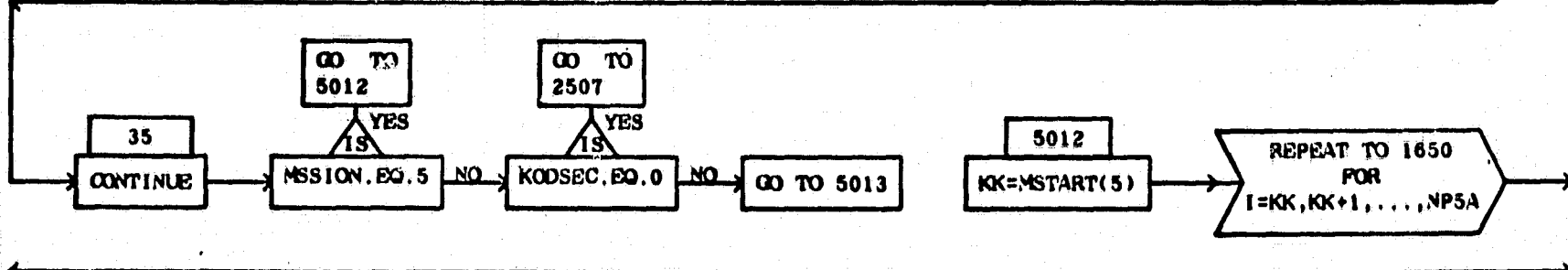
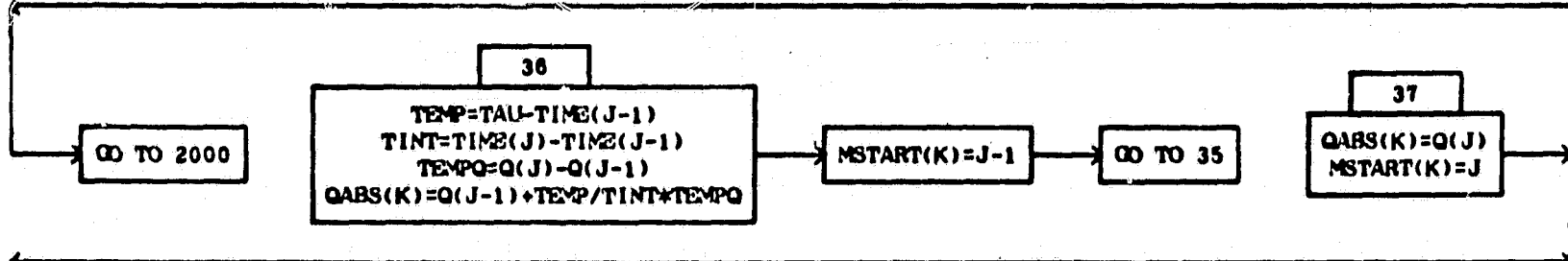
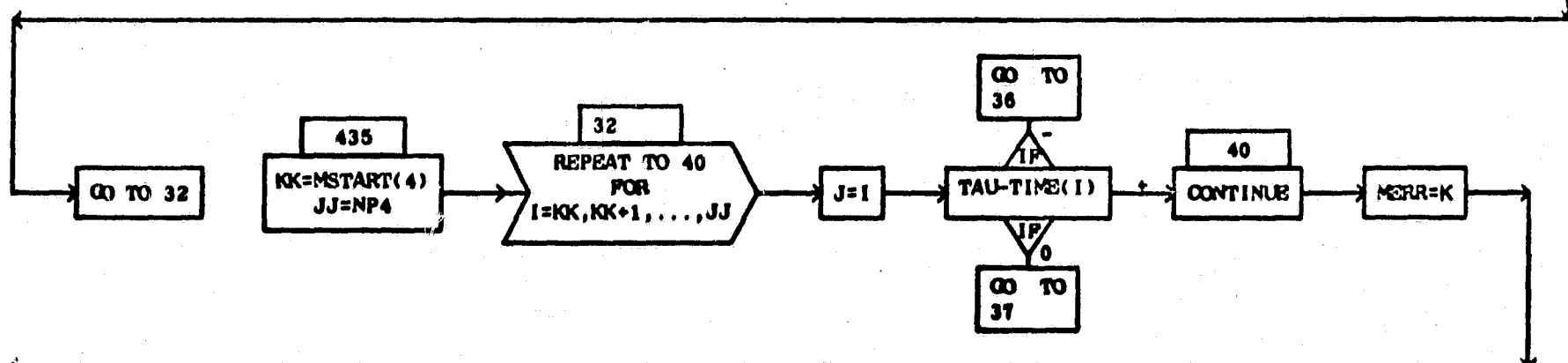
432
KK=MSTART(1)
JJ=NP1

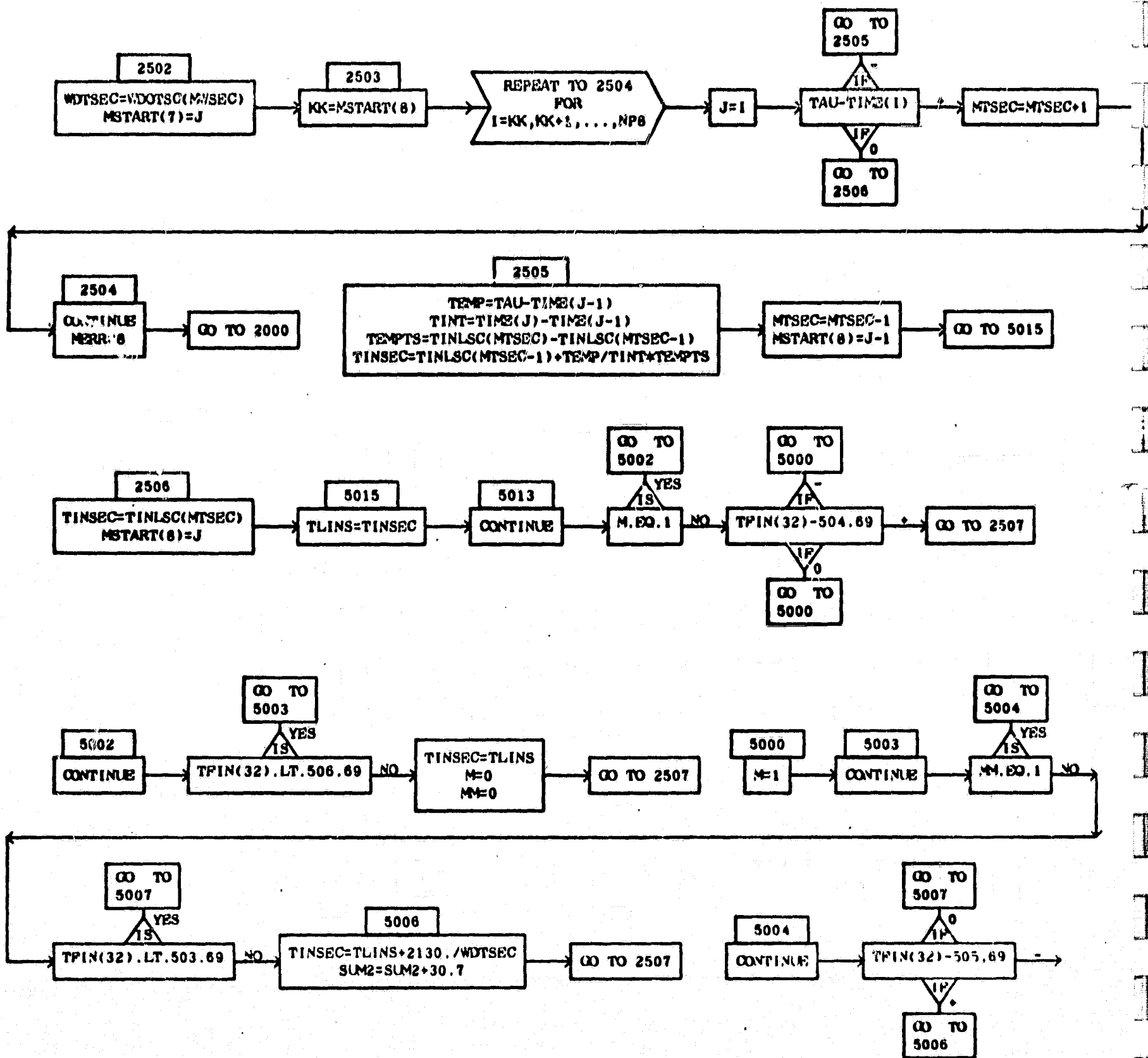
GO TO 32

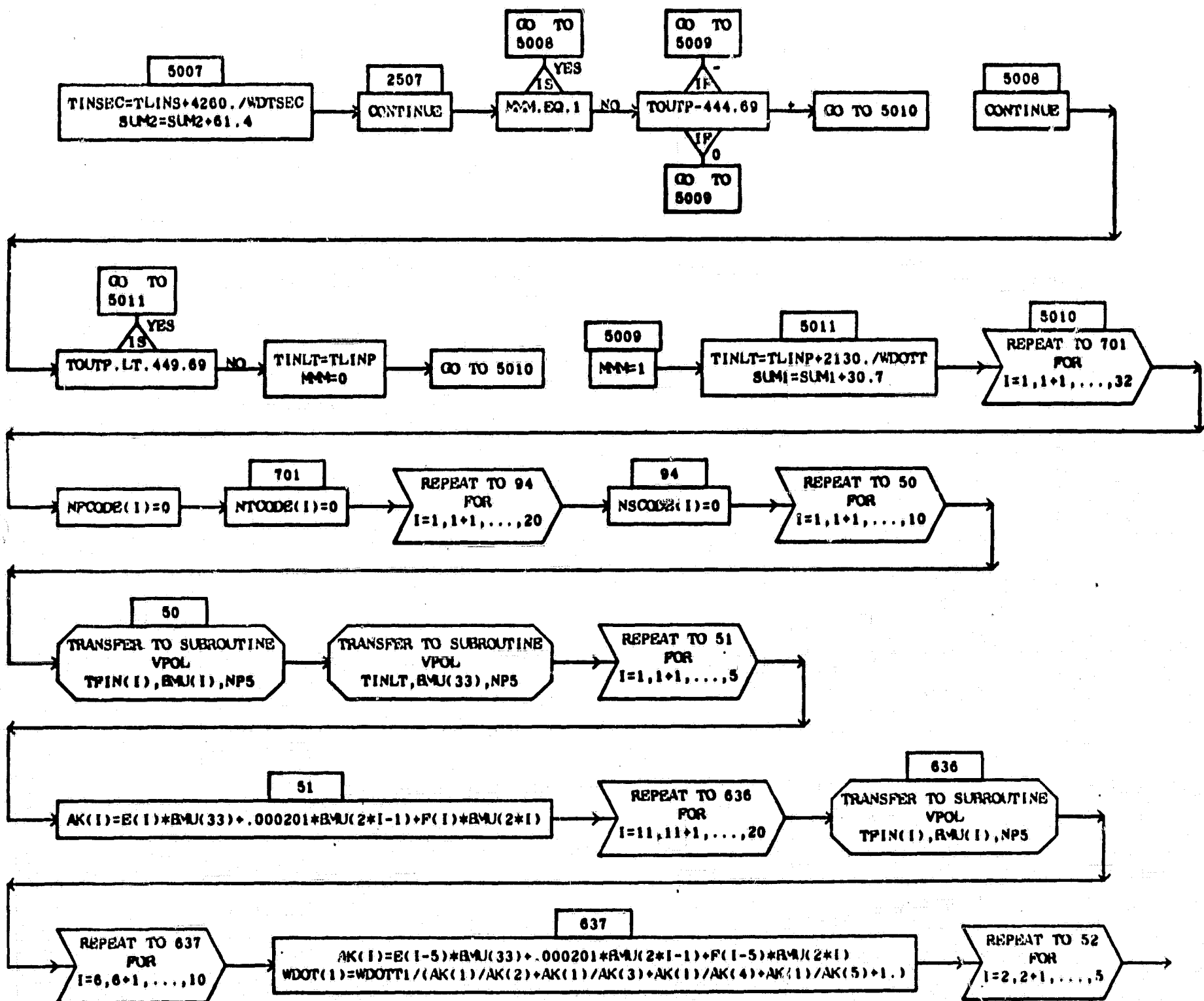
433
KK=MSTART(2)
JJ=NP2

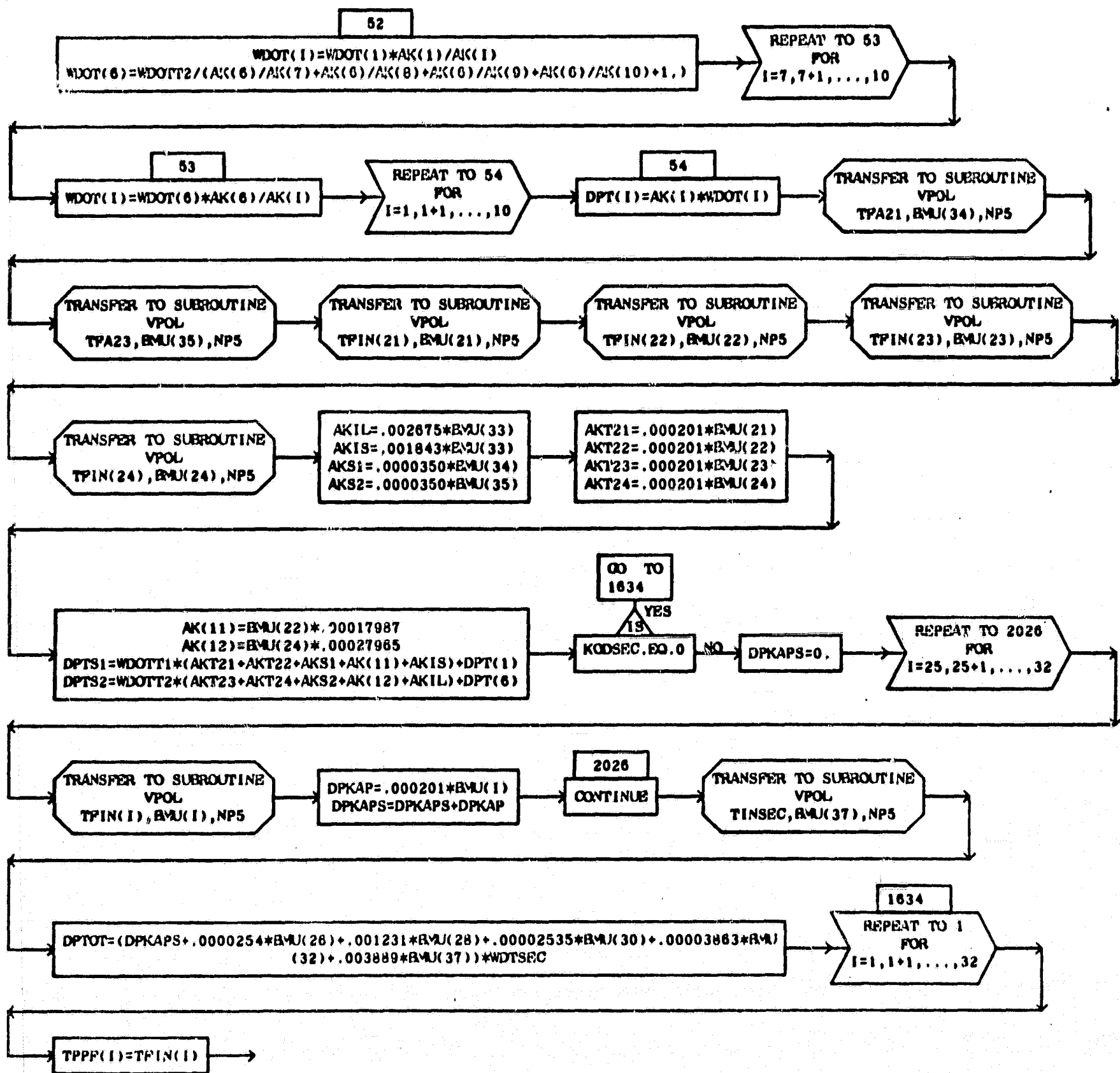
GO TO 32

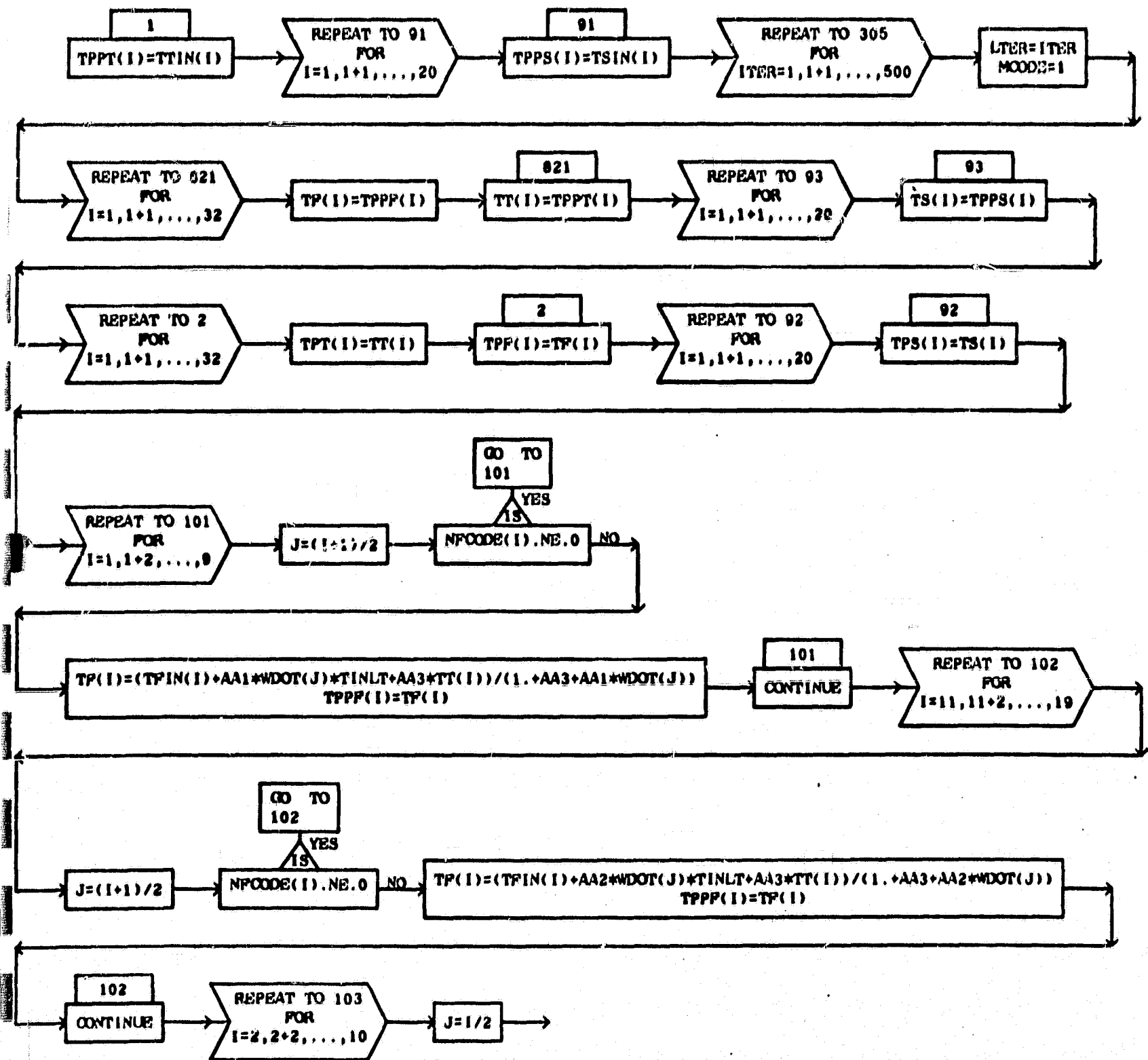
434
KK=MSTART(3)
JJ=NP3

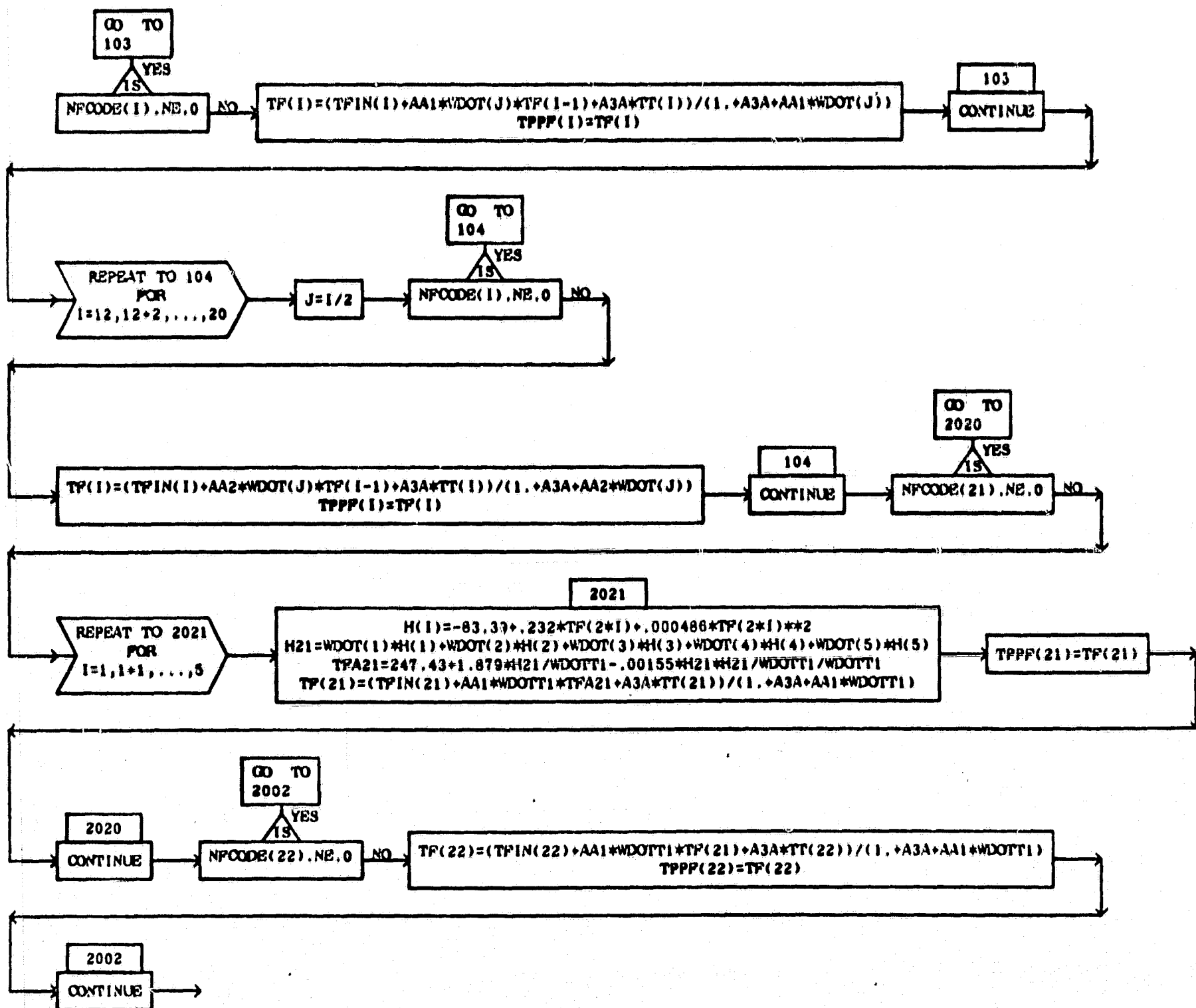


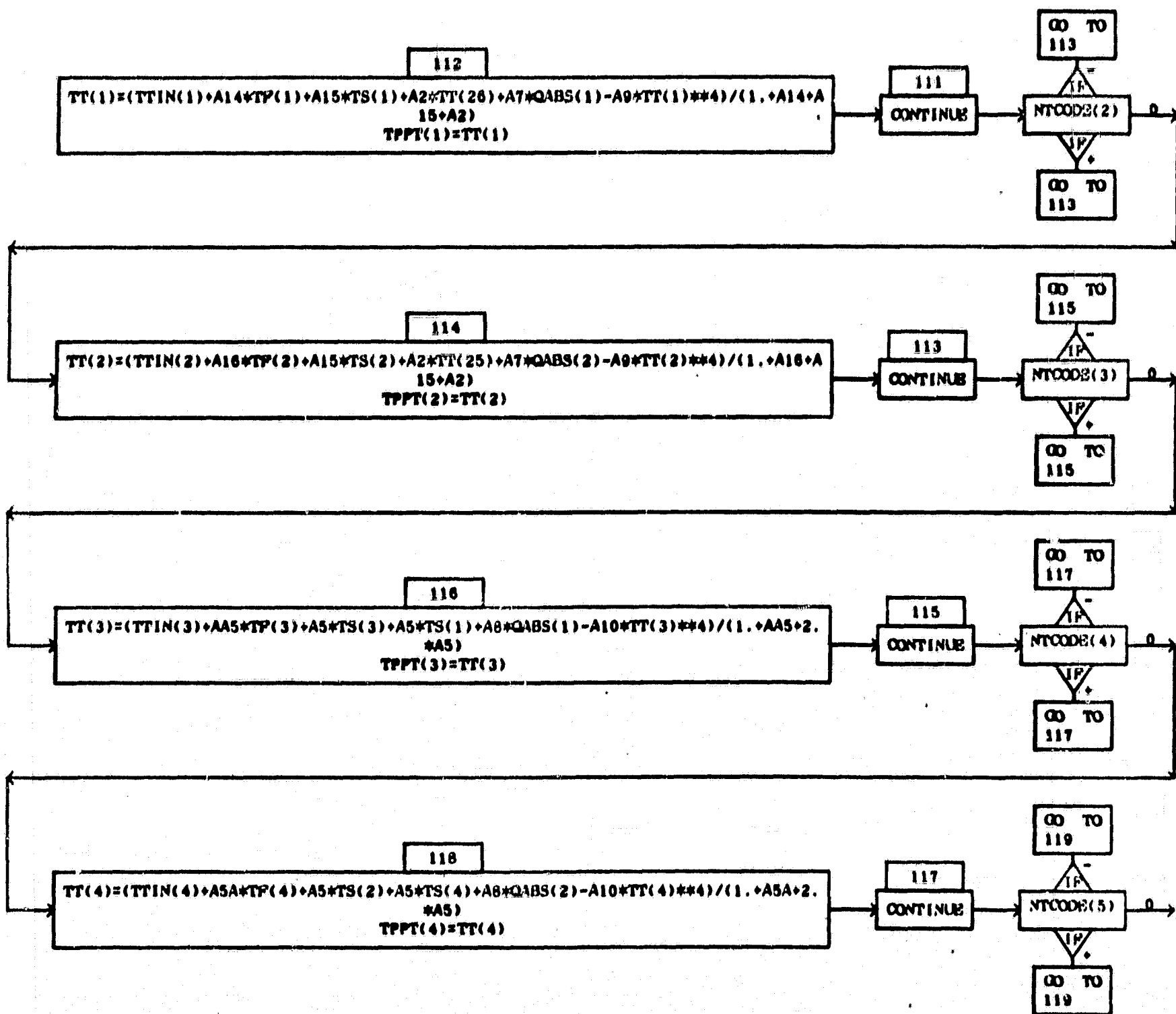


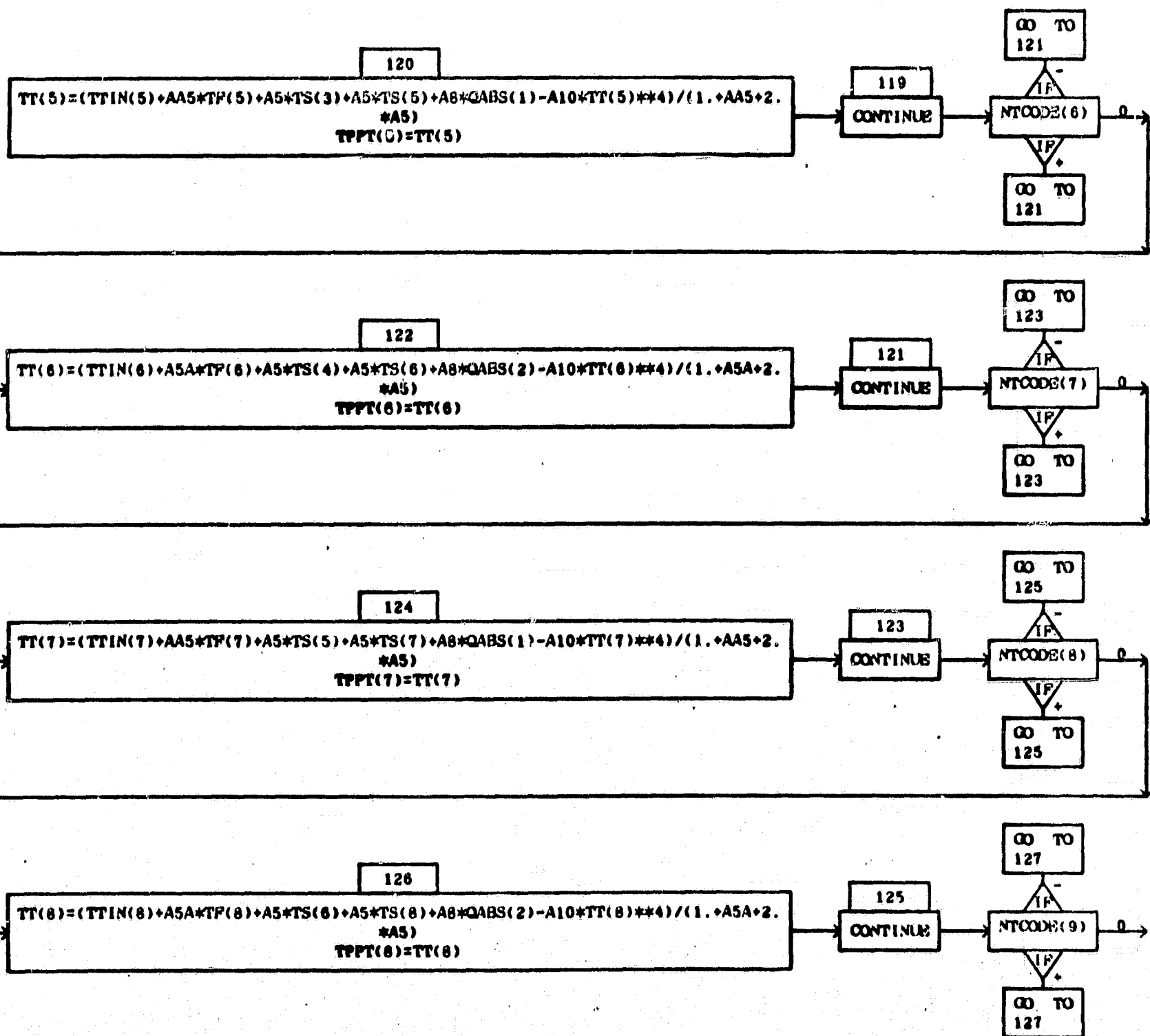




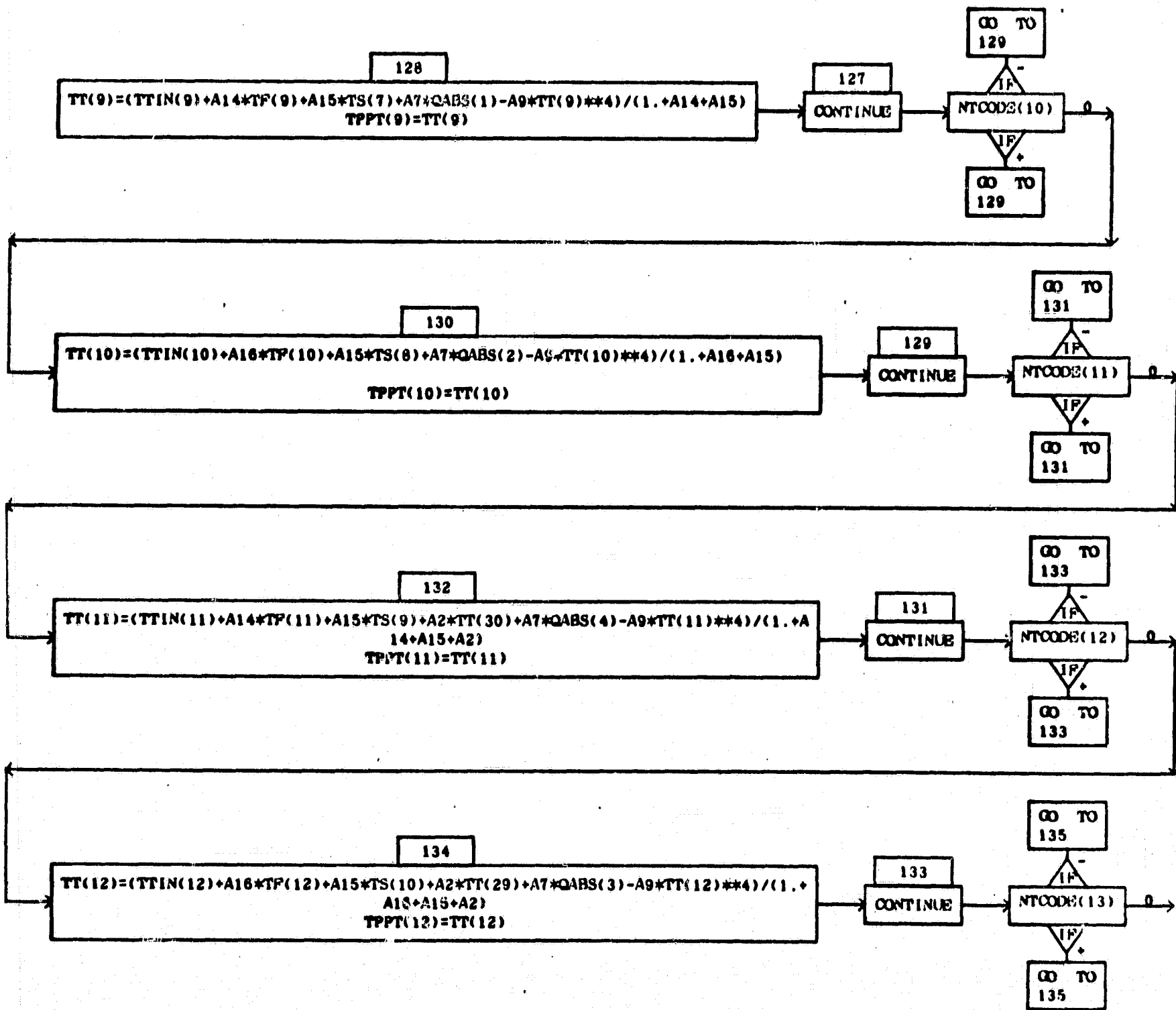








III



136

$$TT(13) = (TTIN(13) + AA0 * TP(13) + A5 * TS(9) + A5 * TS(11) + A8 * QABS(4) - A10 * TT(13) ** 4) / (1. + A8 + 2. * A5)$$

$$TPPT(13) = TT(13)$$

135

CONTINUE

GO TO
137IF
NTCODE(14)IF
+GO TO
137

138

$$TT(14) = (TTIN(14) + A5A * TP(14) + A5 * TS(10) + A5 * TS(12) + A8 * QABS(3) - A10 * TT(14) ** 4) / (1. + A5A + 2. * A5)$$

$$TPPT(14) = TT(14)$$

137

CONTINUE

GO TO
139IF
NTCODE(15)IF
+GO TO
139

140

$$TT(15) = (TTIN(15) + AA6 * TP(15) + A5 * TS(11) + A5 * TS(13) + A8 * QABS(4) - A10 * TT(15) ** 4) / (1. + AA6 + 2. * A5)$$

$$TPPT(15) = TT(15)$$

139

CONTINUE

GO TO
141IF
NTCODE(16)IF
+GO TO
141

142

$$TT(16) = (TTIN(16) + A5A * TP(16) + A5 * TS(12) + A5 * TS(14) + A8 * QABS(3) - A10 * TT(16) ** 4) / (1. + A5A + 2. * A5)$$

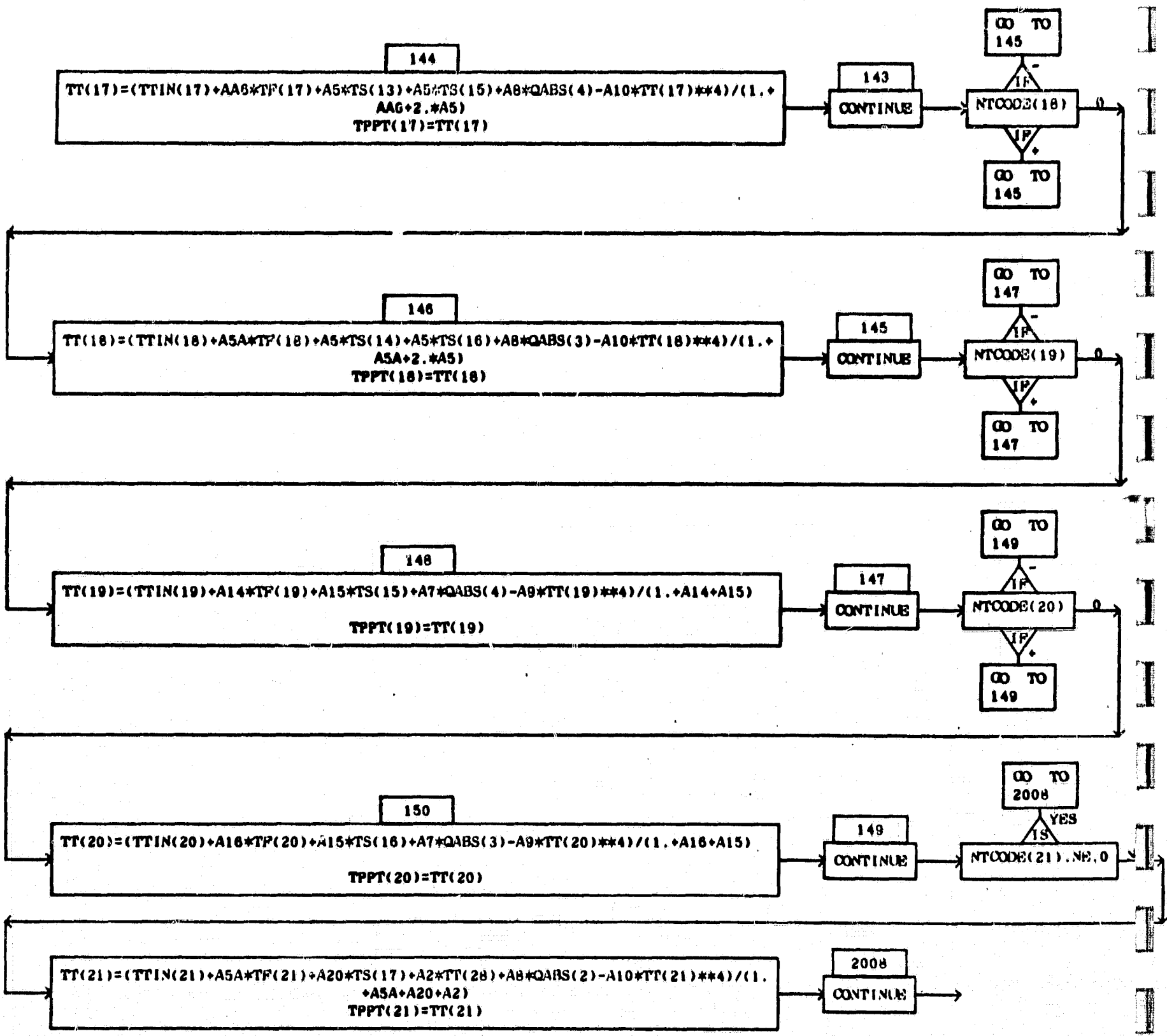
$$TPPT(16) = TT(16)$$

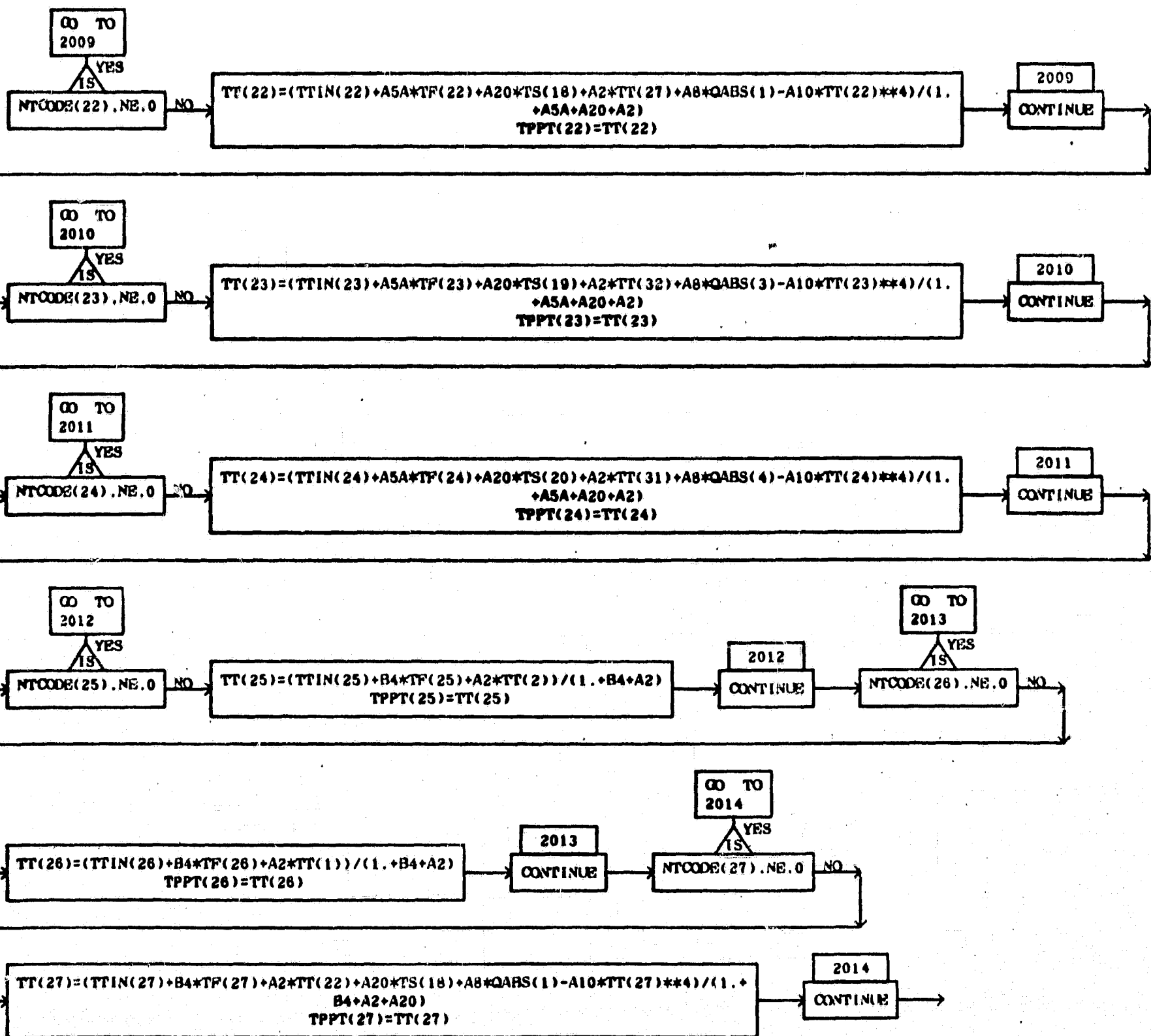
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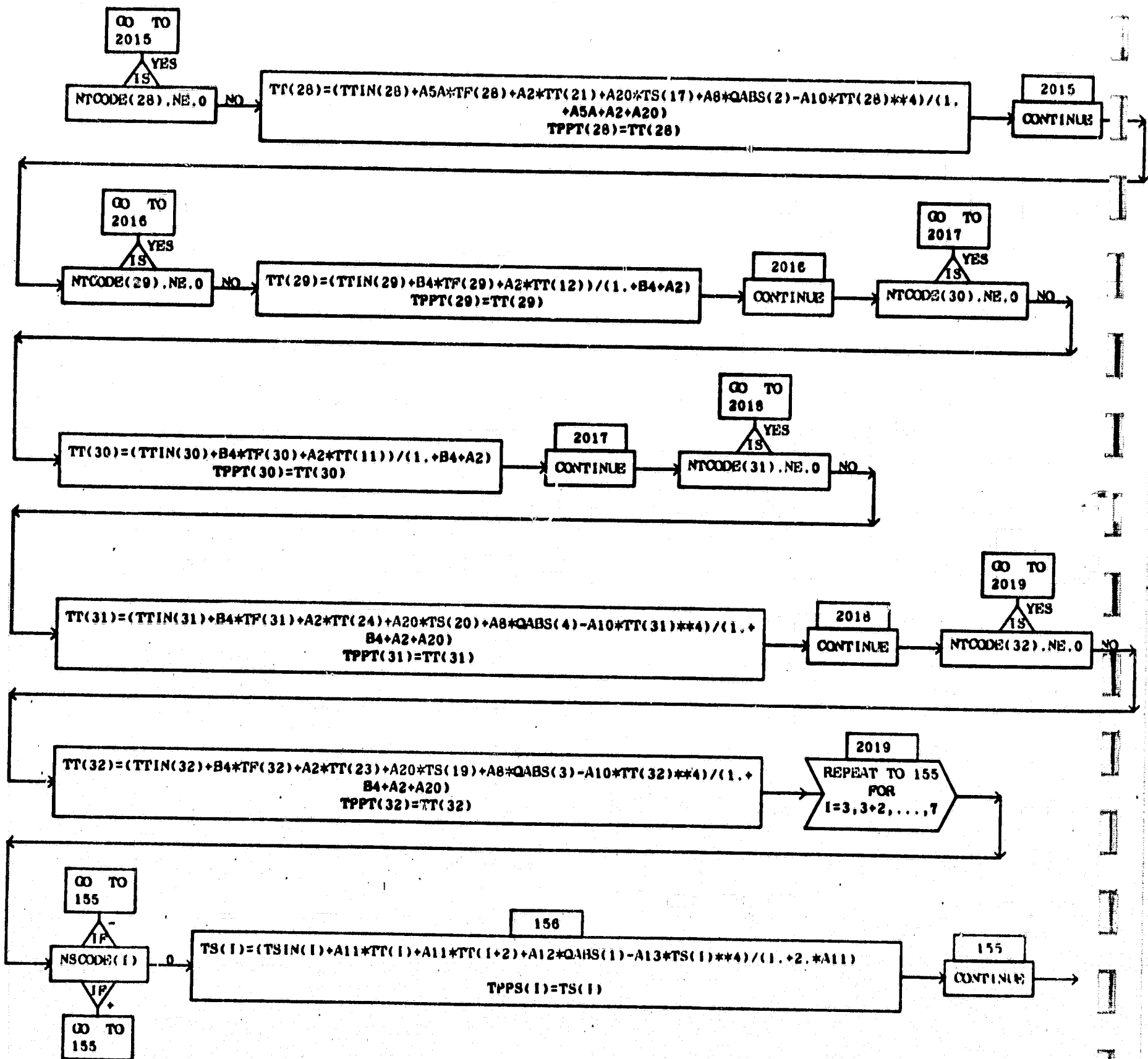
CONTINUE

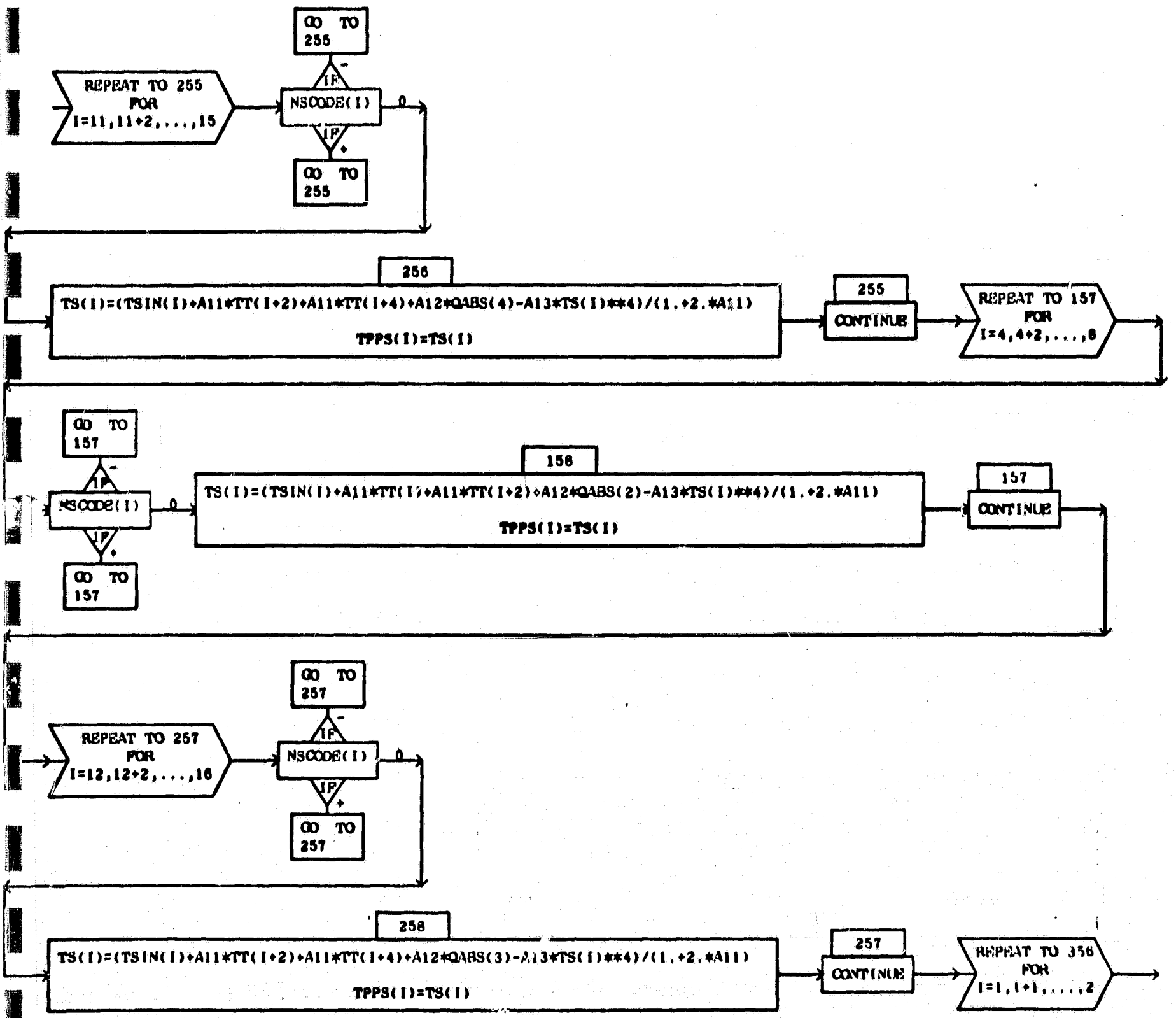
GO TO
143IF
NTCODE(17)IF
+GO TO
143

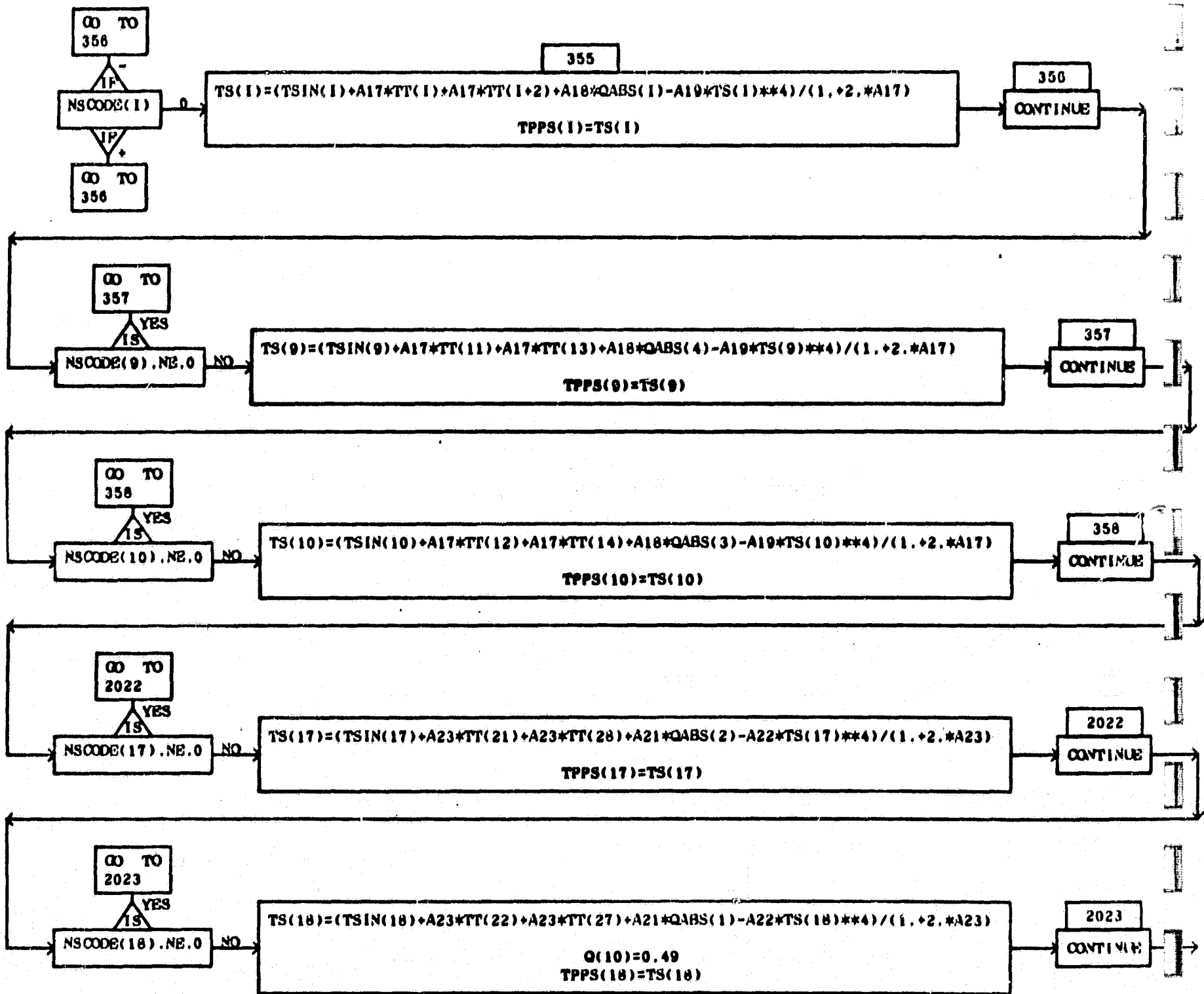
III

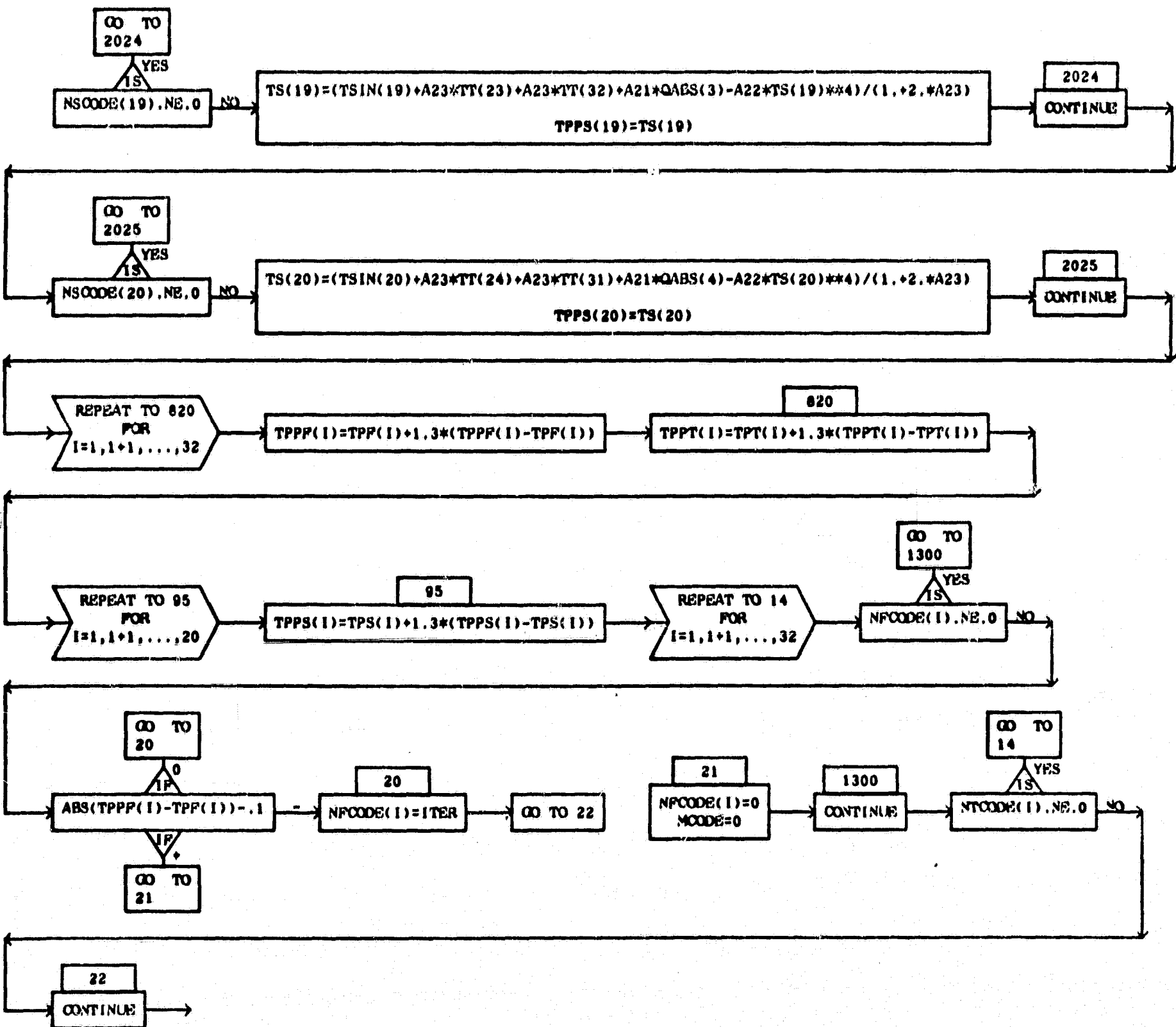


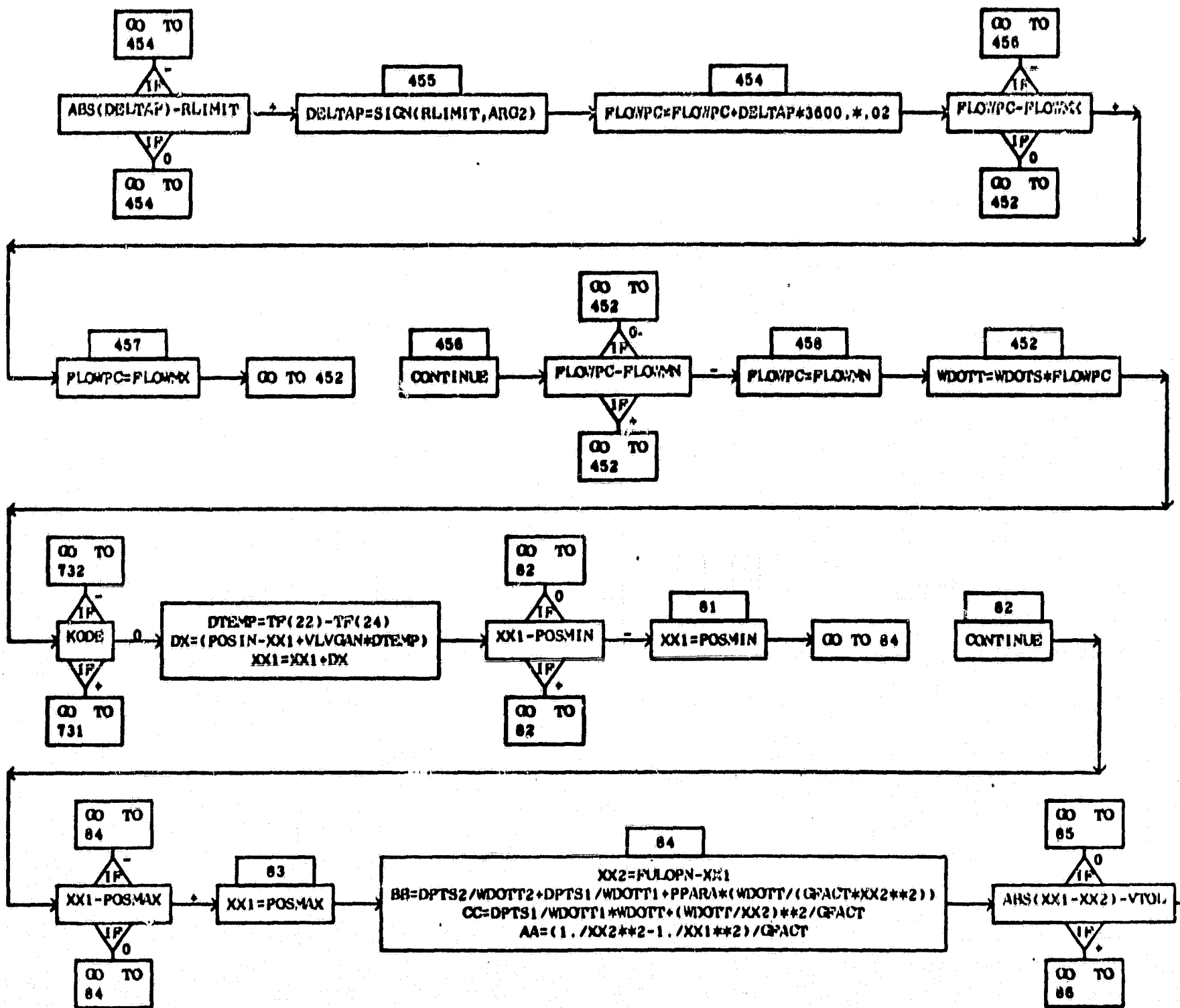


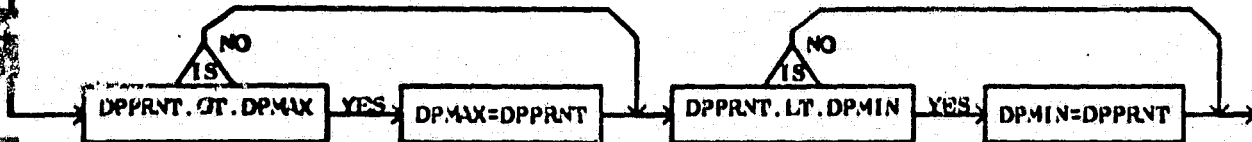
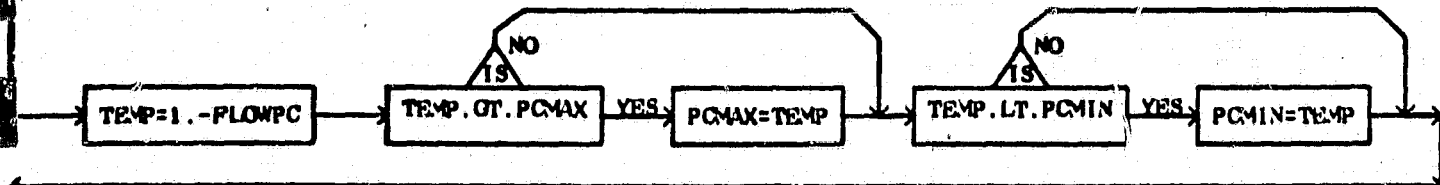
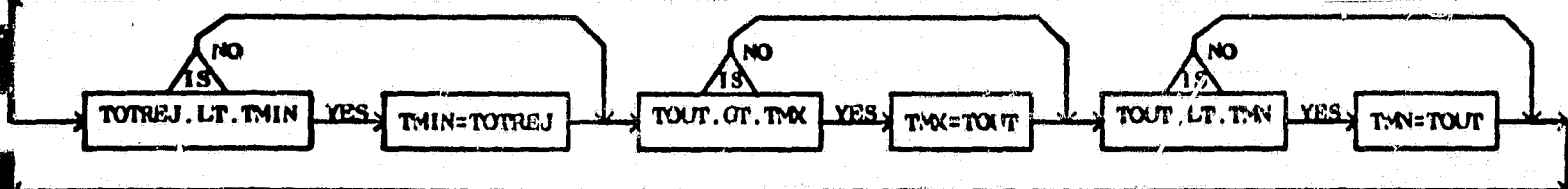
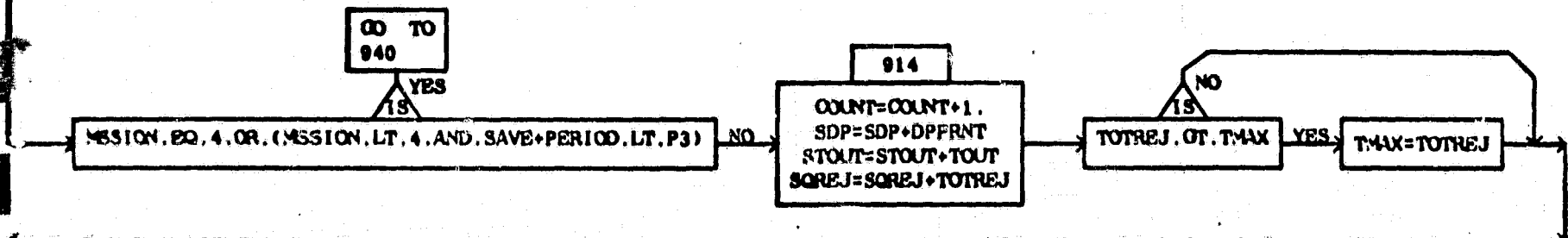
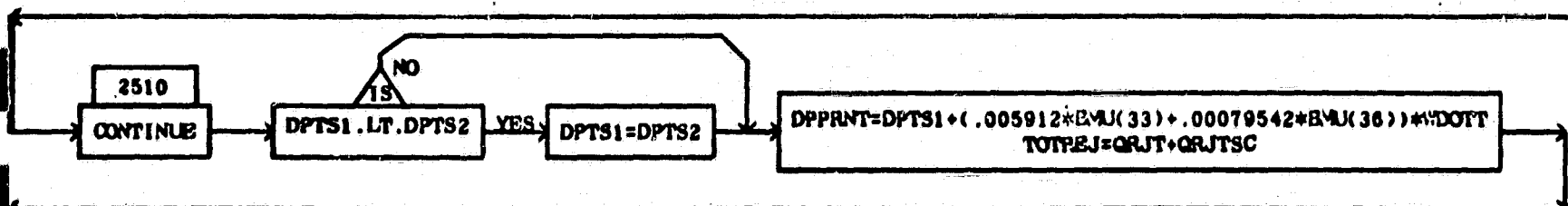
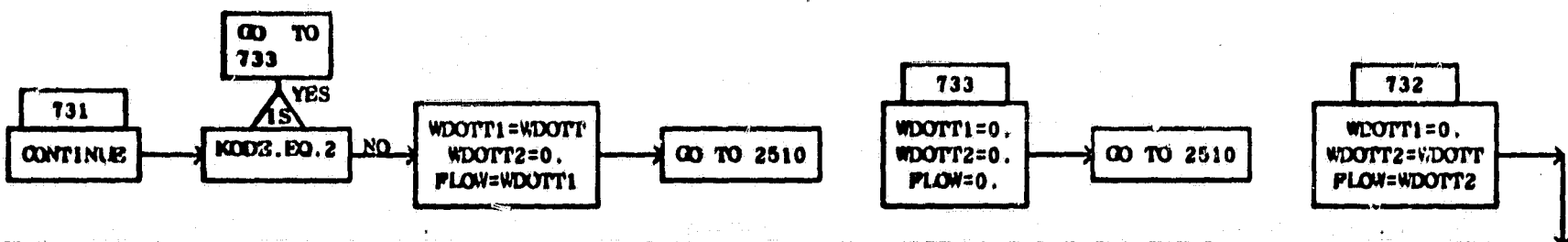
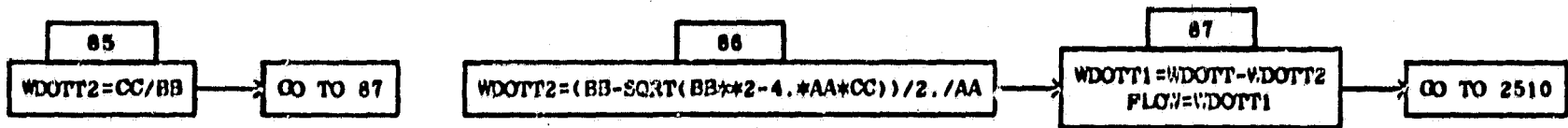


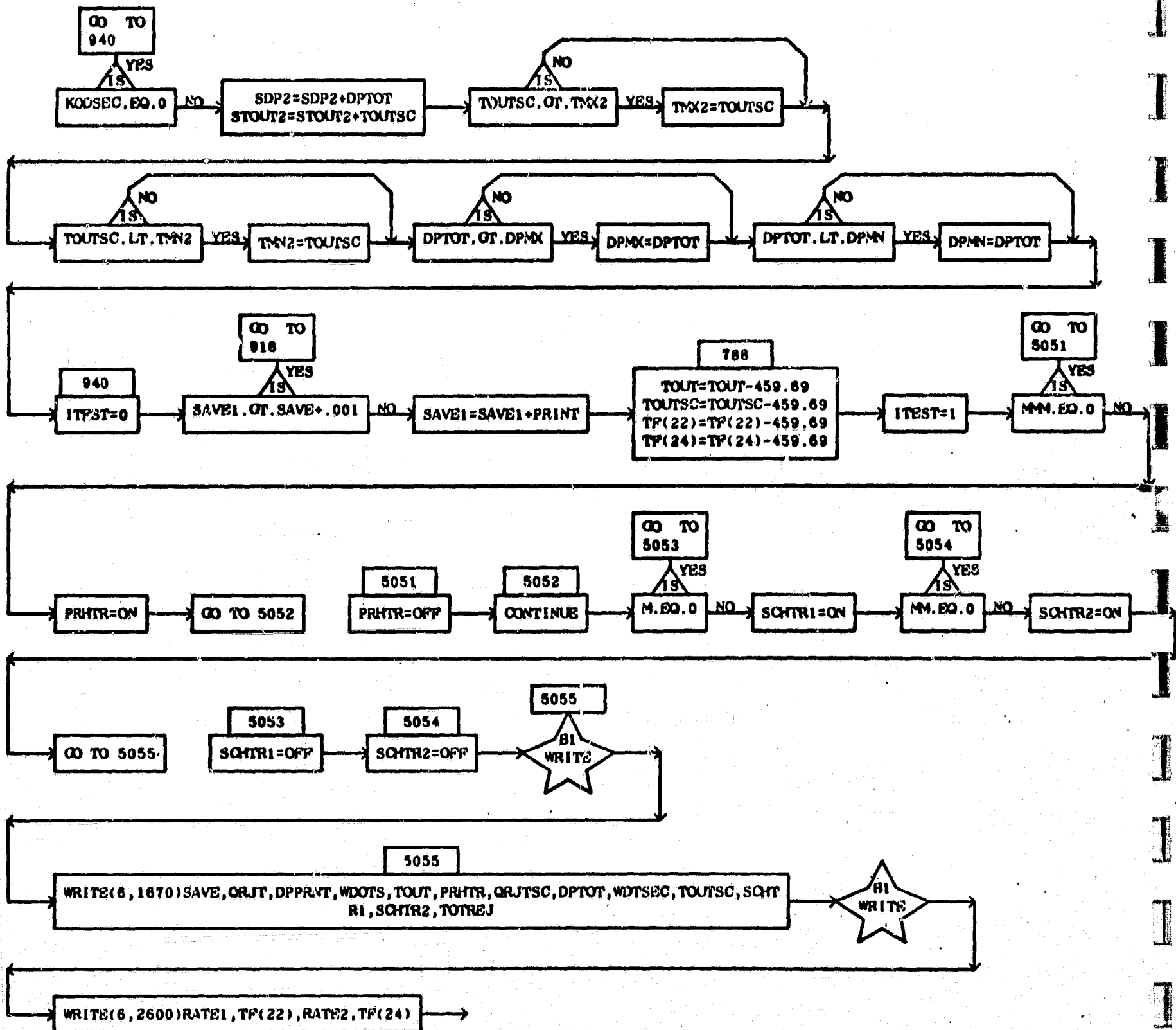


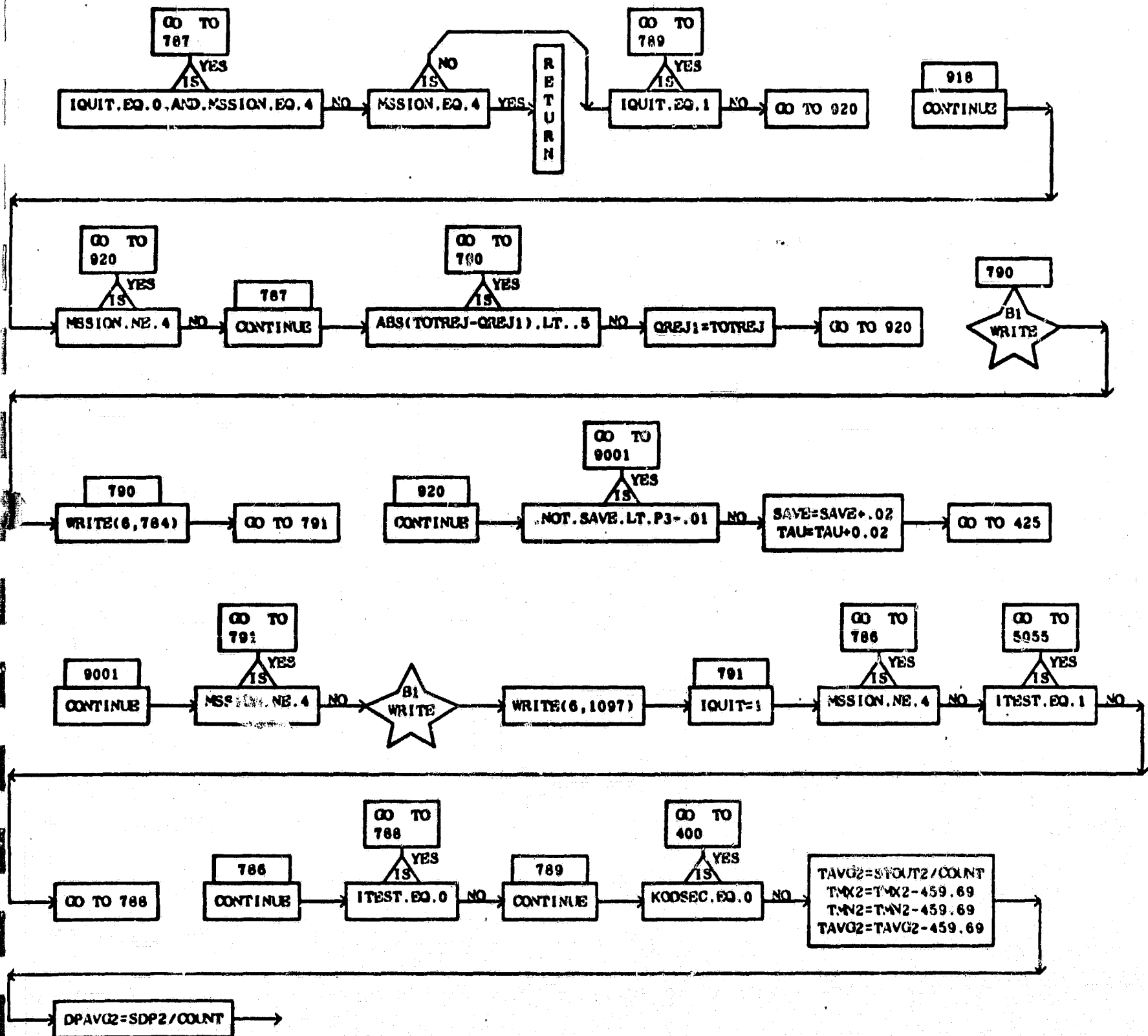


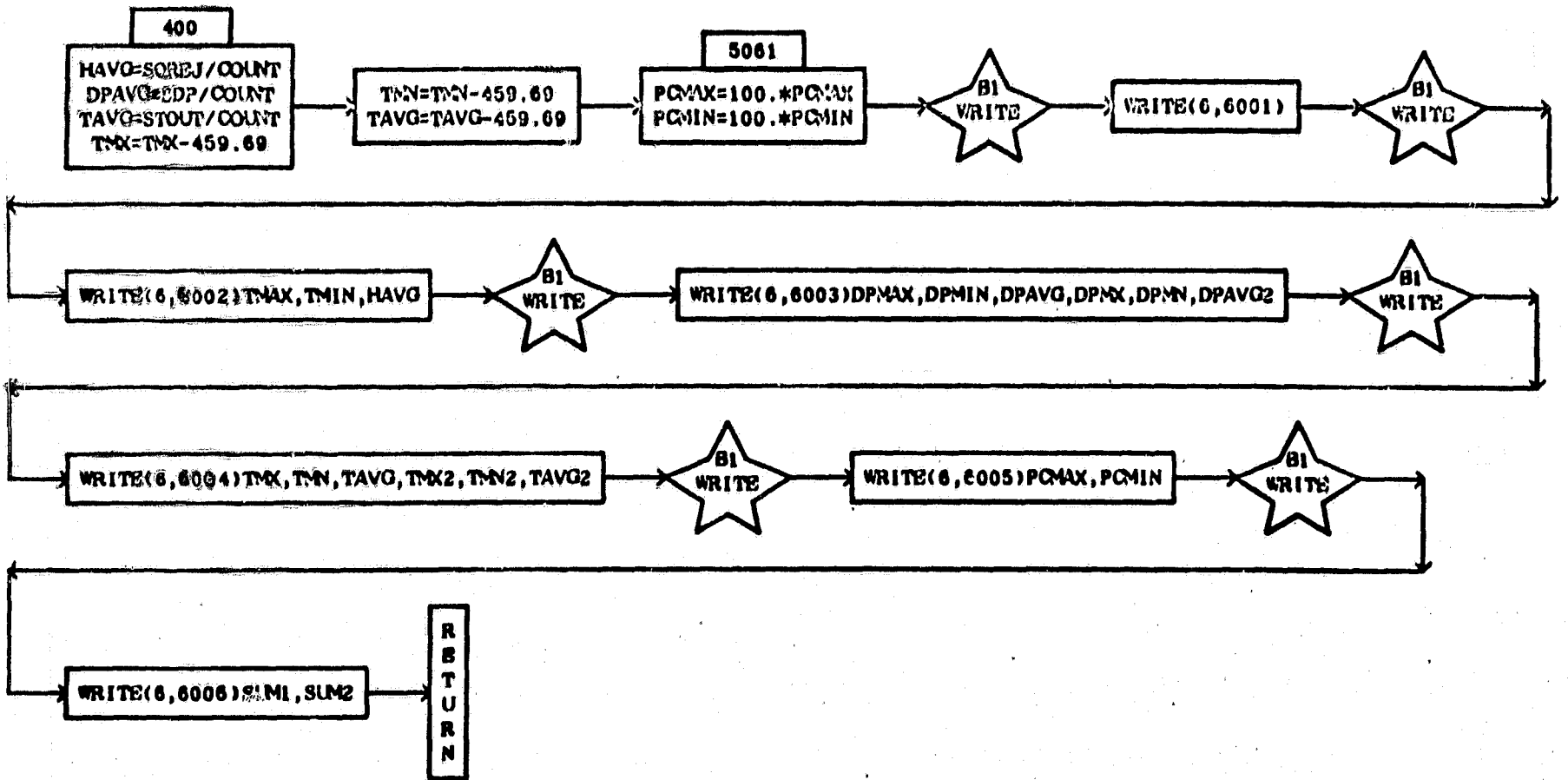








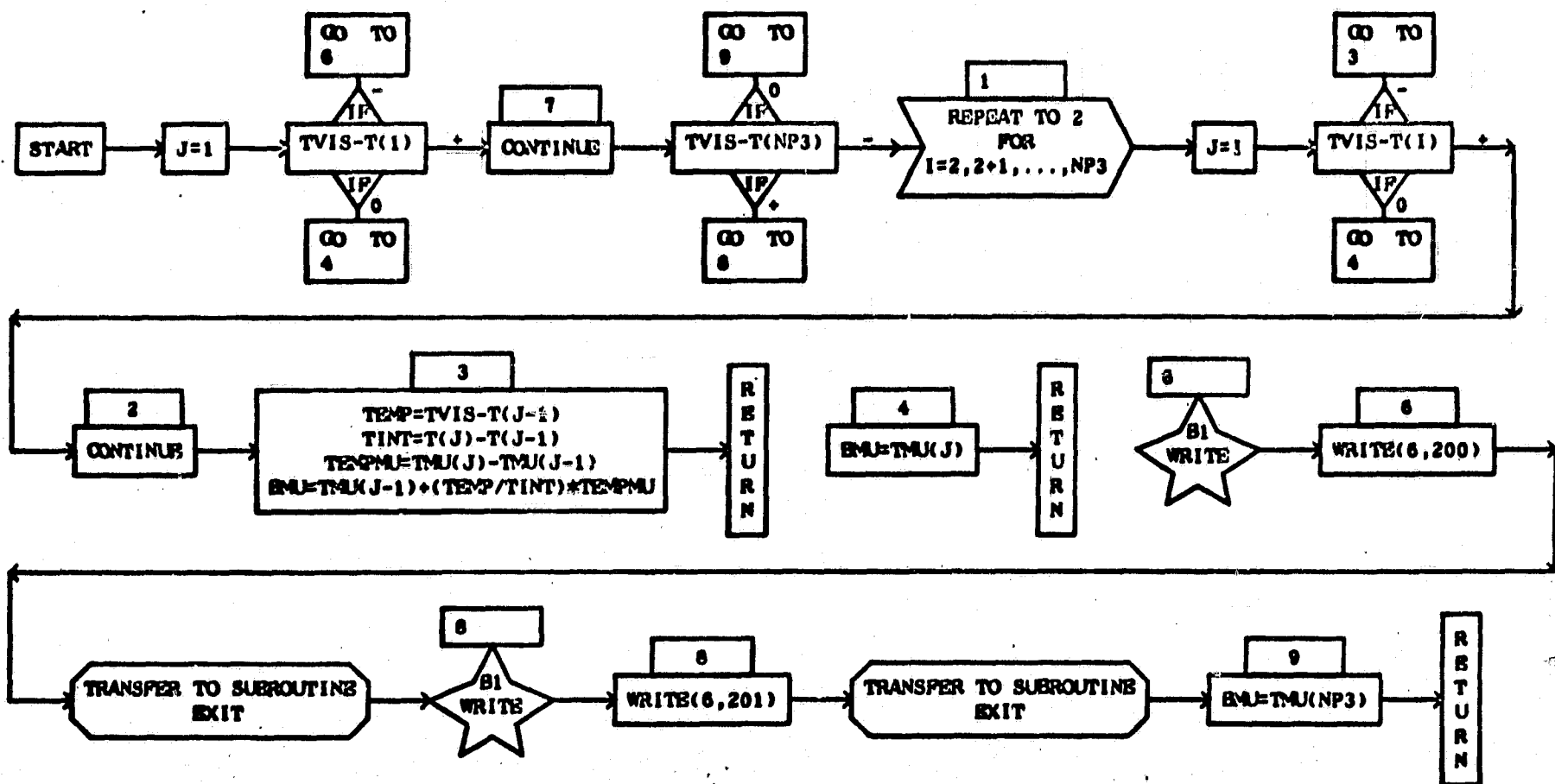




D I M E N S I O N E D V A R I A B L E S

SYMBOL	STORAGES	SYMBOL	STORAGES	SYMBOL	STORAGES	SYMBOL	STORAGES	SYMBOL	STORAGES
T	33	TNU	33						

SUBROUTINE VPOL(TVIS,BMU,NP3)



APPENDIX C
DICTIONARY OF FORTRAN TERMS

AA	Squared coefficient in quadratic equation used to solve proportioning valve flow distribution
AA1	Upstream fluid temperature coefficient, fluid nodes 1, 3, 5, 7 and 9
AA2	Upstream fluid temperature coefficient; fluid nodes 11, 13, 15, 17 and 19
AA3	Fluid to tube convection temperature coefficient, fluid nodes 1, 3, 5, 7 and 9
AA5	Tube to fluid convection temperature coefficient; tube nodes 3, 5, 7, 21 and 22
AA6	Tube to fluid convection temperature coefficient; tube nodes 13, 15 and 17
AK	Pressure drop divided by flow; $\Delta P/\dot{w}$
AKIL	Pressure drop from proportioning valve to panel 3-4 inlet manifold divided by flow rate
AKIS	Pressure drop from proportioning valve to panel 1-2 inlet manifold divided by flow rate
AKS1	Pressure drop from selective stagnation panel 1-2 outlet manifold to series panel divided by flow rate
AKS2	Pressure drop from selective stagnation panel 3-4 outlet manifold to series panel divided by flow rate
AKT21	Pressure drop in fluid node 21 divided by flow rate
AKT22	Pressure drop in fluid node 22 divided by flow rate
AKT23	Pressure drop in fluid node 23 divided by flow rate
AKT24	Pressure drop in fluid node 24 divided by flow rate
ALPHA	Curve title
ARG2	Difference between radiator mixed outlet temperature and bypass valve set point
Al0	Radiation temperature coefficient; tube nodes 3 through 8, 13 through 18 and 21 through 32
All	Structure to tube conductance temperature coefficient; structure nodes 3 through 8 and 11 through 16

A12 Absorbed heat coefficient; structure nodes 3 through 8 and 11 through 16

A13 Radiation temperature coefficient; structure nodes 3 through 8 and 11 through 16

A14 Tube to fluid convection temperature coefficient; tube nodes 1, 9, 11 and 19

A15 Tube to structure conductance temperature coefficient; tube nodes 1, 2, 9, 10, 11, 12, 19 and 20

A16 Tube to fluid convection temperature coefficient; tube nodes 2, 10, 12 and 20

A17 Structure to tube conductance temperature coefficient; structure nodes 1, 2, 9 and 10

A18 Absorbed heat coefficient; structure nodes 1, 2, 9 and 10

A19 Radiation temperature coefficient; structure nodes 1, 2, 9 and 10

A2 Tube to tube conduction temperature coefficient; tube nodes 1, 2, 11, 12 and 21 through 32

A20 Tube to structure conduction temperature coefficient; tube nodes 21 through 24, 27, 28, 31 and 32

A21 Absorbed heat coefficient; structure nodes 17 through 20

A22 Radiation temperature coefficient; structure nodes 17 through 20

A23 Structure to tube conduction temperature coefficient; structure nodes 17, 18, 19 and 20

A3A Upstream fluid temperature coefficient, fluid nodes 2, 4, 6, 8, 10, 12, 14, 16, 18 and 20

A5 Tube to structure conductance temperature coefficient, tube nodes 3, 4, 5, 6, 7, 8, 13, 14, 15, 16, 17 and 18

A5A Tube to fluid convection temperature coefficient; tube nodes 4, 6, 8, 14, 16, 18, 21, 22, 23 and 24

A7 Absorbed heat coefficient; tube nodes 1, 2, 9, 10, 11, 12, 19 and 20

A8 Absorbed heat coefficient; tube nodes 3 through 8, 13 through 18 and 21 through 32

A9	Radiation temperature coefficient; tube nodes 1, 2, 9, 10, 11, 12, 19 and 20
BB	Linear coefficient in quadratic equation used to solve proportioning valve flow distribution
BMU	Radiator fluid viscosity
B1	Upstream fluid temperature coefficient; fluid nodes 25 through 32
B3	Fluid to tube convection temperature coefficient; fluid nodes 25 through 32
B4	Tube to fluid convection temperature coefficient; tube nodes 25, 26, 27, 29, 30, 31 and 32
CC	Constant in quadratic equation used to solve proportioning valve flow distribution
COUNT	Number of calculation intervals
DBAND	Bypass valve deadband
DELTA	Difference between maximum and minimum heat rejection
DELTAP	Bypass valve movement rate
DELTAT	Temperature difference used in calculating bypass valve movement rate
DPKAP	Pressure drop divided by flow rate for secondary system nodes
DPKAPS	Sum of pressure drop divided by flow rate for secondary system nodes
DPPRNT	Total primary system pressure drop
DPT	Pressure drop in individual tubes on the selective stagnation panels
DPTOT	Secondary system total pressure drop
DPTS1	Pressure drop for panel 1-2 loop from proportioning valve to bypass line mix junction
DPTS2	Pressure drop for panel 3-4 loop from proportioning valve to bypass line mix junction
DTEMP	Difference in panel outlet temperatures used to determine proportioning valve position

DX	Change in proportioning valve position
E	Pressure drop coefficient for inlet end of selective stagnation panels
F	Pressure drop coefficient for outlet end of selective stagnation panels
FLOW	Flow rate used in calculating total primary system pressure drop
FLOWMN	Minimum fraction of total flow through the radiator
FLOWMX	Maximum fraction of total flow through the radiator
FLOWPC	Fraction total flow through the radiator as determined by the bypass valve
FULOPN	Proportioning valve maximum possible position from the left
GFACT	Proportioning valve geometry factor
H	Fluid enthalpy
HIN	Fluid enthalpy for redundant system inlet
HINLT	Fluid enthalpy for primary system inlet
HMIX	Fluid enthalpy downstream of primary system bypass valve
HOUT	Fluid enthalpy for redundant system outlet
HPRI	Fluid enthalpy upstream of primary system bypass valve
H21	Fluid enthalpy out of selective stagnation panel 1-2
H22	Fluid enthalpy out of series panel 1-2
H23	Fluid enthalpy out of selective stagnation panel 3-4
H24	Fluid enthalpy out of series panel 3-4
ITER	Iteration counter
LAST	Used for curve read in; indicates location of last variable
LTER	Iteration counter
M	On/off code for first stage redundant system inline heater
MCODE	Code which indicates whether or not further iterations are required

MERR	Error code which indicates which curve limits were exceeded
MM	On/off code for second stage redundant system inline heater
MMM	On/off code for primary system inline heater
MSTART	Address of first time point on each curve to be considered during an iteration
MT	Address of first dependent variable on primary system inlet temperature curve to be considered during an iteration
MTSEC	Address of first dependent variable on secondary system inlet temperature curve to be considered during an iteration
MW	Address of first dependent variable on primary system flow rate curve to be considered during an iteration
MWSEC	Address of first dependent variable on secondary system flow rate curve
NEXT	Used for curve read in; indicates location of first variable
NFCODE	Fluid lump code for suspending temperature iteration
NOPTS	Number of points on flow rate and inlet temperature curves
NPTS	Number of points on time dependent curves
NP1	Address of last time point for absorbed heat curve for panel 1
NP2	Address of last time point for absorbed heat curve for panel 2
NP3	Address of last time point for absorbed heat curve for panel 3
NP4	Address of last time point for absorbed heat curve for panel 4
NP5	Number of points on viscosity versus temperature curve
NP5A	Address of last time point for primary system flow rate curve
NP5B	Number of points on primary system flow rate curve
NP6A	Address of last time point for primary system inlet temperature curve
NP6B	Number of points on primary system inlet temperature curve
NP7	Address of last time point for secondary system flow rate curve

NP8	Address of last time point for secondary system inlet temperature curve
NSCODE	Structure lump code for suspending temperature iteration
NTCODE	Tube lump code for suspending temperature iteration
PCBYP	Flow rate through bypass line
PERIOD	Total mission time
POSIN	Initial proportioning valve position from left
POSMAX	Proportioning valve maximum allowable position from left
POSMIN	Proportioning valve minimum allowable position from left
PPARA	Panel parameter used in proportioning valve calculations
Q	Dependent values for absorbed heat curves
QABS	Absorbed heat for each panel
QRJT	Primary system heat rejection rate
QRJTSC	Redundant system heat rejection rate
RATE1	Flow rate in panel 1-2
RATE2	Flow rate in panel 3-4
RLIMIT	Bypass valve rate limit, fraction bypass per time interval
RTFCTR	Bypass valve rate factor, fraction bypass per time interval per °F
SAVE	Mission time
SAVE1	Print interval indicator
SDP	Sum of system pressure drop
SETPT	Bypass valve control point temperature
SLTEMP	Temperature downstream of bypass valve
SQREJ	Sum of total heat rejection
STOUT	Sum of radiator outlet temperatures
SUM1	Total primary system inline heater heat output

SUM2	Total redundant system inline heater heat output
T	Independent variable for viscosity curve
TAU	Mission time
TEMP	Difference in mission time and time on time-dependent curves used for interpolation
TEMPQ	Difference in adjacent absorbed heat values on absorbed heat curve; used for interpolation
TEMPT	Difference in adjacent inlet temperatures values on primary inlet temperature curve; used for interpolation
TEMPTS	Difference in adjacent inlet temperature values on secondary inlet temperature curve; used for interpolation
TEMPW	Difference in adjacent flow rate values on primary flow rate curve, used for interpolation
TEMPWS	Difference in adjacent flow rate values on secondary flow rate curve; used for interpolation
TF	Fluid lump temperatures
TFA21	Inlet temperature for panel 1-2 series panel
TFA23	Inlet temperature for panel 3-4 series panel
TFIN	Fluid lump temperatures calculated last iteration
TIME	Independent variable for time-dependent curves
TIN	Dependent values for primary system inlet temperature curve
TINLSC	Dependent values for secondary system inlet temperature curve
TINT	Difference in adjacent time values on time dependent curve; used for interpolation
TMIX	Temperature downstream of bypass valve
TMU	Dependent variable for viscosity curve
TOTREJ	Total heat rejection rate for both primary and secondary systems
TOUT	Primary system outlet temperature
TOUTSC	Redundant system outlet temperature

TPF	Used for checking fluid lump temperatures
TPPF	Fluid lump temperatures calculated this iteration
TPPS	Structure lump temperatures calculated this iteration
TPPT	Tube lump temperatures calculated this iteration
TPS	Used for checking structure lump temperatures
TPT	Used for checking tube lump temperatures
TS	Structure lump temperatures
TSIN	Structure lump temperatures calculated last iteration
TT	Tube lump temperatures
TTIME	Time for mission 4
TTIN	Tube lump temperatures calculated last iteration
VLVGAN	Proportioning valve gain
VTOL	Proportioning valve null position tolerance
WDOT	Flow rate in a tube
WDOTSC	Dependent values for secondary system flow rate curve
WDOTT	Primary system flow through radiator panels
WDOTT1	Primary system flow rate to panel 1-2
WDOTT2	Primary system flow rate to panel 3-4
WDTTOT	Dependent values for primary system flow rate curve
XX1	Proportioning valve position from left
XX2	Proportioning valve position from right

THIS FORM MUST BE COMPLETED BY TYPEWRITER

01 4		01 7 PROGRAM NO		COMPUTER PROGRAM ABSTRACT				01 14 DATE 30 Sept. 1968	
01 29 TITLE OF PROGRAM (61 CHARACTERS MAXIMUM) LTV-APOLLO II ECS Radiator Analysis						PARENT PROGRAM			
02 26 CATEGORY		02 27 LANGUAGE NO. 1		02 32 LANGUAGE NO. 2		02 37 KEY WORDS (8 MAXIMUM, SEPARATED BY COMMAS) Radiator, Temperature, Fluid Flow			
<input type="checkbox"/> F		FOR5							
WHO TO CONTACT ABOUT THE PROGRAM						05 48 STATUS		05 49	
05 14 CONTACT D. W. Morris		05 28 SITE MSC		05 31 ORGN CODE EC 34		05 39 PROJECT NO 3475		<input type="checkbox"/> A. UNDER DEVELOPMENT <input type="checkbox"/> B. OPERATIONAL <input checked="" type="checkbox"/> C. COMPLETED	
05 50 INITIATED 0167		05 54 COMPLETED 0568		05 58 REVISION CODE <input type="checkbox"/> A REVISION <input type="checkbox"/> B CANCELLATION		05 59 MAN/MONTHS 2.5		05 64 MACHINE HOURS 5.0	
						05 67 COMPUTER TYPE 1108		05 74 TOTAL COST (DOLLARS) 6000	
						59 60 61 62 63		64 65 66 67 68	
								74 75 76 77 78 79 80	
CARD NUMBER		COLUMN 14		ELITE MARGIN					
				PICA MARGIN					
				ABSTRACT					
06				This computer routine provides for rapid performance					
07				predictions of the Apollo Block II ECS radiator. Specific					
08				equations for a simplified thermal model of the radiators are					
09				written directly in the program. The temperature equations for					
10				each node in the thermal model are solved by an implicit finite					
11				difference method. A characterization of the flow proportioning					
12				valve, bypass valve, and low load heater are included in the					
13				routine. Provisions are also included for single panel and					
14				redundant system operation.					
15									
16				Two lunar orbital and a translunar thermal cycle					
17				radiator environments are contained in the routine. Time de-					
18				pendent values may also be input for any radiator environment,					
19				inlet temperature and flow rate. The routine outputs radiator					
20				heat rejection, pressure drop, low load heater on/off operation,					
21				flow rate, and outlet fluid temperature at times specified by					
22				the user. Following completion of the problem, maximum, minimum					
23				and average values for heat rejection, pressure drop and fluid					
24				outlet temperature and total heat dissipated by the low load					
25				heater are also output.					
26									
27				Computer time required to analyze a 4.08 hour lunar					
28				orbit mission (two orbits) with a calculation and print interval					
29				of .02 hours is 53 seconds on the Univac 1108 computer. This					
30				represents a routine run speed of better than 250 times real					
31				time.					
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